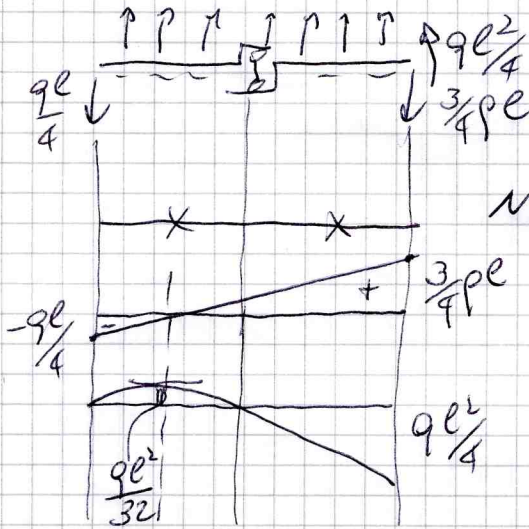


$$q_y(z) = -q$$

$$s = 5 \quad n_s = 2 \rightarrow S = 3 + 2$$

const. momento  
 tra equilibris  
 isonotico  
 rotazionale

Equilibrio:

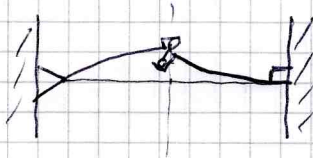


$$N(z) = 0$$

$$M(z) = -\frac{ql}{4}z + \frac{qz^2}{2}$$

$$0 \leq z \leq l$$

Definite qualitative:



$$\uparrow v(l/2) < 0$$

$$\Delta v(l/2) = 0 \quad \Delta q \neq 0$$

Limite elastico:

$$\frac{dv}{dx} = -\frac{M(z)}{K_X}$$

Integrale generale:

$$v(z) = -\frac{qz^4}{24K_X} + \frac{qlz^3}{24K_X} + C_1z + C_2$$

Condizioni di contorno e di continuita:

$$\begin{cases} v(0) = 0 \\ v(l) = 0 \\ \varphi(l) = -\frac{dv(l)}{dx} = 0 \\ v(l/2^+) = v(l/2^-) \end{cases}$$

Note:

La scelta di un'unica funzione  $v(z)$  (determinando  $C_1$  e  $C_2$  con due delle 4 cond. di contorno) non comporta il soddisfacimento delle altre condizioni di contorno

⇒ occorre determinare 2 funzioni

$$v_1(z) = A + C_1z + C_2 \quad 0 \leq z \leq l/2$$

$$v_2(z) = A + C_3z + C_4 \quad l/2 \leq z \leq l$$

Soluzione:

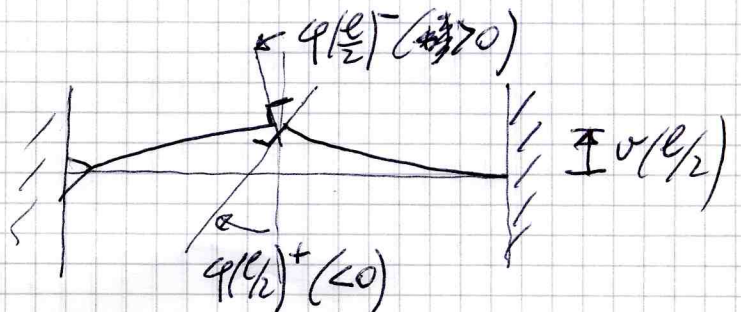
$$v_1(z) = A - \frac{ql^3}{24K_X}$$

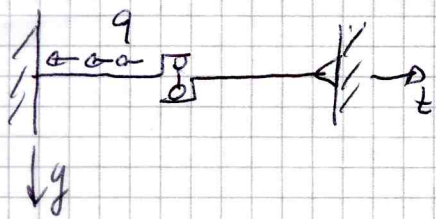
$$v_2(z) = A + \frac{ql^3}{24K_X}z - \frac{ql^4}{24K_X}$$

$$v_1(l/2) = v_2(l/2) = -\frac{7}{384} \frac{ql^4}{K_X}$$

$$\Delta \varphi(l/2) = \varphi(l/2)^+ - \varphi(l/2)^- = -\frac{ql^3}{12K_X}$$

$$N(l/2) = 0$$



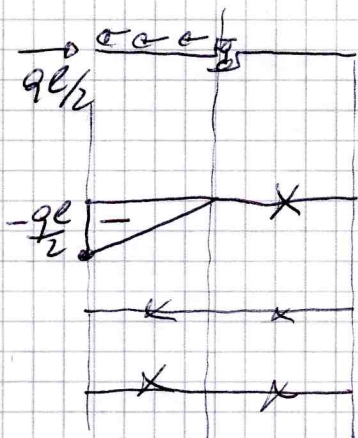


$$q_z(z) = -q$$

$$S = S \quad M_S = 0$$

→ cond. normale per equilibrio isotropico e rotazionale

Equilibrio:



$$N(z) = \begin{cases} -q\frac{l}{2} + qz & 0 \leq z \leq \frac{l}{2} \\ 0 & z > \frac{l}{2} \end{cases}$$

$$T(z) = 0$$

$$M(z) = 0$$

Linea elastica: (per  $0 \leq z \leq \frac{l}{2}$ )

$$\frac{d^2 w}{dz^2} = \frac{N(z)}{KE}$$

Integrale generale:  $w(z) = -\frac{q\frac{l}{2}}{2KE}z + \frac{qz^2}{2KE} + C_1$

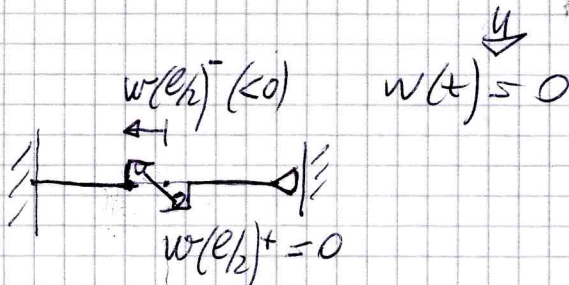
Condizione al contorno

$$w(0) = 0 \rightarrow C_1 = 0$$

Linea elastica (per  $\frac{l}{2} \leq z \leq l$ ):

trave libera, bella in  $l/2 \rightarrow w(z) = 0 \quad \forall z \in \frac{l}{2} \leq z \leq l$

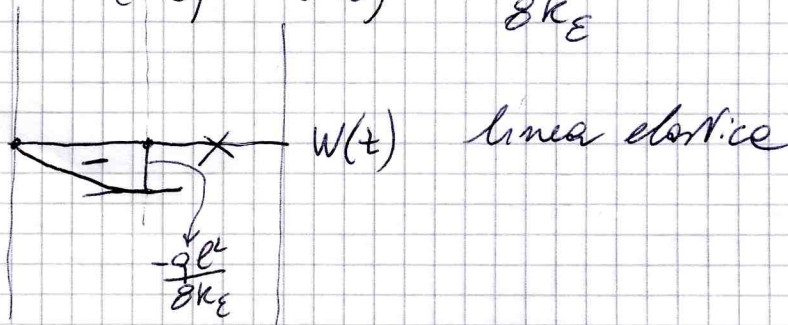
si verifica con  $\frac{dw}{dz} = 0$  • integrale generale  $w(z) = C_1$   
• cond. contorno  $w(l) = 0 \rightarrow C_1 = 0$



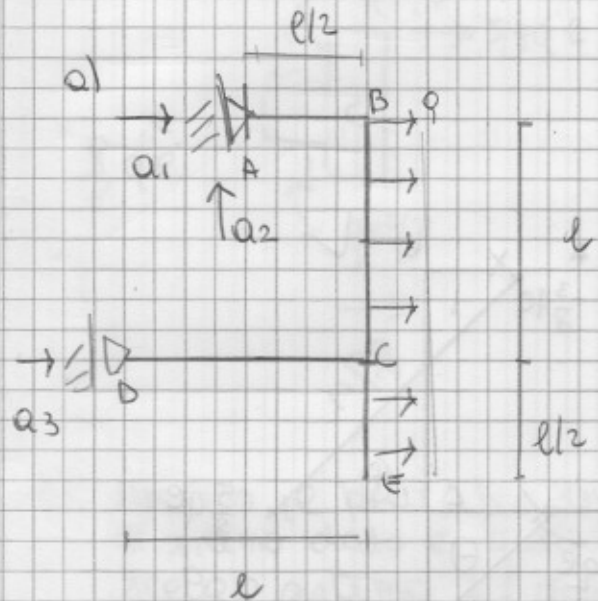
Deformata:

$$w(l/2)^- = -\frac{q\frac{l^2}{8}}{8KE}$$

$$\Delta w = w(l/2)^+ - w(l/2)^- = +\frac{q\frac{l^2}{8}}{8KE}$$



Problema 2)



$S = 2 + 1$   
 $(m_s = 0)$

$S > 3$

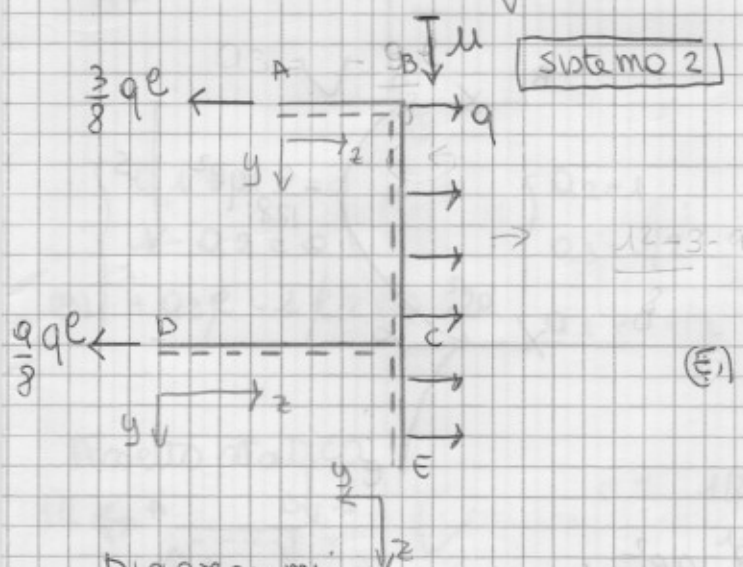
$3 = 3$

cond. necessarie  
 per l'equilibrio statico  
 soddisfatta ✓

Determino le reaz. vincolari:

$$\begin{cases} a_1 + a_3 + q \frac{3}{2} l = 0 \\ a_2 = 0 \\ a_3 l + \frac{3}{2} q l \left( l + \frac{l}{2} \right) \cdot \frac{1}{2} = 0 \end{cases} \quad \begin{cases} a_2 = 0 \\ a_3 = -\frac{q}{8} q l \\ a_1 = \frac{q}{8} q l - \frac{3}{2} q l = -\frac{3}{8} q l \end{cases} \quad \begin{matrix} [F] [K] \checkmark \\ [K] \\ [F] [K] \checkmark \\ [K] \end{matrix}$$

Assetto statico ✓



Verifiche:

$\frac{3}{2} ql - \frac{3}{8} ql - \frac{q}{8} ql = 0 \checkmark$

$0 = 0 \checkmark$

(E)  $+\frac{3}{8} ql \frac{3}{2} l - \frac{3}{2} ql \cdot \frac{3}{4} l + \frac{q}{8} ql \frac{l}{2} = 0 \checkmark$

$\frac{9}{16} - \frac{9}{8} + \frac{9}{16} = 0$

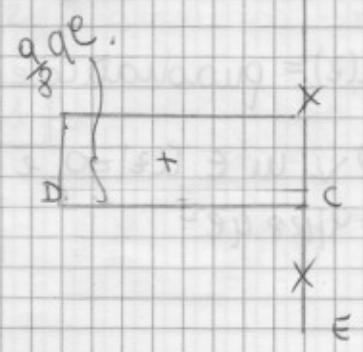
Diagrammi

Forze normali

AB  $q_z(z) = 0 \quad N(z) = \text{cost} = \frac{3}{8} ql$

BE  $q_z(z) = 0 \quad N(z) = 0$

CD  $q_z(z) = 0 \quad N(z) = \text{cost} = +\frac{q}{8} ql$



## Taglio

AB  $q_y(z) = 0$   $T(z) = \text{cost} = 0$

CD  $q_y(z) = 0$   $T(z) = \text{cost} = 0$

BC  $q_y(z) = \text{cost} = -q \Rightarrow T(z) = \text{lineare e}$

crescente perché  $\frac{dT}{dz} = -q_y(z) > 0$ .

In B\*  $T(z) = -\frac{3}{8} qe$ .

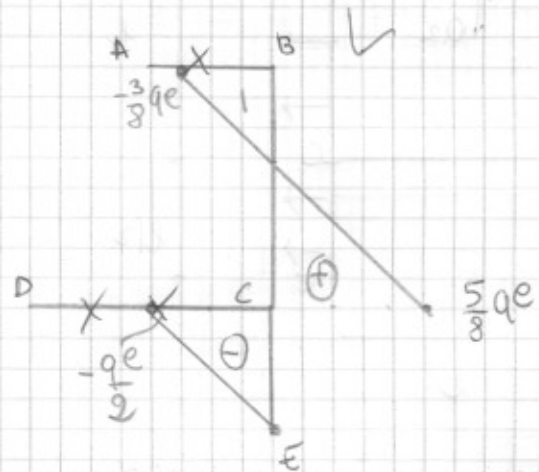
In C  $T(z) = -\frac{3}{8} qe + qe = \frac{5}{8} qe$ .

In C, ho una discontinuità di taglio

CE  $q_y(z) = \text{cost} \Rightarrow T(z) = \text{lineare e}$   
crescente

In D\*  $T(z) = \frac{3}{8} qe + qe - \frac{9}{8} qe = -\frac{1}{2} qe$

In E  $T(z) = 0$ .



## Momento

AB  $T(z) = 0$   $M(z) = \text{cost} = 0$

CD  $T(z) = 0$   $M(z) = \text{cost} = 0$

BC  $T(z) = \text{lin}$   $M(z) = \text{quadratico e}$   
convesso.

In B  $M(z) = 0$ .

In C  $M(z) = +\frac{qe}{2} \cdot \frac{e}{4} = +\frac{qe^2}{8}$

$T(z) = 0$  in  $z = \frac{3}{8} e$ .

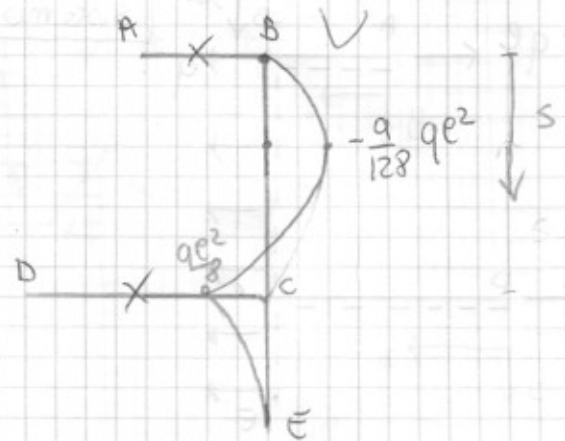
$T(z)$  è punto di minimo ( $\frac{d^2M}{dz^2} > 0$ )

$M(z = \frac{3}{8} e) = -\frac{3}{8} qe \cdot \frac{3e}{8} + q \frac{9}{64} \frac{e^2}{2} = -\frac{9}{128} qe^2$

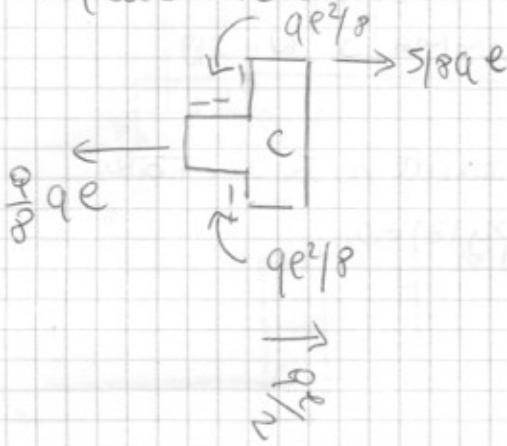
CE  $T(z) = \text{lin}$ .  $M(z) = \text{quadratico e convesso}$ .

In E  $M(z) = 0$ . In C  $M(z) = 0$  è punto a tg orizz, il punto di min.

In C  $M(z) = -\frac{9}{128} qe^2$



Equilibrio modo e



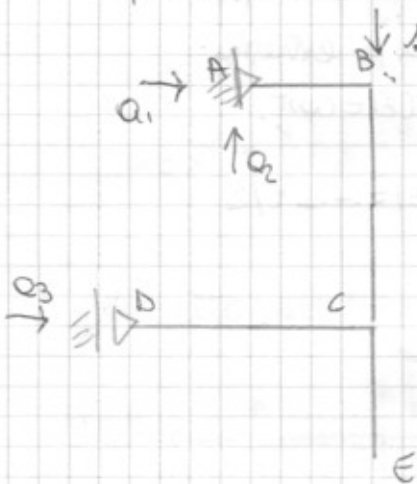
$$\frac{qe^2}{8} - \frac{qe^2}{8} = 0 \quad \checkmark$$

$$0 = 0 \quad \checkmark$$

$$\frac{qe}{2} + \frac{5}{8} qe - \frac{q}{8} qe = 0 \quad \checkmark \quad \frac{4+5-q}{8}$$

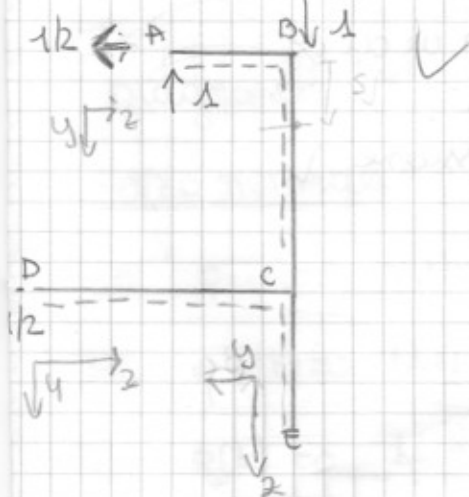
b) Applico il principio dei lavori virtuali. Il sistema dato e quello di spostamenti e deformazioni congruenti. Scelgo sistema 1 (di forze e sollecitazioni equilibrate), formato dalla stessa travatura e truccioli del sistema congruente, in cui applico in B una forza unitaria nella direzione dello spostamento cercato. (Suppongo positivo lo spostamento verso il basso).

$S = 3$  recond. nec. per equilibrio  
 $\rightarrow$  isostatico soddisfatto.



$$\begin{cases} a_1 + a_3 = 0 \\ 1 - a_2 = 0 \\ +a_3 e - 1/2 e = 0 \end{cases} \quad \begin{cases} a_2 = 1 \\ a_3 = 1/2 \\ a_1 = -1/2 \end{cases}$$

Ametto statico



$$-1/2 - 1/2 = 0 \quad \checkmark$$

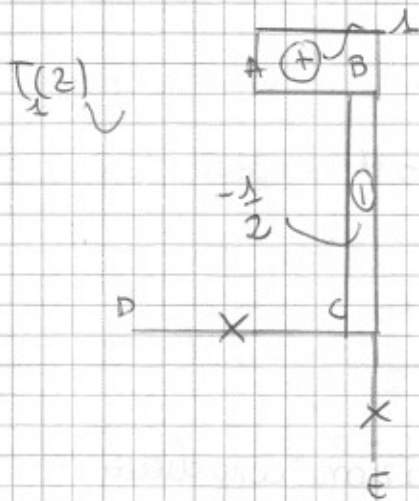
$$1 - 1 = 0 \quad \checkmark$$

$$(B) \quad -1 \cdot \frac{e}{2} + \frac{1}{2} \cdot e = 0 \quad \checkmark$$

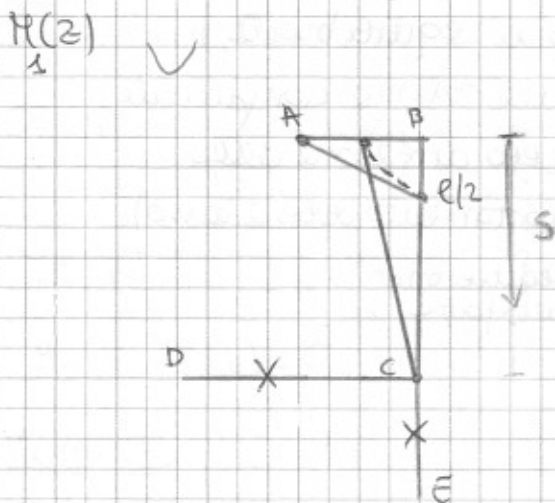
**Sistema 1**

Si come è indeformabile a Poisson normale, calcolerò solo  
 le Taglie e il momento.

FARE DIAGRAMMA DI N



$T(z)$  è costante su tutti i tratti  
 perché  $p_y(z) = 0$ .



$M(z) = \text{cost}$  sui tratti in cui  $T(z) = 0$   
 e nullo perché non ci sono  
 momenti applicati agli estremi.  
 $M(z) = \text{lineare}$  dove  $T(z) = \text{cost}$ .

$\text{in A } M(z) = 0$  ;  $\text{in B } M(z) = +1 \cdot e/2$   
 $\text{in C } M(z) = 0$

Applico Pl V.

$L_F = 1 \cdot u$

$L_i = \int_B (N_1 \epsilon_z + T_1 \gamma + M_1 \chi) ds = \int_B (N_1 \frac{N}{KE} + T_1 \frac{T}{KT} + M_1 \frac{M}{KX}) ds$

$L_i = \int_B M_1 \frac{M}{KX} ds$

Considero tratto BC perché è l'unico tratto in cui il  
 momento è diverso da 0 in entrambi i sistemi.

Sceglio lo stesso coordinato s per definire l'espressione.

$M_1(s) = \frac{1}{2} + \frac{1}{2} \frac{e-s}{e} = \frac{1}{2} - \frac{1}{2} \frac{s}{e}$

$M(s) = -\frac{q}{8} q e s + \frac{q s^2}{2} = q \frac{s^2}{2} - \frac{3}{8} q e s$

$$x_1 = \int_0^l \left( \frac{l-s}{2} \right) \left( \frac{qs^2}{2} - \frac{3}{8} \frac{qes}{s} \right) \frac{1}{kx} ds = l$$

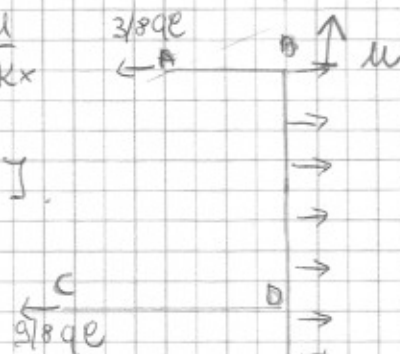
$$l = \int_0^l \left( \frac{l-s}{2} \right) \left( \frac{qs^2}{2} - \frac{3}{8} \frac{qes}{s} \right) \frac{1}{kx} ds =$$

$$= \frac{1}{2kx} \int_0^l \left( \frac{qs^3}{2} - \frac{3}{8} qe s^2 - \frac{q}{2} s + \frac{3}{8} qes^2 \right) ds =$$

$$= \frac{1}{2kx} \left[ \frac{qe}{2} \frac{l^4}{4} - \frac{3}{8} qe \frac{l^3}{3} - \frac{q}{2} \frac{l^2}{2} + \frac{3}{8} qe \frac{l^3}{3} \right] =$$

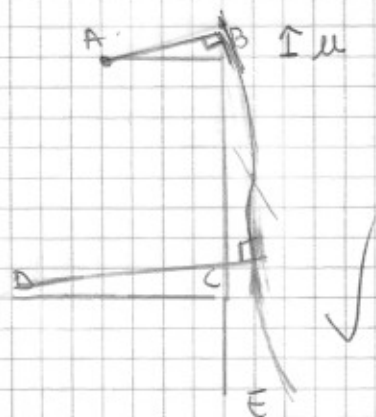
$$= \left( \frac{qe^4}{12} - \frac{3qe^4}{32} - \frac{qe^4}{16} + \frac{3qe^4}{48} \right) \frac{1}{kx}$$

$$= -\frac{1}{96} \frac{qe^4}{kx} \quad \checkmark \quad \frac{[F][L]^4}{[L][E][L]^2} = [L]$$



B si sposta verso l'alto, continuamente e quanto ipotizzato.

c) deformato qualitativo



guardare i diagrammi del momento.

Dove ho disegnato il grafico del momento e sono le fibre tese.

Rispetto i vincoli e i nodi rigidi in B e C.

Dove M è nullo ho una rotazione rigida.