

Optimal control of complex flows: versus real applications

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The work presented has been carried out in collaboration with:

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- **Marco Carini** Post-Doc, IMFT, Toulouse, France;
- **Paolo Luchini** DIIN, University of Salerno, Italy;
- **Tom Bewley** UCSD, USA;

1 Introduction

2 Methods to solve large scale optimization problems without introducing approximations

- MCE: Minimal Control Energy
- ADA: Adjoint of the Direct-Adjoint

3 Applications

- Control of cylinder wake at low values of Re
- Control of boundary-layer flow

4 High Re number application

5 Appendix

Introduction

Definitions

Complex flows

here problems with **large** number of degrees of freedom

Optimal control

The linearized system

$$\frac{dx}{dt} = Ax + Bu \quad \text{on } 0 < t < T, \quad \text{with } x = x_0 \quad \text{at } t = 0.$$

- x has dimension n and u dimension m
- here $n \gg m$
- find u that minimizes a quadratic cost function J
- consider: full state information, no estimation **for now** (but we have done that as well)

The linear optimal control problem

The classical full-state-information control problem is formulated as: find the control \mathbf{u} that minimizes the cost function

$$J = \frac{1}{2} \int_0^T [\mathbf{x}^H \mathbf{Q} \mathbf{x} + l^2 \mathbf{u}^H \mathbf{R} \mathbf{u}] dt,$$

where l is the penalty of the control, and the state \mathbf{x} and the control \mathbf{u} are related via the state equation

$$\frac{d\mathbf{x}}{dt} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \quad \text{on } 0 < t < T, \quad \text{with } \mathbf{x} = \mathbf{x}_0 \quad \text{at } t = 0.$$

The solution depends on: \mathbf{x}_0 , T , \mathbf{Q} , \mathbf{R} and l .

Solution approaches

- With a feedback rule $\mathbf{u} = K\mathbf{x}$, and a system which is LTI, then the feedback matrix K is computed **once off-line** (**convenient** since K is independent of \mathbf{x}_0).
- Optimal control \mathbf{u} corresponding to the state at each time step is computed in **real time**, normally with a **finite horizon** (value of T) to make it tractable. (Example: Adjoint-based control optimization.)

Both approaches can be solved using the **adjoint** of the state equation (see the appendix for discussions, derivations and comparison with other methods to compute gradients).

Derivation of adjoint I

The **adjoint variable** \mathbf{p} is introduced as a **Lagrange multiplier**. The **augmented cost function** is written

$$J = \int_0^T \left(\frac{1}{2} [\mathbf{x}^H \mathbf{Q} \mathbf{x} + l^2 \mathbf{u}^H \mathbf{R} \mathbf{u}] - \mathbf{p}^H \left[\frac{d\mathbf{x}}{dt} - \mathbf{A} \mathbf{x} - \mathbf{B} \mathbf{u} \right] \right) dt,$$

linearize + integration by parts and $\delta J = 0$ gives

$$0 = \int_0^T \left(\delta \mathbf{u}^H [\mathbf{B} \mathbf{p} + l^2 \mathbf{R} \mathbf{u}] + \delta \mathbf{x}^H \left[\frac{d\mathbf{p}}{dt} + \mathbf{A}^H \mathbf{p} + \mathbf{Q} \mathbf{x} \right] \right) dt + [\delta \mathbf{x}^H \mathbf{p}]_0^T,$$

Derivation of adjoint II

The adjoint variable \mathbf{p} is introduced as a Lagrange multiplier. The augmented cost function is written

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gives adjoint equations (obs! $\delta \mathbf{x}(0) = 0$), backward in time integration

$$-\frac{d\mathbf{p}}{dt} = \mathbf{A}^H \mathbf{p} + \mathbf{Q} \mathbf{x}, \quad \text{with } \mathbf{p}(t = T) = 0,$$

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$$-\frac{d\mathbf{p}}{dt} = \mathbf{A}^H \mathbf{p} + \mathbf{Q} \mathbf{x}, \quad \text{with } \mathbf{p}(t = T) = 0,$$

and optimality condition

$$\mathbf{u} = -\frac{1}{l^2} \mathbf{R}^{-1} \mathbf{B}^H \mathbf{p}.$$

Optimal control using feedback

If we consider a feedback rule $\mathbf{u} = K\mathbf{x}$ then

$$\mathbf{u} = K\mathbf{x} = -\frac{1}{l^2}R^{-1}B^H\mathbf{p}.$$

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How does it work ?

Note that **state** is often denoted **direct**

Two-point boundary value problem

Write the **direct and adjoint equations** on a **combined matrix form**

$$\frac{dz}{dt} = Zz \quad \text{where} \quad Z = Z_{2n \times 2n} = \begin{bmatrix} A & -I^{-2}BR^{-1}B^H \\ -Q & -A^H \end{bmatrix} \quad (1)$$

$$z = \begin{bmatrix} \mathbf{x} \\ \mathbf{p} \end{bmatrix}, \quad \text{and} \quad \begin{cases} \mathbf{x} = \mathbf{x}_0 & \text{at } t = 0, \\ \mathbf{p} = 0 & \text{at } t = T. \end{cases}$$

(Z has a **Hamiltonian symmetry**; **eigenvalues** appear in pairs of **equal** λ_i and **opposite** λ_r .)

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This linear ODE is a **two-point boundary value problem** and may be solved using $\mathbf{p} = X\mathbf{x}$.

If the system is time invariant (**LTI**) and we take the limit that $T \rightarrow \infty$. The steady state solution for X satisfies the **algebraic Riccati equation**

$$0 = A^H X + XA - XI^{-2}BR^{-1}B^H X + Q,$$

where additionally X is constrained such that $A + BK$ is stable.

The classical way of solution

A linear time-invariant system (LTI) can be solved using its **eigenvectors**. Assume that an eigenvector **decomposition** of the $2n \times 2n$ matrix Z is available such that

$$Z = V\Lambda_c V^{-1} \quad \text{where} \quad V = \begin{bmatrix} V_{11} & V_{12} \\ V_{21} & V_{22} \end{bmatrix} \quad \text{and} \quad \mathbf{z} = \begin{bmatrix} \mathbf{x} \\ \mathbf{p} \end{bmatrix}$$

and the eigenvalues of Z appearing in the diagonal matrix Λ_c are **enumerated in order of increasing real part**. Since

$$\mathbf{z} = V e^{\Lambda_c t} V^{-1} \mathbf{z}_0$$

the **solutions** \mathbf{z} that **obey the boundary conditions** at $t \rightarrow \infty$ are spanned by the **first n columns** of V . The direct (\mathbf{x}) and adjoint (\mathbf{p}) parts of these columns are related as $\mathbf{p} = \mathbf{X}\mathbf{x}$, where

$$[\mathbf{p}_1, \mathbf{p}_2, \dots, \mathbf{p}_n] = \mathbf{X}[\mathbf{x}_1, \mathbf{x}_2, \dots, \mathbf{x}_n] \quad \rightarrow \quad \mathbf{X} = V_{21} V_{11}^{-1}$$

Motivation

- **Optimal control** via application of modern control algorithms (**Riccati equation**) is **intractable** because of the very **large number of degrees of freedom** deriving from the discretization of the Navier-Stokes equations.

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- Here, we present two exact methods which do not rely on such modeling,

where

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- ① In the limit that $l^2 \rightarrow \infty$, **MCE: Minimal Control Energy**
- ② For any value of l^2 , more general, **ADA: Adjoint of the Direct-Adjoint**

Minimal Control Energy

Minimal-energy control feedback I

In the limit that $l^2 \rightarrow \infty$ we consider

$$J = \int_0^T \frac{1}{2} [l^{-2} \mathbf{x}^H Q \mathbf{x} + \mathbf{u}^H R \mathbf{u}]$$

With this definition the same derivation as before leads to

$$\frac{dz}{dt} = Zz \quad \text{where} \quad Z = Z_{2n \times 2n} = \begin{bmatrix} A & -BR^{-1}B^H \\ -l^{-2}Q & -A^H \end{bmatrix}$$

Taking the limit $l^2 \rightarrow \infty$ we get

Minimal-energy control feedback II

In the limit that $l^2 \rightarrow \infty$ we consider

$$J = \int_0^T \frac{1}{2} [l^{-2} \mathbf{x}^H Q \mathbf{x} + \mathbf{u}^H R \mathbf{u}]$$

With this definition the same derivation as before leads to

$$\frac{dz}{dt} = Zz \quad \text{where} \quad Z = Z_{2n \times 2n} = \begin{bmatrix} A & -BR^{-1}B^H \\ \mathbf{0} & -A^H \end{bmatrix}$$

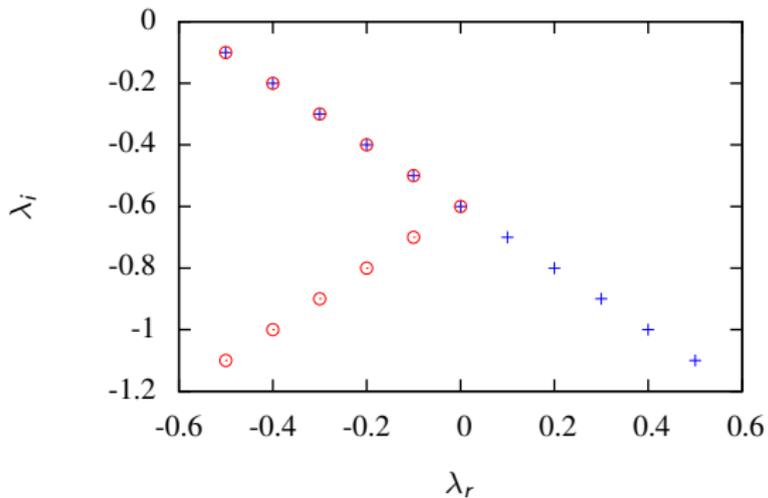
Z becomes **block triangular**. The direct and adjoint equations are

$$\frac{\partial \mathbf{x}}{\partial t} = A\mathbf{x} + B\mathbf{u}, \quad \mathbf{u} = -R^{-1}B^H\mathbf{p}, \quad \frac{\partial \mathbf{p}}{\partial t} = -A^H\mathbf{p} + \mathbf{0}$$

The **eigenvalues** of this system is given by the **union** of the eigenvalues of A and $-A^H$.

Minimal-energy control feedback III

What we would like to do (and **will** do) is to stabilize only the unstable eigenvalues.



The eigenvalues of (+) the discretized open-loop system, and (o) the closed-loop system $A + BK$ after minimal-energy control is applied.

Minimal-energy control feedback IV

Here we **know** the eigenvalues and only need to compute

$$X = V_{21} V_{11}^{-1}$$

It can be shown (see the appendix) that X is only function of V_{21} . K is finally given as a function of the **unstable eigenvalues** and corresponding **left eigenvectors**.

$$K = -B^H T_u F^{-1} T_u^H$$

where F has elements

$$f_{ij} = c_{ij} / (\lambda_i + \lambda_j^*)$$

and

$$C = T_u^H B B^H T_u$$

T_u is the matrix containing **unstable left eigenvectors**, λ_i the corresponding **eigenvalues** and * **conjugate**.

ADA: Adjoint of the Direct-Adjoint

Riccati-less optimal control I

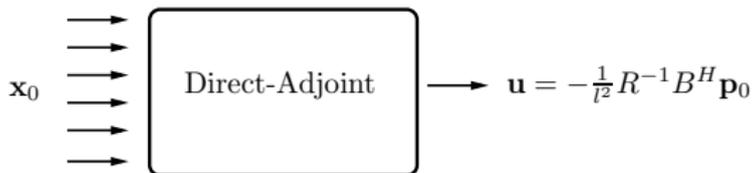
The **aim** is to compute the solution for K , which is **independent of x_0** and **time invariant**. This can be solved using an iterative procedure to “try” different x_0 (**computationally expensive**), (no. of computations = no. of d.o.f. of x_0)

ALTERNATIVELY

For a converged solution at $t = 0$ we can write

$$\mathbf{u} = K\mathbf{x}_0 = -\frac{1}{l^2}R^{-1}B^H\mathbf{p}_0.$$

This is a **linear relation** between the **input x_0** and **output \mathbf{u}** .

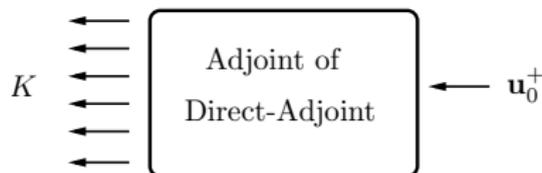


The input has a **large** dimension and the output a **small** dimension.

Riccati-less optimal control II

Such a problem is efficiently solved using the adjoint equations.

The adjoint input has a **small** dimension and the output a **large** dimension.



K is obtained from the solution of the **adjoint** of the **direct-adjoint system**.

Adjoint of the Direct-Adjoint system I

Introduce the adjoint variables \mathbf{x}^+ and \mathbf{p}^+ and multiply with the direct-adjoint equations, then integrate in time from $t = 0$ to $t = T$. Obs! here we consider that \mathbf{u} has dimension $m = 1$.

$$\int_0^T \mathbf{x}^{+H} \left(\frac{d\mathbf{x}}{dt} - A\mathbf{x} + \frac{1}{l^2} B R^{-1} B^H \mathbf{p} \right) dt + \int_0^T \mathbf{p}^{+H} \left(\frac{d\mathbf{p}}{dt} + A^H \mathbf{p} + Q\mathbf{x} \right) dt = 0.$$

Adjoint of the Direct-Adjoint system II

Using integration by parts, and considering that both R and Q are symmetric, we obtain

$$\begin{aligned}
 & - \int_0^T \mathbf{p}^H \left(\frac{d\mathbf{p}^+}{dt} - A\mathbf{p}^+ - \frac{1}{l^2} BR^{-1}B^H\mathbf{x}^+ \right) dt - \int_0^T \mathbf{x}^H \left(\frac{d\mathbf{x}^+}{dt} + A^H\mathbf{x}^+ - Q\mathbf{p}^+ \right) dt \\
 & + [\mathbf{p}^H \mathbf{p}^+]_0^T + [\mathbf{x}^H \mathbf{x}^+]_0^T = 0.
 \end{aligned}$$

If we now define the **new** adjoint equations as

Adjoint of the Direct-Adjoint system III

Using integration by parts, and considering that both R and Q are symmetric, we obtain

$$\begin{aligned}
 & - \int_0^T \mathbf{p}^H \left(\underbrace{\frac{d\mathbf{p}^+}{dt} - A\mathbf{p}^+ - \frac{1}{J^2} BR^{-1} B^H \mathbf{x}^+}_{=0} \right) dt - \int_0^T \mathbf{x}^H \left(\underbrace{\frac{d\mathbf{x}^+}{dt} + A^H \mathbf{x}^+ - Q\mathbf{p}^+}_{=0} \right) dt \\
 & \quad + [\mathbf{p}^H \mathbf{p}^+]_0^T + [\mathbf{x}^H \mathbf{x}^+]_0^T = 0.
 \end{aligned}$$

If we now define the **new** adjoint equations as

$$\begin{aligned}
 \frac{d\mathbf{p}^+}{dt} &= A\mathbf{p}^+ + \frac{1}{J^2} BR^{-1} B^H \mathbf{x}^+, \\
 \frac{d\mathbf{x}^+}{dt} &= -A^H \mathbf{x}^+ + Q\mathbf{p}^+,
 \end{aligned}$$

Adjoint of the Direct-Adjoint system IV

with $\mathbf{x}^+(t = T) = 0$ and $\mathbf{p}(t = T) = 0$, the remaining terms are

$$\mathbf{x}^{+H}(0)\mathbf{x}(0) + \mathbf{p}^{+H}(0)\mathbf{p}(0) = 0.$$

Recall that the original linear relation was

$$K\mathbf{x}_0 = -\frac{1}{j^2}R^{-1}B^H\mathbf{p}_0$$

- Choosing $\mathbf{p}^{+H}(t = 0)$ as one row of $-\frac{1}{j^2}R^{-1}B^H$ ($m = 1$)
- we can **identify** one row of K as $\mathbf{x}^{+H}(0)$. ($m = 1$)

Riccati-less optimal control: solution procedure

If we let $\mathbf{x}^+ \rightarrow -\mathbf{p}$ and $\mathbf{p}^+ \rightarrow \mathbf{x}$ we easily obtain the original (Direct-Adjoint) system. (self-adjoint), see *Semeraro et al. (2013)*

Finally: solve the **original** linear system with **new** b.c.

$$\frac{d\mathbf{x}}{dt} = A\mathbf{x} - \frac{1}{l^2}BR^{-1}B^H\mathbf{p} \quad \text{on } 0 < t < T, \quad \mathbf{x}^H(0) \text{ is one row of } \frac{1}{l^2}R^{-1}B^H,$$

$$\frac{d\mathbf{p}}{dt} = -A^H\mathbf{p} - Q\mathbf{x} \quad \text{on } 0 < t < T, \quad \text{with } \mathbf{p}(T) = 0.$$

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Avoid solving $X_{n \times n}$

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IMPORTANT

Avoid solving $X_{n \times n}$ solve original system $\mathbf{x}_{n \times 1}$ m times

Applications

Applications

MCE: Minimal Control Energy

In this method the feedback matrix K is evaluated from the **unstable** open-loop solutions of the system.

Case: control of the cylinder wake (globally unstable flow)

Refs:

- *Carini, Pralits, Luchini, JFS, 2013*

ADA: Adjoint of the Direct-Adjoint

This method is more general and does not depend on whether the system is unstable or not.

Cases: control of the cylinder wake, boundary layer transition

Refs:

- *Pralits, Luchini, IUTAM Proceeding, 2010*
- *Semeraro, Pralits, Rowley, Henningson, JFM, 2013*

Control of the cylinder wake

Control strategy

MCE & ADA

Numerical procedure

- All equations are discretized using second-order finite-differences over a staggered, stretched, Cartesian mesh.
- An immersed-boundary technique is used to enforce the boundary conditions on the cylinder.
- The nonlinear mean-flow equations, along with their boundary conditions, are solved by a Newton-Raphson procedure.
- The linear and nonlinear evolution equations are solved using Adams-Bashforth/Crank-Nicholson.
- The eigenvalue problems are solved using an Inverse Iteration algorithm
- Discrete adjoint equations (accurate to machine precision).

Cases:

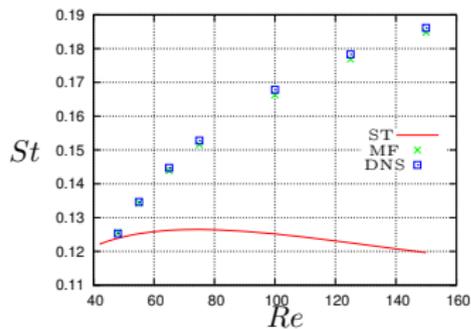
Reynolds numbers close to the first bifurcation, two-dimensional flow

MCE

The linear feedback matrix K which suppresses vortex shedding from a circular cylinder has been computed using:

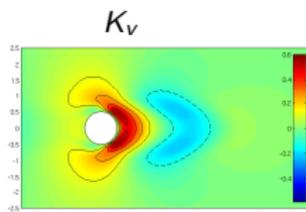
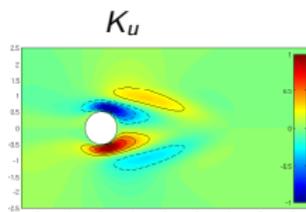
- Full state information
- Actuator: angular oscillation
- $Re = UD/\nu$
- $l^2 \rightarrow \infty$

Dimension of control \mathbf{u} is $m = 1$

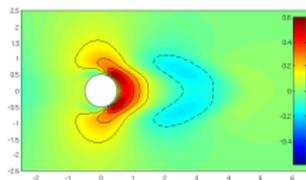
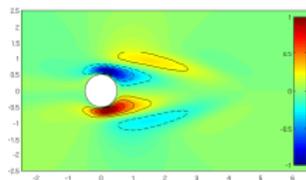


The feedback matrix K ($\mathbf{u} = K\mathbf{x}$)

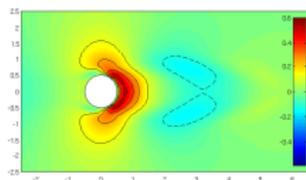
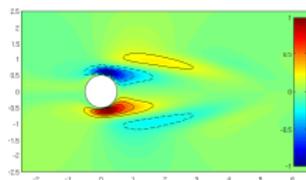
- $Re = 55$



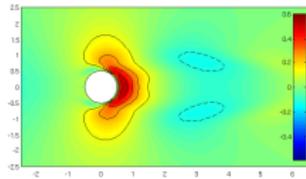
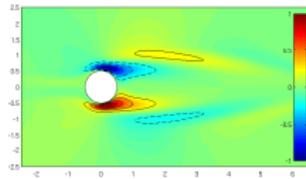
- $Re = 75$



- $Re = 100$



- $Re = 150$

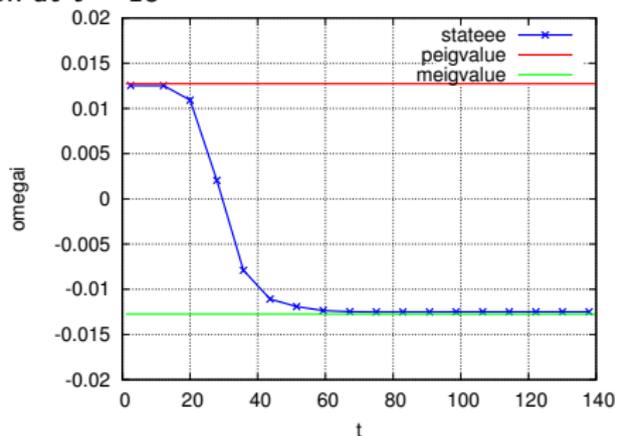
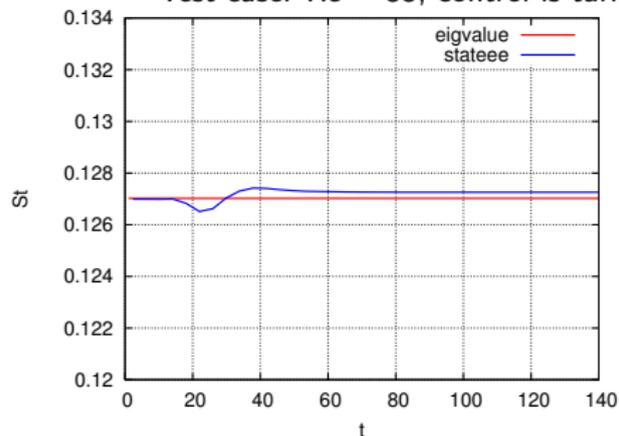


Results: linearized N-S equations

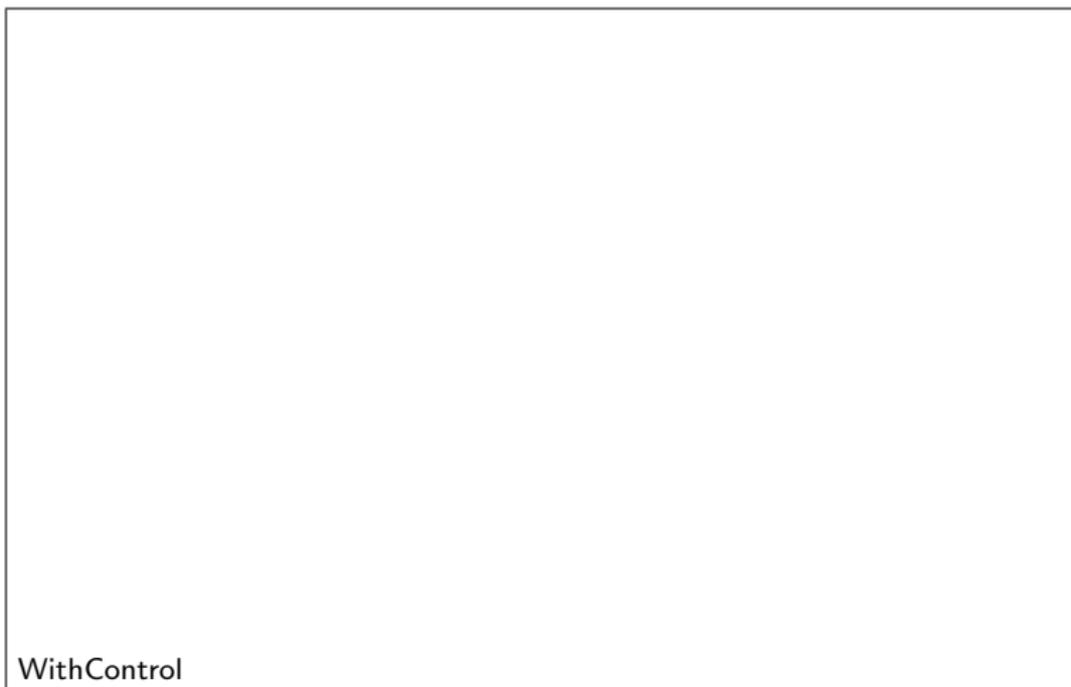
The temporal evolution of the frequency and growth rate is compared with the eigenvalue λ

- The Strouhal number: $St = fD/U$ compared to $St = \lambda_r/2\pi$
- The growth rate: $\sigma = \frac{d}{dt} \log(u(t))$ compared to λ_i

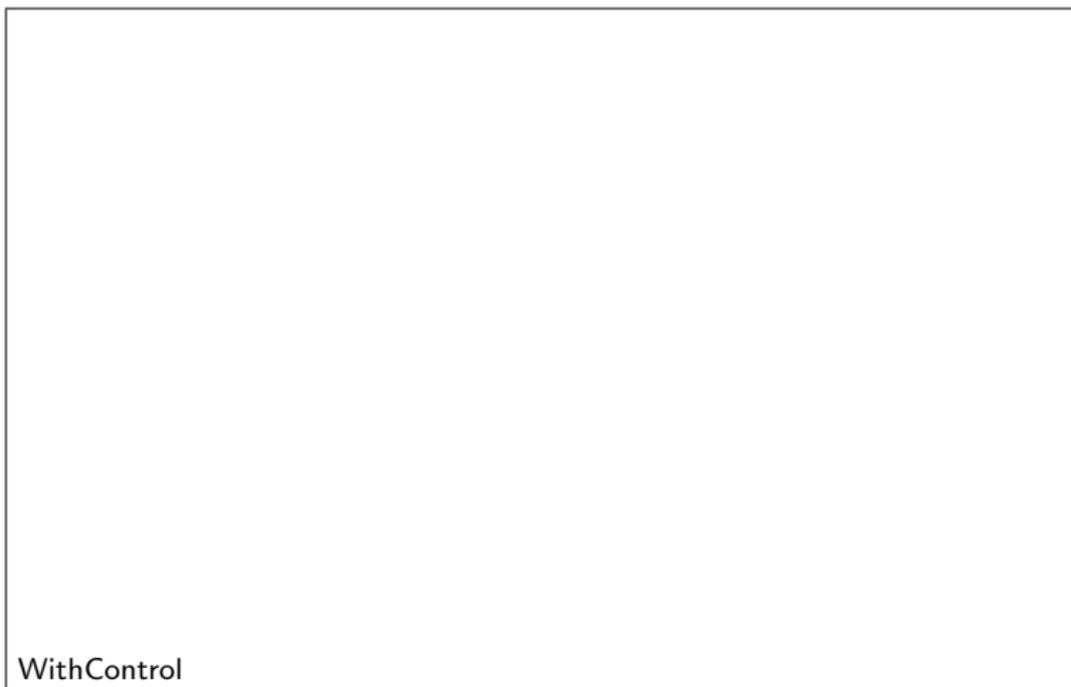
Test case: $Re = 55$, control is turned on at $t = 18$



Control of vortex shedding: $Re = 55$

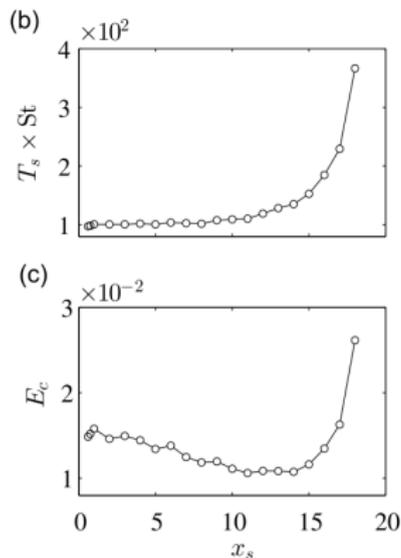
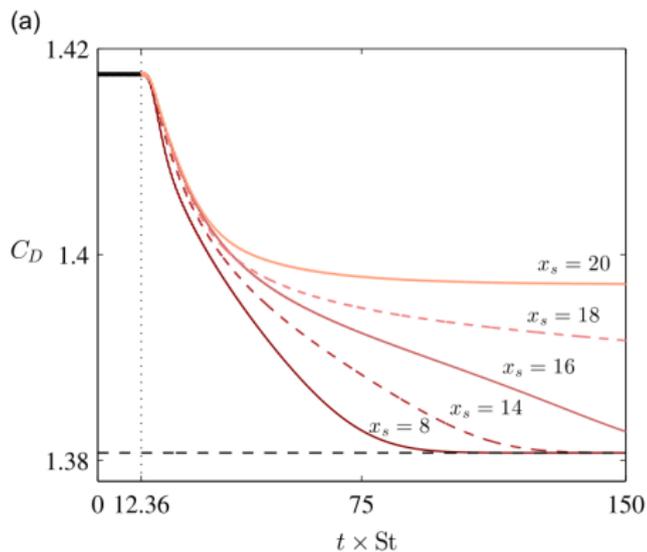


Control of vortex shedding: $Re = 55$



Using a single sensor

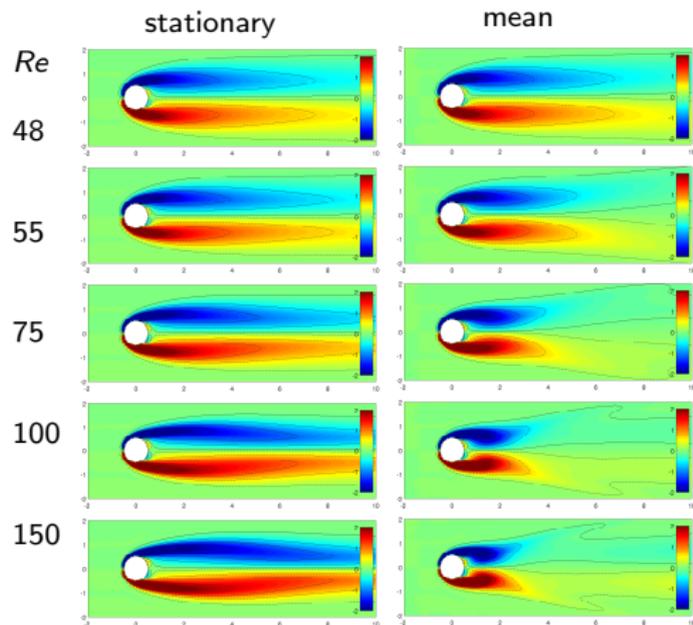
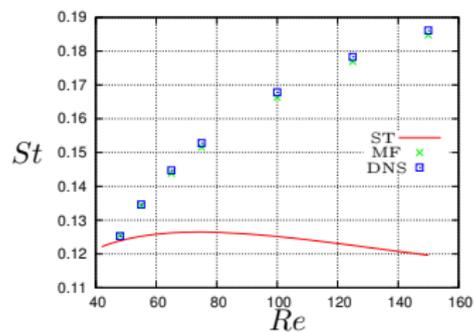
- $Re = 50$
- Actuator: **angular oscillation**
- Sensor: **one** on the symmetry line at a position x_s from the cylinder center.



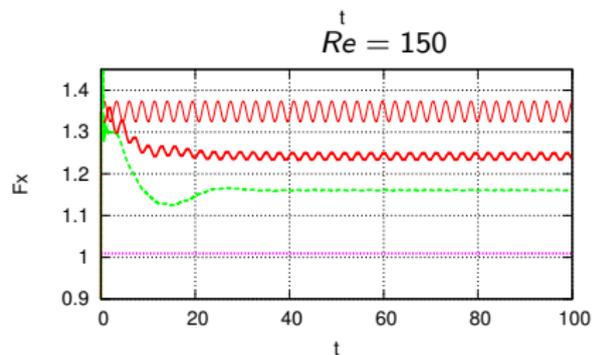
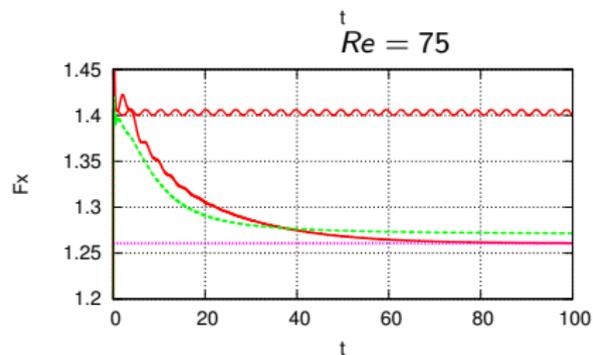
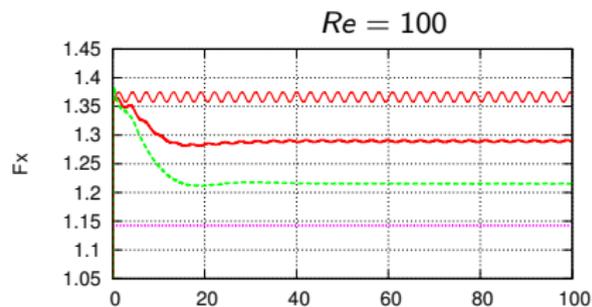
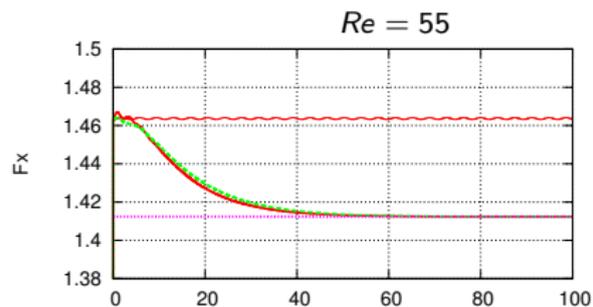
A **trade off** is found between **actuation time** and **minimal control energy** used.

Stationary vs. mean flow

St for **limit cycle** coincide with **mean-flow** eigenfrequency



Control of vortex shedding: stationary vs. mean

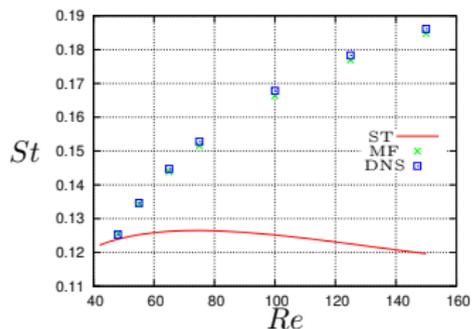


Red: Stationary baseflow, **Green:** Meanflow, **Purple:** Stabilized solution

ADA

The linear feedback matrix K which suppresses vortex shedding from a circular cylinder has been computed using:

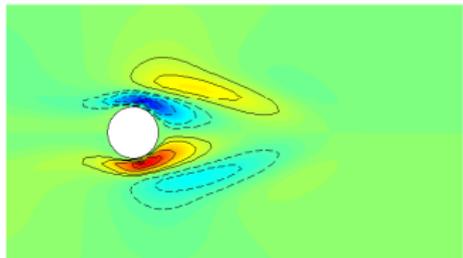
- Full state information
- Actuator: angular oscillation
- l^2 can take **any** value



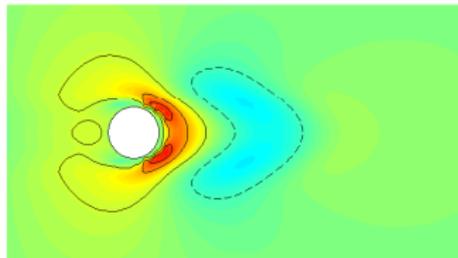
Results: K for $Re = 55$

Comparison with MCE ($l^2 \rightarrow \infty$)

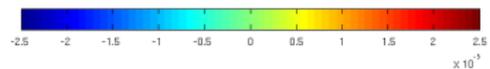
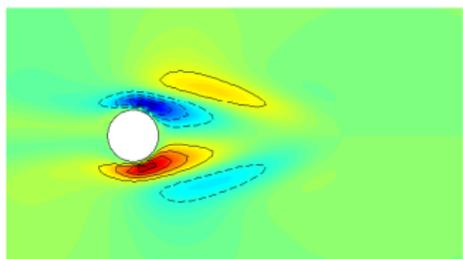
$$K_u, l^2 = 1$$



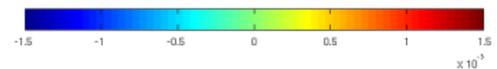
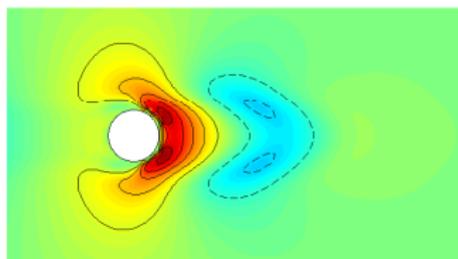
$$K_v, l^2 = 1$$



$$K_u, l^2 \rightarrow \infty$$



$$K_v, l^2 \rightarrow \infty$$

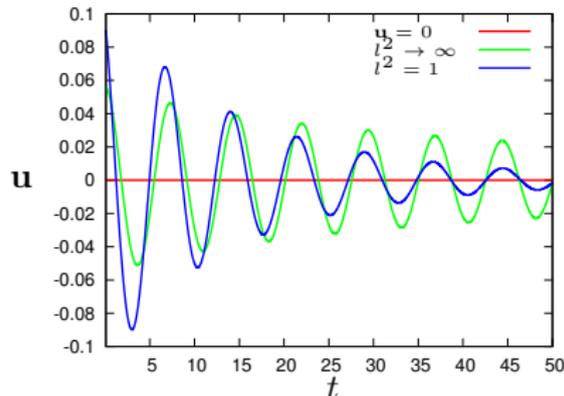
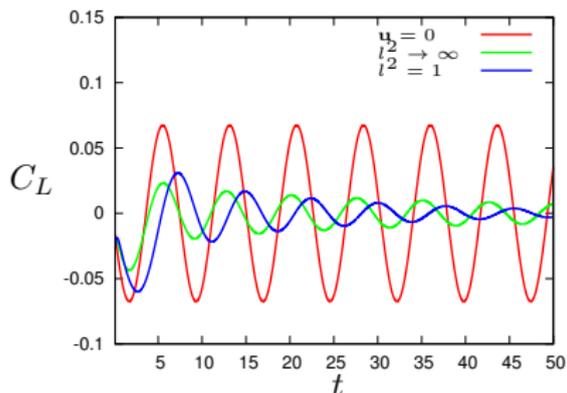


Control of vortex shedding

In the temporal evolution of the lift (C_L) and control \mathbf{u} :

- C_L and \mathbf{u} tend to zero as the control is applied
- Control \mathbf{u} strengthens as l^2 decrease

Test case: $Re = 55$, control is turned on at $t = 0$

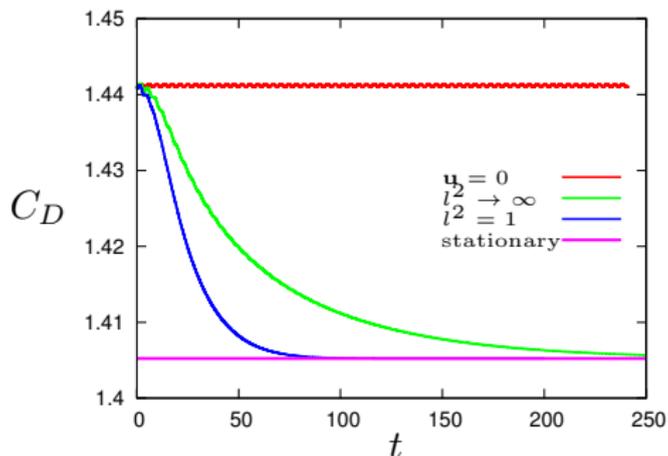


Control of vortex shedding

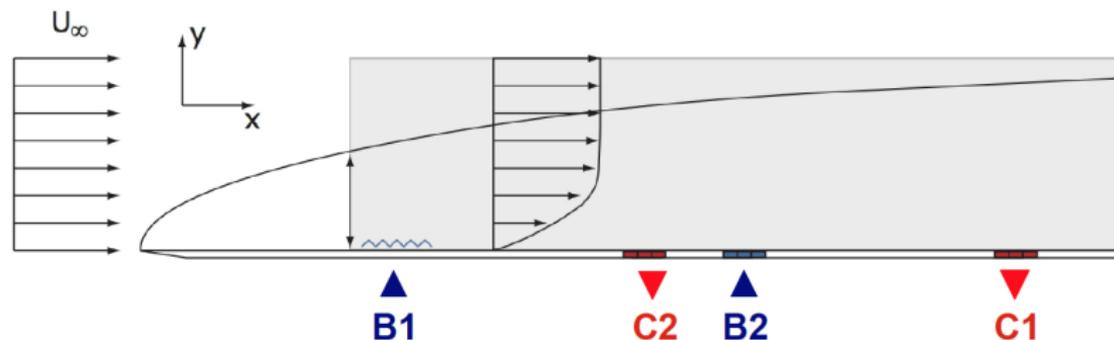
In the temporal evolution of drag (C_D) coefficient:

- As the control is applied C_D tends to the constant value corresponding to the steady state solution
- The control acts more quickly as l^2 is decreased

Test case: $Re = 55$, control is turned on at $t = 0$



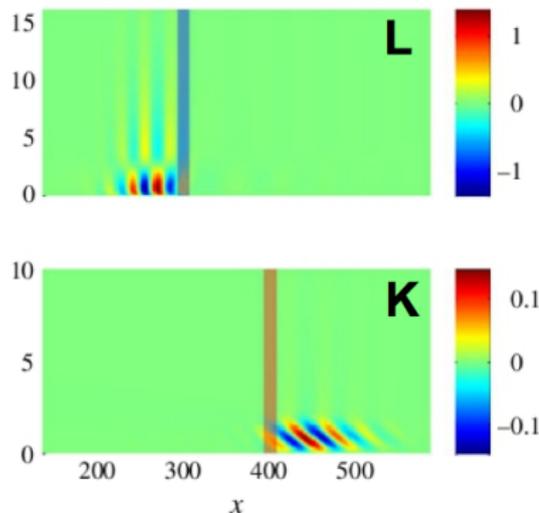
Control of the flat plate boundary layer



- Actuators are modelled as volume forcing
- Sensors are localized measurements of the flow

Control of the flat plate boundary layer

Closed-loop system



$$\begin{aligned}\frac{\partial \mathbf{u}}{\partial t} &= \mathbf{A}\mathbf{u} + \mathbf{B}_1\mathbf{w} + \mathbf{B}_2\phi \\ \mathbf{z} &= \mathbf{C}_1\mathbf{u} \\ \psi &= \mathbf{C}_2\mathbf{u}\end{aligned}$$

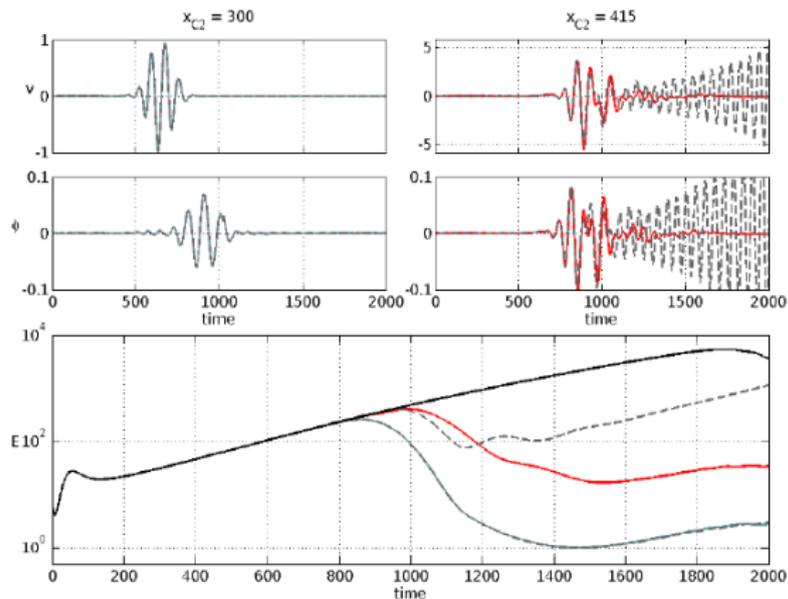
+

$$\begin{aligned}\frac{\partial \hat{\mathbf{u}}}{\partial t} &= \mathbf{A}\hat{\mathbf{u}} + \mathbf{B}_1\mathbf{w} - \mathbf{L}\mathbf{C}_2(\mathbf{u} - \hat{\mathbf{u}}) \\ \phi &= \mathbf{K}\hat{\mathbf{u}}\end{aligned}$$

Full-order design for optimal control (an iterative approach)

Pralits & Luchini (IUTAM 2010)
Semeraro et al. (JFM 2013b)

Control of the flat plate boundary layer



Feedback - Full dimensional (sensor downstream)

Feedforward - Full dimensional (sensor upstream)

Model reduction

Semeraro, Pralits, Rowley, Henningson, JFM, 2013

Stability, Sensitivity and Control of Turbulent Wakes

Work performed within the CARPE project by:

- **Marco Carini**
- **Christophe Airiau**
- **Jan O. Pralits**

CARPE (Contrôle Actif Robuste d'Écoulement de Plaque Epaisse)

Background

There is an increased interest in understanding the **linear dynamics** of **mean-flow** velocity profiles since treating the fully resolved turbulent flow (dns) is unfeasible (large quantities of data, not realistic Re , strong nonlinear effects, chaotic flows, ...).

It has been seen numerically and experimentally that the **leading frequency** of many **oscillator flows** (eg. wakes, cavities, ...) compares well between **dns** (or experiments) with the stability analysis made on the **mean-flow** profiles, see eg. *Sipp & Lebedev (2007)*, *Barkley (2006)*.

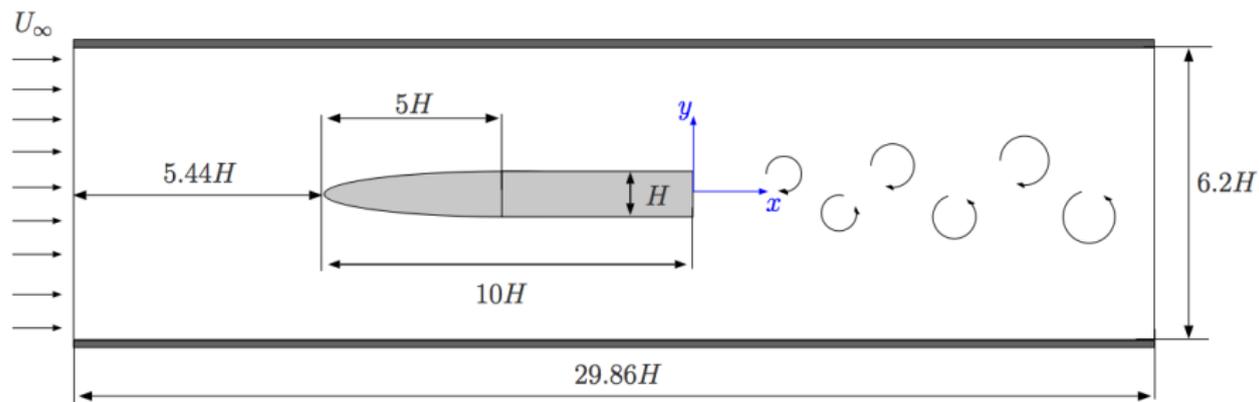
Recently different groups have continued to analyze this remarkable findings and several explanations exist: eg. Mantic-Lugo et al. (2015), Marquet & Lesshaft (2015), ...

Moreover, numerical **stability- and sensitivity analysis** has been performed including/excluding the linearization of the **Reynolds-stress terms** when using the **mean flow**: Meliga et al. (2012,2014), Mettot et al. (2014a,b).

Also experimental sensitivity analysis has been performed recently on high Re flows: Parezanovic & Cadot (2012), Grandemange (2012),...

Here the idea is to evaluate if it is possible to extend what we know about **optimal control theory** to **high Re** cases by linearizing around the **mean flow**.

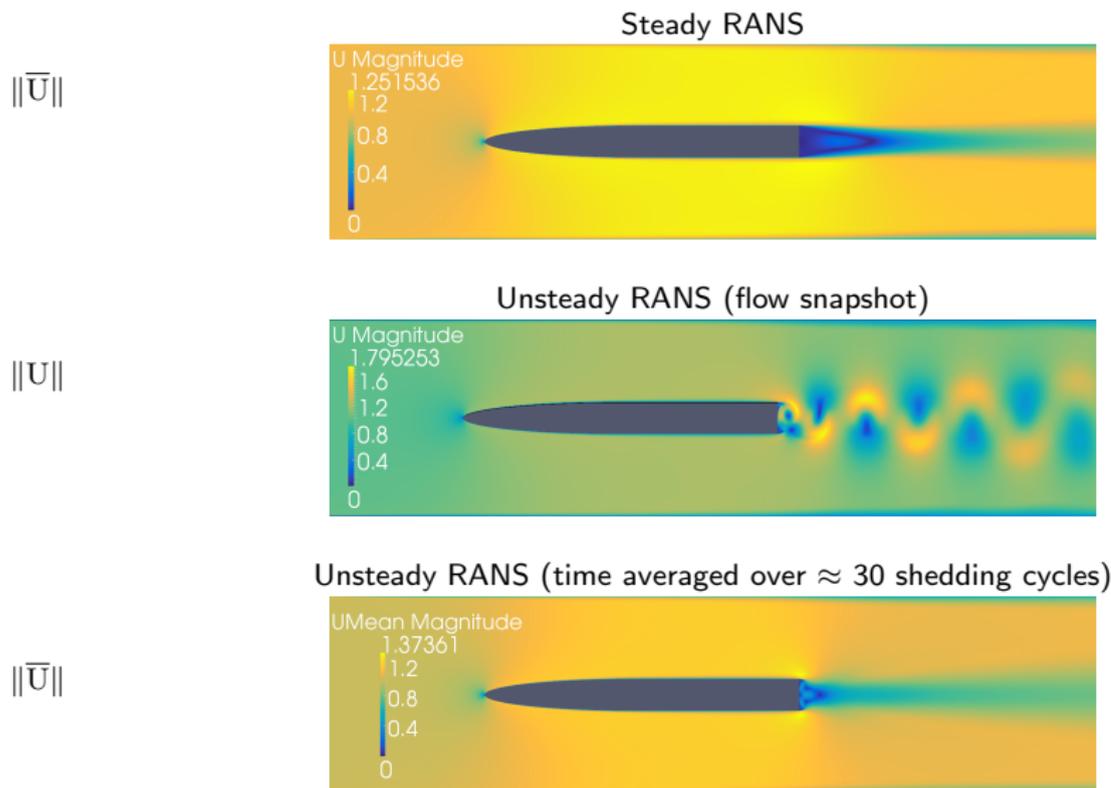
Flow configuration



Incompressible wind-tunnel flow past a thick flat plate with elliptic leading edge:

- $AR = 10$
- $Re = U_\infty H / \nu = 32000$
- at the inlet $Tu = 2\%$ and $\nu_t / \nu = 100$.

Base flow



Global stability of mean flows

Mettot, Sipp & Bézard, PoF 2014

Linearised Navier-Stokes eqs. around the mean flow $\bar{\mathbf{U}}(\mathbf{x})$:

$$\lambda \hat{\mathbf{u}} + (\bar{\mathbf{U}} \cdot \nabla) \hat{\mathbf{u}} + (\hat{\mathbf{u}} \cdot \nabla) \bar{\mathbf{U}} - \tilde{\nu}_{eff} \nabla \cdot (\nabla \hat{\mathbf{u}} + \nabla \hat{\mathbf{u}}^T) + \nabla \hat{p} = \mathbf{0},$$

$$\nabla \cdot \hat{\mathbf{u}} = 0.$$

+ boundary conditions with homogeneous data.

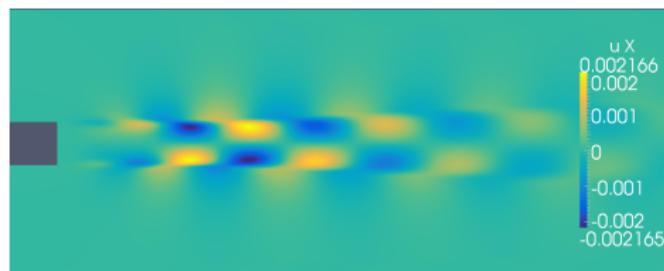
Two approaches:

- **Quasi-laminar:** $\tilde{\nu}_{eff} = \nu$ (*laminar viscosity only*)
- **Quasi-laminar Mixed:** $\tilde{\nu}_{eff} = \nu + \nu_t$ (*laminar+frozen turbulent viscosity*)

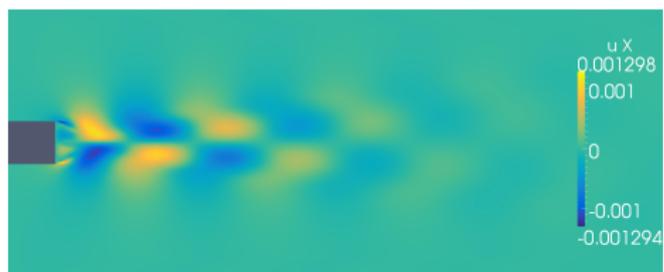
Direct global mode

Real part of the velocity x -component

	URANS	LSA-S-RANS	LSA-U-RANS
St	0.275	0.276	0.272



S-RANS



U-RANS

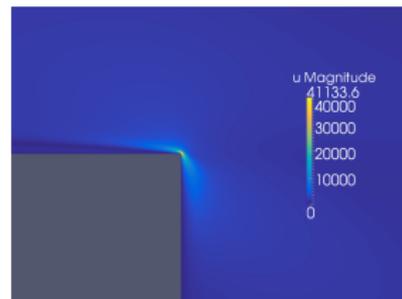
Adjoint global mode

Magnitude of the real part of the velocity field (**Saturated colormap**)

S-RANS



U-RANS



Linear Optimal Control

Infinite Horizon

- Linear system dynamics (*descriptor form*)

$$E \frac{d\mathbf{q}}{dt} = A\mathbf{q}(t) + B\mathbf{q}(t).$$

- Full-information feedback control $\mathbf{u} = K\mathbf{u} \Rightarrow$ solution of a **Riccati eq.**
- Numerical solution of Riccati eq. is **unfeasible** for large-scale plants, i.e. fluid plant.

Minimal Control Energy solution

- Reflection of the unstable spectrum.
- Only the **unstable left eigenmode** is required to **exactly compute** K

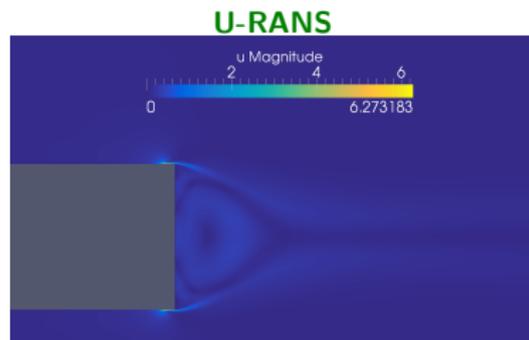
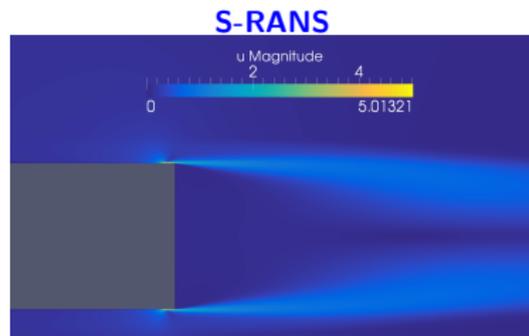
Control implementation

Continuous setting

- Boundary control via anti-symmetric blowing/suction (zero net-mass injection) located at $x \in [-0.1, -0.05]$.
- **Lifting procedure** to recover the standard plant definition.
- Additional time integration \Rightarrow PI control

$$\mathbf{u}(t) = \int_0^t \int_{\Omega} \mathbf{k} \cdot \mathbf{u} d\Omega dt$$

where $\mathbf{k}(\mathbf{x})$ is the continuous control gain field.



'Control mode' resulting from the lifting procedure.

Continuous gain field

Magnitude of the vector field



S-RANS



U-RANS

The gain field $\mathbf{k}(\mathbf{x})$ shares the same spatial structure of $\hat{\mathbf{u}}^\dagger$ with sharp peaks close to the t.e.

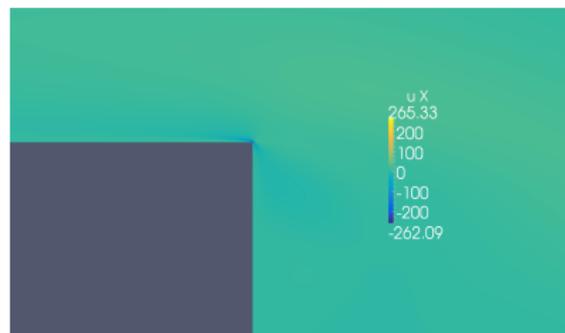
Continuous gain field

Vector field components

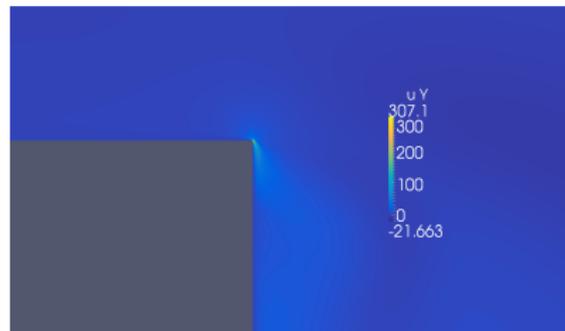
S-RANS

U-RANS

$k_u(\mathbf{x})$



$k_v(\mathbf{x})$



Appendix

Some numerical issues

Continuous vs. Discrete Adjoint Equations

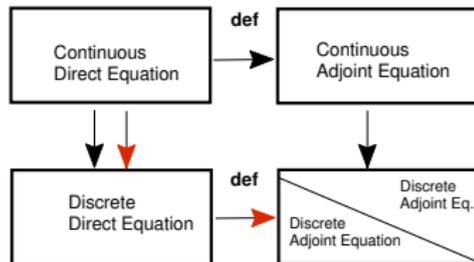
The adjoint equations can be derived using two different approaches.

Both with advantages and disadvantages.

By definition we have

$$\langle p, Lx \rangle = \langle L^* p, x \rangle + \text{B.T.}$$

- Continuous approach** \rightarrow : The adjoint equations are derived by definition using the continuous direct equations.
 - + Straightforward derivation, reuse old code when programming
 - Accuracy depends on discretization, difficulties with boundary conditions
- Discrete approach** \rightarrow : The adjoint equations are derived from the discretized direct equations.
 - + Accuracy can be achieved close to machine precision, and can be independent of discretization !!
 - Tricky derivation, usually requires making a new code, or larger changes of an existing code.



Here "def" means definition of the adjoint operator.

In the top row it is on continuous form while in the bottom row it is on discrete form.

Derivation of the adjoint equation I

Consider the following optimal control problem (ODE) where ϕ is the state and g the control.

$$\frac{d\phi(t)}{dt} = -A\phi(t) + Bg(t), \quad \text{for } 0 \leq t \leq T,$$

with initial condition

$$\phi(0) = \phi_0$$

We can now define an optimization problem in which the goal is to find an optimal $g(t)$ by minimizing the following objective function

$$J = \frac{\gamma_1}{2} [\phi(T) - \Psi]^2 + \frac{\gamma_2}{2} \int_0^T g(t)^2 dt,$$

Derivation of the adjoint equation II

Continuous approach

We can solve this problem using an **adjoint identity** approach or by introducing **Lagrange multipliers**.

$$\int_0^T a \left[\frac{d\phi}{dt} + A\phi - Bg \right] dt = \int_0^T \left[-\frac{da}{dt} + A^*a \right] \phi dt - \int_0^T aBg dt + a(T)\phi(T) - a(0)\phi(0).$$

If we now define the adjoint equation as $-da/dt = -A^*a$ with an arbitrary initial condition $a(T)$ then the identity reduces to

$$\text{LHS} = - \int_0^T aBg dt + a(T)\phi(T) - a(0)\phi(0)$$

By definition the Left Hand Side is identically zero but this is exactly what must be checked numerically, i.e. $\text{error} = |\text{LHS}|$.

Derivation of the adjoint equation III

The gradient of J w.r.t. g can be derived considering the J is nonlinear in ϕ and g . We linearise by $\phi \rightarrow \phi + \delta\phi$, $g \rightarrow g + \delta g$ and then write the linearised objective function as

$$\gamma_1[\phi(T) - \Psi]\delta\phi(T) = \delta J - \gamma_2 \int_0^T g \delta g dt,$$

If we choose $a(T) = \gamma_1[\phi(T) - \Psi]$ then the equation for δJ can be substituted into the expression for the adjoint identity. If you further define the adjoint equations, remember that $\delta\phi(0) = 0$, then the final identity is written

$$\delta J = \int_0^T [\gamma_2 g + B^* a] \delta g dt$$

The adjoint equations and gradient of J w.r.t. g are written

$$-\frac{da}{dt} + A^* a, \quad a(T) = \gamma_1[\phi(T) - \Psi], \quad \text{and} \quad \nabla J_g = \gamma_2 g + B^* a.$$

The so called optimality condition is given by $\nabla J_g = 0$.

Derivation of the adjoint equation IV

- The accuracy of the adjoint solution is important since it quantifies a "gradient" in the optimization problem.
- The "error" must be evaluated to quantify the accuracy the adjoint solution.
- Note that the adjoint solution depends on the resolution (Δt), and likewise the accuracy.
- **Can we do better ?**

Derivation of the adjoint equation V

Discrete approach

A discrete version of the direct equation is written

$$\frac{\phi^{i+1} - \phi^i}{\Delta t} = -A\phi^i + Bg^i, \quad \text{for } i = 1, \dots, N-1,$$

where N denotes the number of discrete points on the interval $[0, T]$, Δt is the constant time step, and

$$\phi^1 = \phi_0,$$

is the initial condition. This can be written as a discrete evolution equation

$$\phi^{i+1} = [I - \Delta t A]\phi^i + \Delta t Bg^i, \quad \text{for } i = 1, \dots, N-1.$$

A discrete version of the objective function can be written

$$J = \frac{\gamma_1}{2} (\phi^N - \Phi)^2 + \frac{\gamma_2}{2} \sum_{i=1}^{N-1} \Delta t (g^i)^2.$$

An adjoint variable a^i is introduced defined on $i = 1, \dots, N$ and by definition

$$a^{i+1} \cdot L\phi^i = (L^* a^{i+1}) \cdot \phi^i, \quad \text{for } i = 1, \dots, N-1.$$

We then introduce the definition of the state equation on the left hand side of and impose that

$$a^i = L^* a^{i+1} \quad \text{for } i = N-1, \dots, 1.$$

This is the discrete adjoint equation. Using the discrete direct and adjoint yields

$$a^{i+1} \cdot (\phi^{i+1} - \Delta t Bg^i) = a^i \cdot \phi^i, \quad \text{for } i = 1, \dots, N-1$$

which must be valid for any ϕ and a . An error can therefore be written as

$$\text{error} = |a^N \cdot \phi^N - a^1 \cdot \phi^1 - \sum_{i=1}^{N-1} \Delta t a^{i+1} \cdot Bg^i|.$$

Derivation of the adjoint equation VI

The discrete optimality condition is then derived. Since J is nonlinear with respect to ϕ and g we must first linearize. This can be written

$$\delta J = \gamma_1(\phi^N - \Phi) \cdot \delta\phi^N + \gamma_2 \sum_{i=1}^{N-1} \Delta t g^i \cdot \delta g^i.$$

We now choose the terminal condition of the adjoint as $a^N = \gamma_1(\phi^N - \Phi)$ and substitute this expression into the discrete adjoint identity. This is written

$$\gamma_1(\phi^N - \Phi) \cdot \delta\phi^N = a^1 \cdot \delta\phi^1 + \sum_{i=1}^{N-1} \Delta t a^{i+1} \cdot B \delta g^i$$

By inspection one can see that the left hand side is identical to the first term in the expression for δJ , and $\delta\phi^1 = 0$. Rearranging the terms, we get

$$\delta J = \sum_{i=1}^{N-1} \Delta t (\gamma_2 g^i + B^* a^{i+1}) \cdot \delta g^i,$$

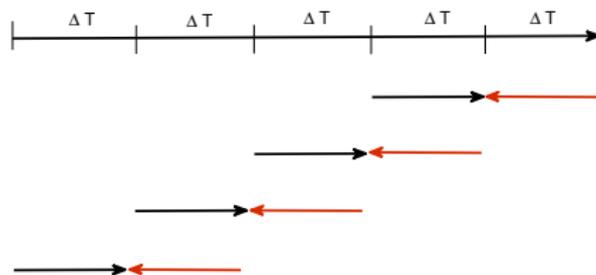
from which we get the discrete optimality condition

$$g^i = -\frac{1}{\gamma_2} B^* a^{i+1} \quad \text{for } i = 1, \dots, N-1.$$

Note that if B is a matrix then $B^* = B^T$.

Checkpointing algorithm

- When the adjoint equation is forced in time by the direct solution (ex. quadratic objective function), then this poses storage requirements (hard ware). This becomes a problem for 2D and 3D problems with high resolution in space and time.
- One way to come around this is to apply Checkpointing. This consists of sampling the direct solution at given rate and then recompute the direct solution for short time intervals when needed. This means in theory that one more solution of the direct system has been added to the computational effort.
- However, since it is common to use parallel computing, and processors is becoming a smaller problem on can do something to obtain the minimal required computational time.
- This is done by recomputing the direct solution, in parallel, while computing the adjoint.



Why introduce Adjoint equations ?

Example gradient computation:

$$J = \mathbf{w}^H \mathbf{x}, \quad \text{where} \quad \mathbf{A} \mathbf{x} = \mathbf{b}, \quad \text{Ex. find} \quad \frac{\partial J}{\partial \mathbf{b}}$$

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Example gradient computation:

$$J = \mathbf{w}^H \mathbf{x}, \quad \text{where } \mathbf{A} \mathbf{x} = \mathbf{b}, \quad \text{Ex. find } \frac{\partial J}{\partial \mathbf{b}}$$

Finite difference approach

$$\left(\frac{\partial J}{\partial \mathbf{b}} \right)_j \approx \frac{J(\mathbf{b} + \epsilon \mathbf{e}_j) - J(\mathbf{b})}{\epsilon}$$

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Requires $n + 1$ solutions of $\mathbf{A} \mathbf{x} = \mathbf{b}$, where n is the dimension of \mathbf{b}

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Alternatively, solve

$$\mathbf{A}^H \mathbf{p} = \mathbf{w} \quad \text{dual problem, adjoint}$$

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Requires $n + 1$ solutions of $\mathbf{Ax} = \mathbf{b}$, where n is the dimension of \mathbf{b}

Alternatively, solve

$$A^H \mathbf{p} = \mathbf{w} \quad \text{dual problem, adjoint}$$

then

$$J = \mathbf{w}^H \mathbf{x} = (A^H \mathbf{p})^H \mathbf{x} = \mathbf{p}^H \mathbf{Ax} = \mathbf{p}^H \mathbf{b},$$

Why introduce Adjoint equations ?

Example gradient computation:

$$J = \mathbf{w}^H \mathbf{x}, \quad \text{where } \mathbf{A} \mathbf{x} = \mathbf{b}, \quad \text{Ex. find } \frac{\partial J}{\partial \mathbf{b}}$$

Finite difference approach

$$\left(\frac{\partial J}{\partial \mathbf{b}} \right)_j \approx \frac{J(\mathbf{b} + \epsilon \mathbf{e}_j) - J(\mathbf{b})}{\epsilon}$$

Requires $n + 1$ solutions of $\mathbf{A} \mathbf{x} = \mathbf{b}$, where n is the dimension of \mathbf{b}

Alternatively, solve

$$\mathbf{A}^H \mathbf{p} = \mathbf{w} \quad \text{dual problem, adjoint}$$

then

$$J = \mathbf{w}^H \mathbf{x} = (\mathbf{A}^H \mathbf{p})^H \mathbf{x} = \mathbf{p}^H \mathbf{A} \mathbf{x} = \mathbf{p}^H \mathbf{b},$$

and

$$\frac{\partial J}{\partial \mathbf{b}} = \mathbf{p} \quad \text{One Solution, independently of } n$$

Background: control using rotational oscillation

Aim: reduce C_D

Exp. Tokumaru & Dimotakis (1991), **-20%**, $Re = 15000$

Feedback control:

Exp. Fujisawa & Nakabayashi (2002) **-16%** (**-70% C_L**), $Re = 20000$

Exp. Fujisawa et al.(2001) "**reduction**", $Re = 6700$

Optimal control (using adjoints):

Num. He et al.(2000) **-30** to **-60%** for $Re = 200 - 1000$

Num. Protas & Styczek (2002) **-7%** at $Re = 75$, **-15%** at $Re = 150$

Bergmann et al.(2005) **-25%** at $Re = 200$ (POD)

Aim: reduce vortex shedding

Feedback control:

Num. Protas (2004) **reduction**, "point vortex model", $Re = 75$

Optimal control (using adjoints):

Num. Homescu et al.(2002) **reduction**, $Re = 60 - 1000$

Minimal-energy control feedback

Denoting:

- \mathbf{x}^i and λ^i the i -th right eigenvector and eigenvalue of A ,
- \mathbf{y}^i and $-\lambda^{i*}$ the i -th right eigenvector and eigenvalue of $-A^H$,
- \mathbf{y}^{i*} is left eigenvector of A ,

we see that the stable eigenvectors of

$$\frac{\partial \mathbf{x}}{\partial t} = A\mathbf{x} + B\mathbf{u}, \quad \mathbf{u} = -R^{-1}B^H\mathbf{p}, \quad \frac{\partial \mathbf{p}}{\partial t} = -A^H\mathbf{p}$$

are of two possible types:

$$\begin{array}{ll} \mathbf{p} = 0, \mathbf{x} = \mathbf{x}^i & \text{if } \Re(\lambda^i) < 0 \quad (\text{stable}) \\ \mathbf{p} = \mathbf{y}^i, \mathbf{x} = (\lambda^{i*} + A)^{-1}BR^{-1}B^H\mathbf{y}^i & \text{if } \Re(\lambda^i) > 0 \quad (\text{unstable}) \end{array}$$

We now project an arbitrary initial condition \mathbf{x}_0 onto these modes,

$$\mathbf{x}_0 = \sum_{\text{stable}} d_j \mathbf{x}^j + \sum_{\text{unstable}} f_j (\lambda^{j*} + A)^{-1} B R^{-1} B^H \mathbf{y}^j \quad (3)$$

and note that in order to reconstruct \mathbf{p} we only need the f_j 's, because the stable modes have $\mathbf{p} = 0$. The coefficients d_j can be eliminated from (3) by projecting the left eigenvectors:

$$\mathbf{y}^{i*} \mathbf{x}_0 = \mathbf{y}^{i*} \sum_{\text{unstable}} f_j (\lambda^{j*} + A)^{-1} B R^{-1} B^H \mathbf{y}^j = \sum_{\text{unstable}} c_{ij} f_j$$

where, since \mathbf{y}^{i*} is also a left eigenvector of $(\lambda^{j*} + A)^{-1}$,

$$c_{ij} = \frac{\mathbf{y}^{i*} B R^{-1} B^H \mathbf{y}^j}{\lambda^i + \lambda^{j*}}$$

Only the **unstable eigenvalues** and **left eigenvectors** are needed.

The main theorem

Summarizing, the solution of the minimal-energy stabilizing control feedback problem can be written in terms of the unstable left eigenvectors only.

Theorem 1. *Consider a stabilizable system $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u}$ with no pure imaginary open-loop eigenvalues. Determine the unstable eigenvalues and corresponding left eigenvectors of \mathbf{A} such that $T_u^H \mathbf{A} = \Lambda_u T_u^H$ (equivalently, determine the unstable eigenvalues and corresponding right eigenvectors of \mathbf{A}^H such that $\mathbf{A}^H T_u = T_u \Lambda_u^H$). Define $\bar{\mathbf{B}}_u = T_u^H \mathbf{B}$ and $\mathbf{C} = \bar{\mathbf{B}}_u \bar{\mathbf{B}}_u^H$, and compute a matrix \mathbf{F} with elements $f_{ij} = c_{ij}/(\lambda_i + \lambda_j^*)$. The minimal-energy stabilizing feedback controller is then given by $\mathbf{u} = \mathbf{K}\mathbf{x}$, where $\mathbf{K} = -\bar{\mathbf{B}}_u^H \mathbf{F}^{-1} T_u^H$.*