



***EFFECT OF A POROELASTIC LAYER ON
LIFT AND DRAG OVER AN AIRFOIL***

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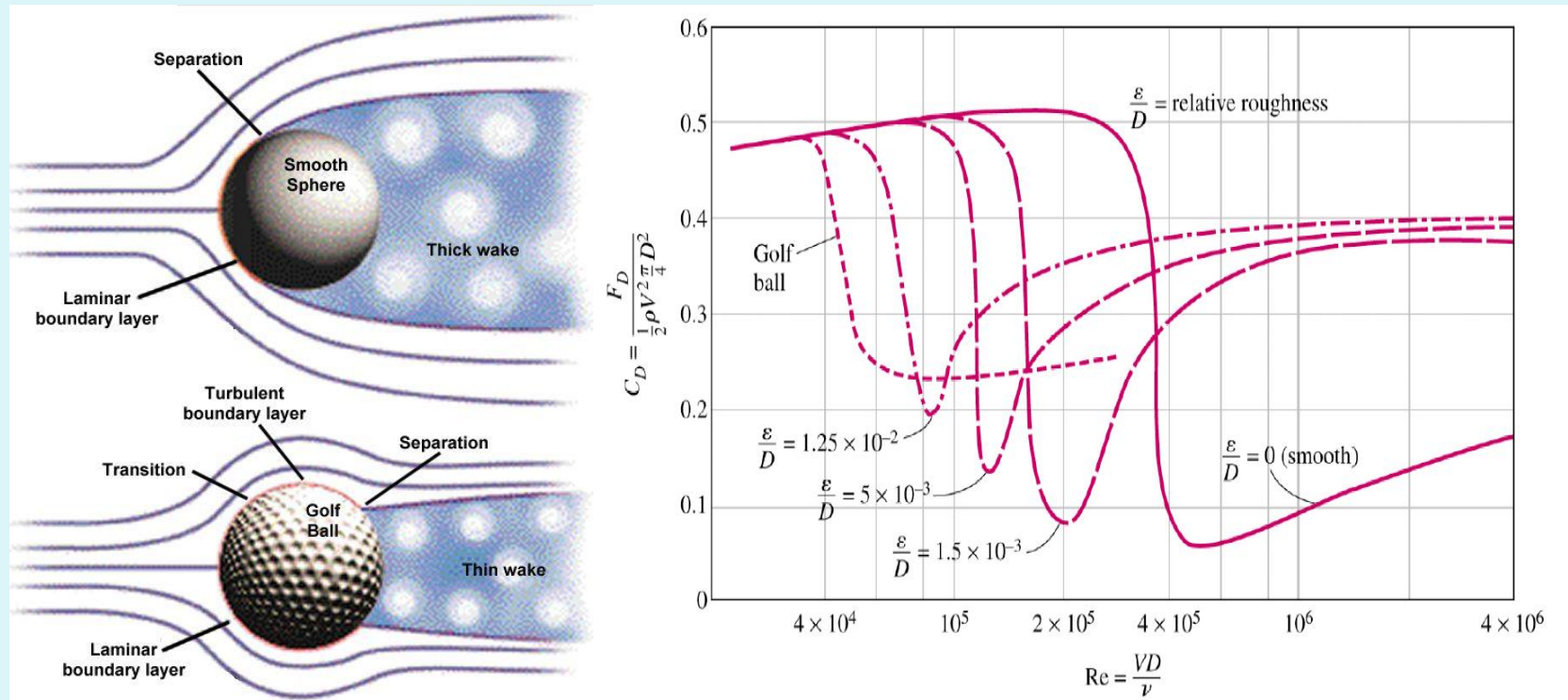
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EFFECT OF A POROELASTIC LAYER ON LIFT AND DRAG OVER AN AIRFOIL

Focus on passive actuators, what works, why it does, how can we select and “optimise” actuators for lift/drag purposes, etc.?

Reducing pressure drag by a passive technique ...

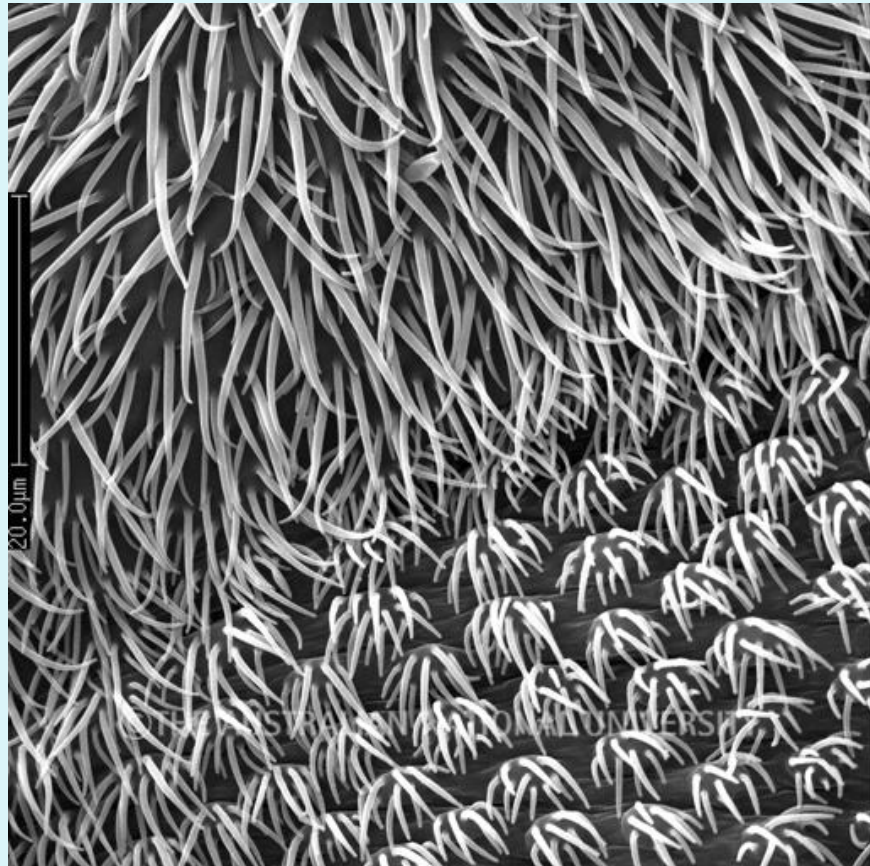


“Coverts” feathers ...

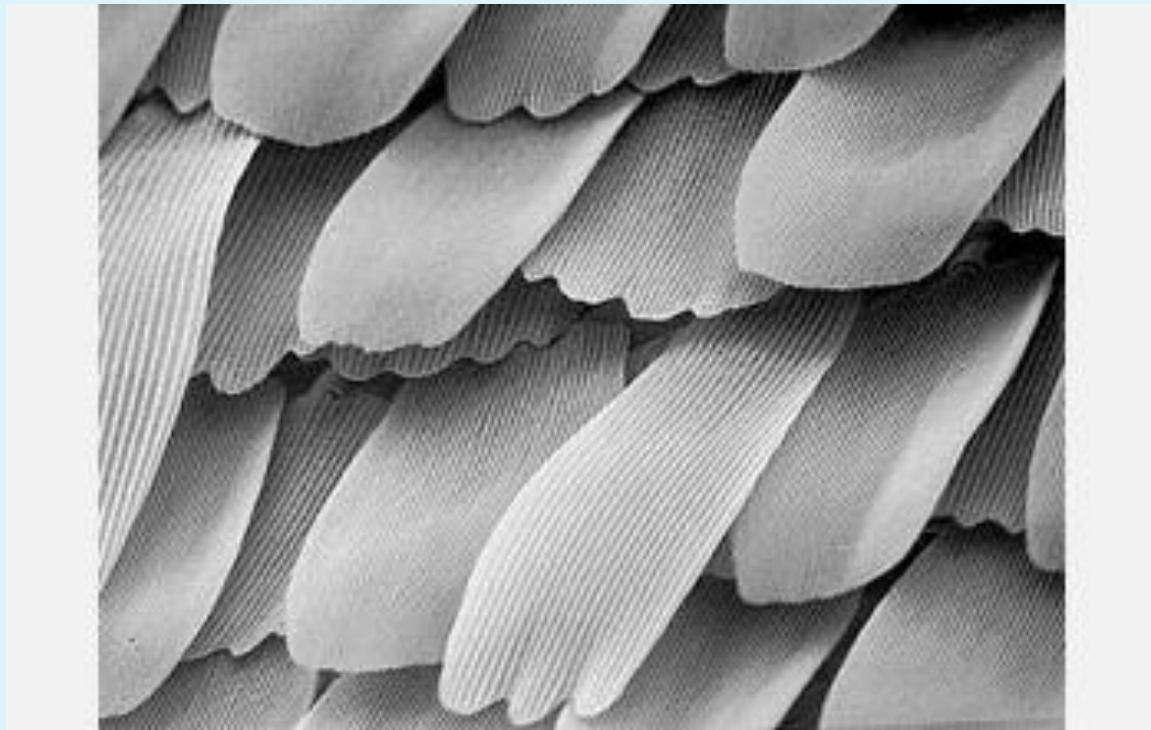


... and other appendages are the norm

mosquito body

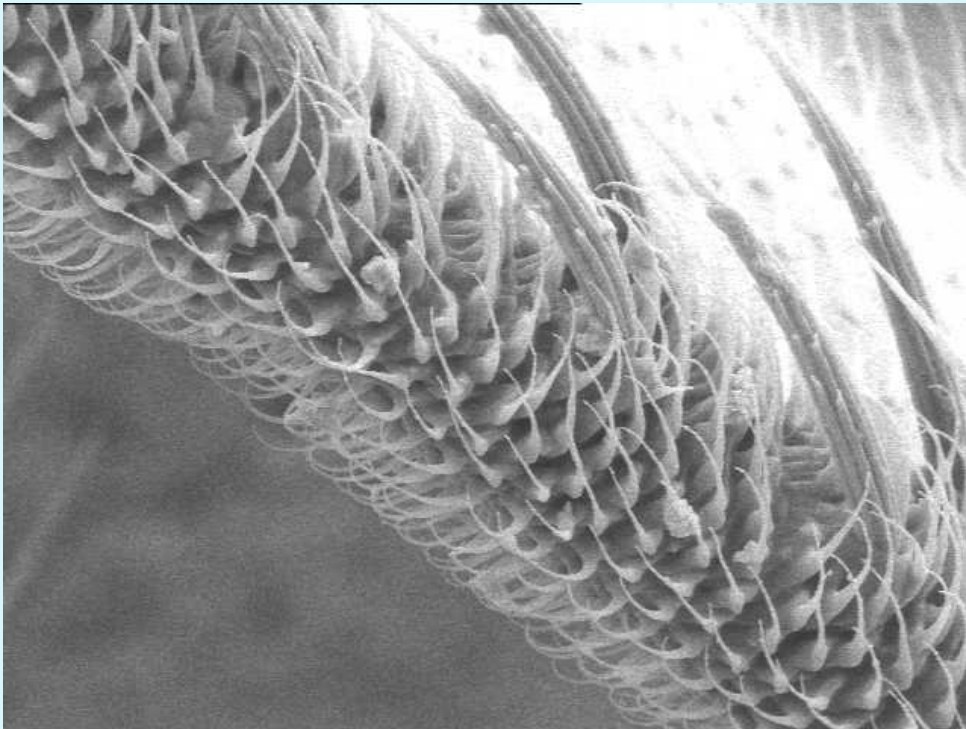


... and other appendages are the norm



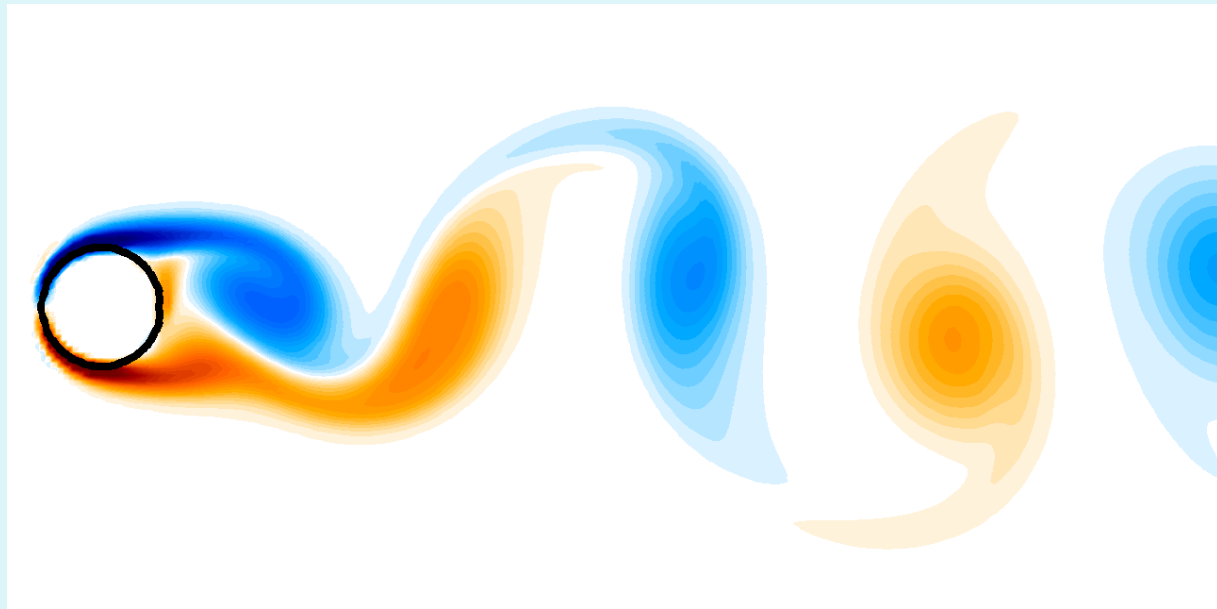
butterfly scales

... and other appendages are the norm



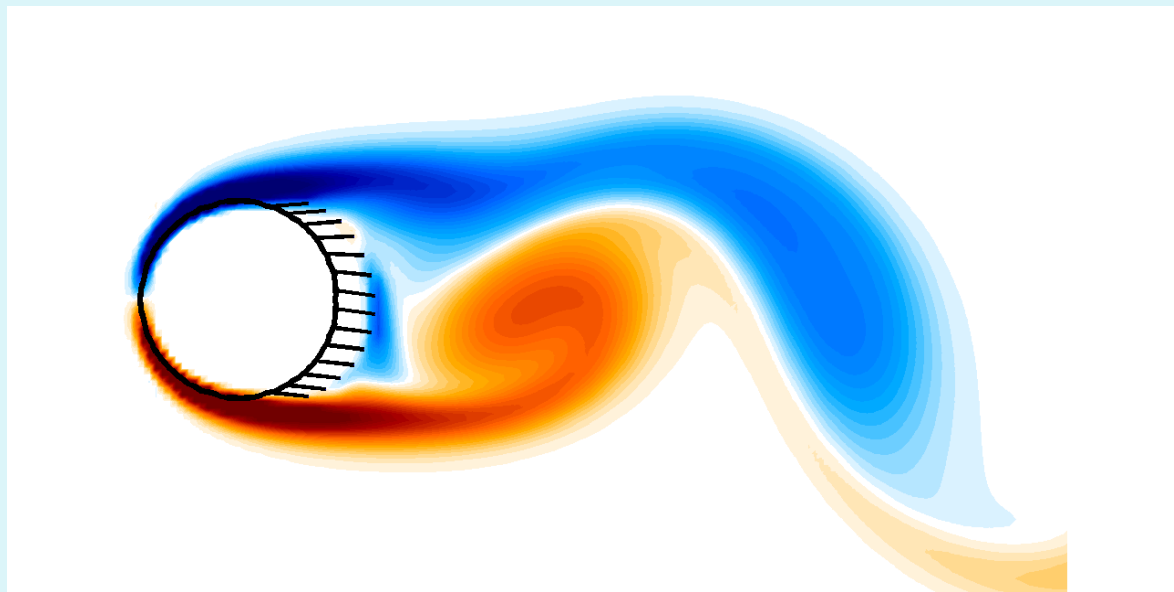
fly wing

Passive control via “append-actuators”



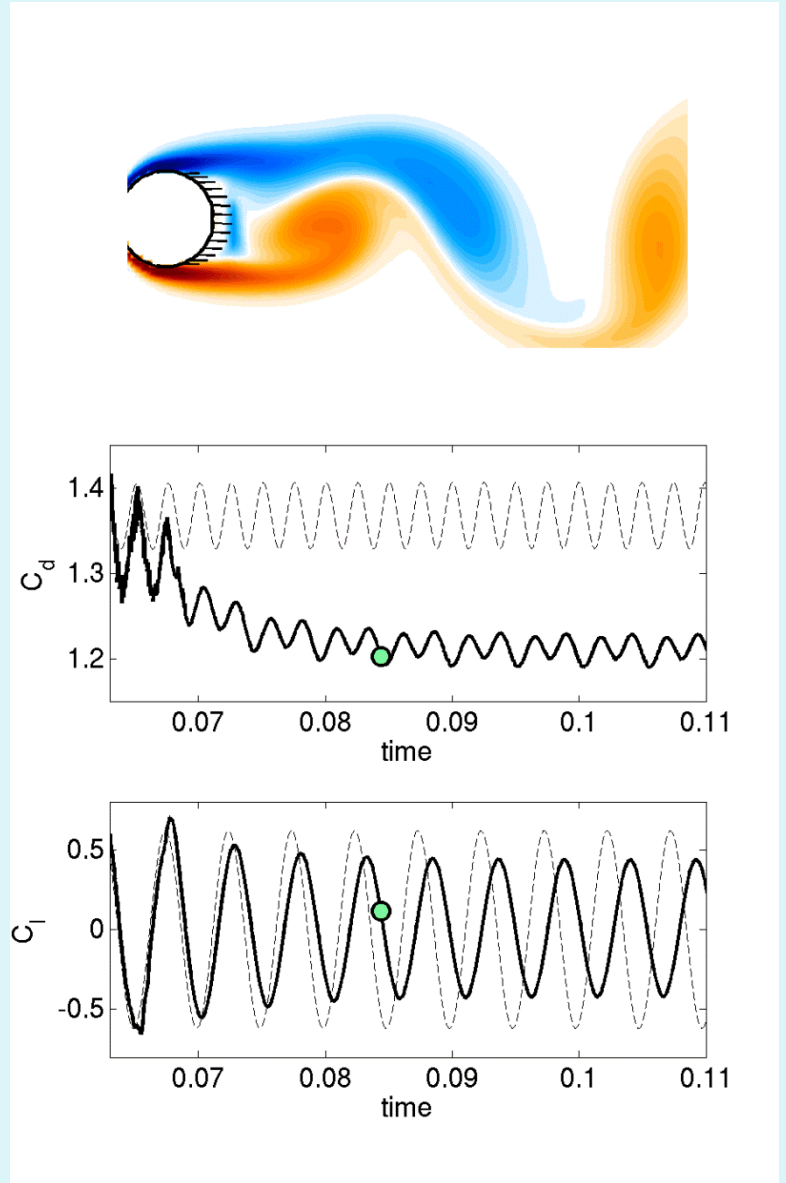
Re = 200

FSI

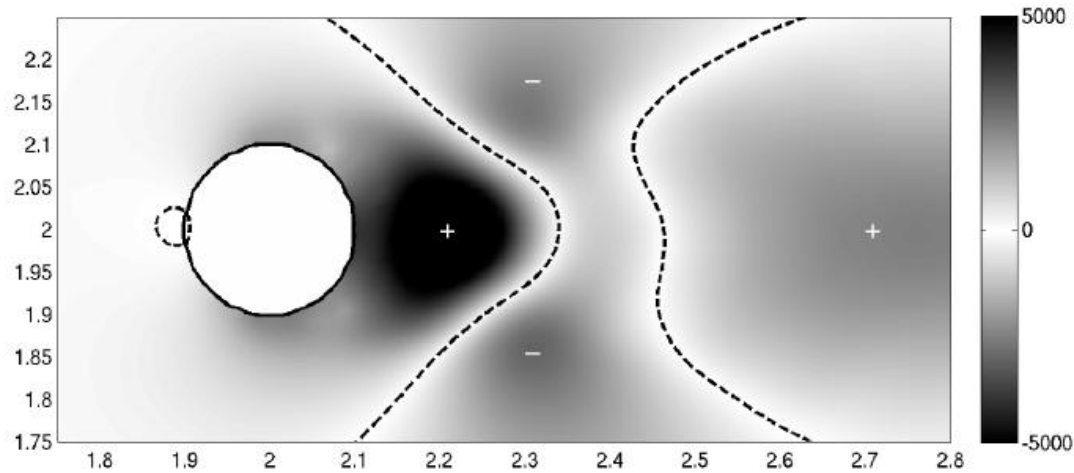


C_d

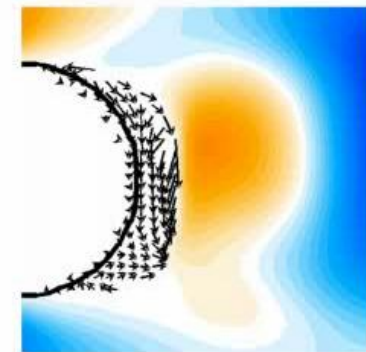
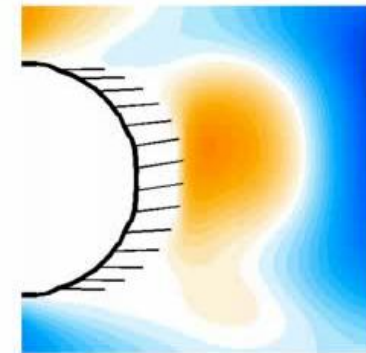
amplitude of C_1 oscillations



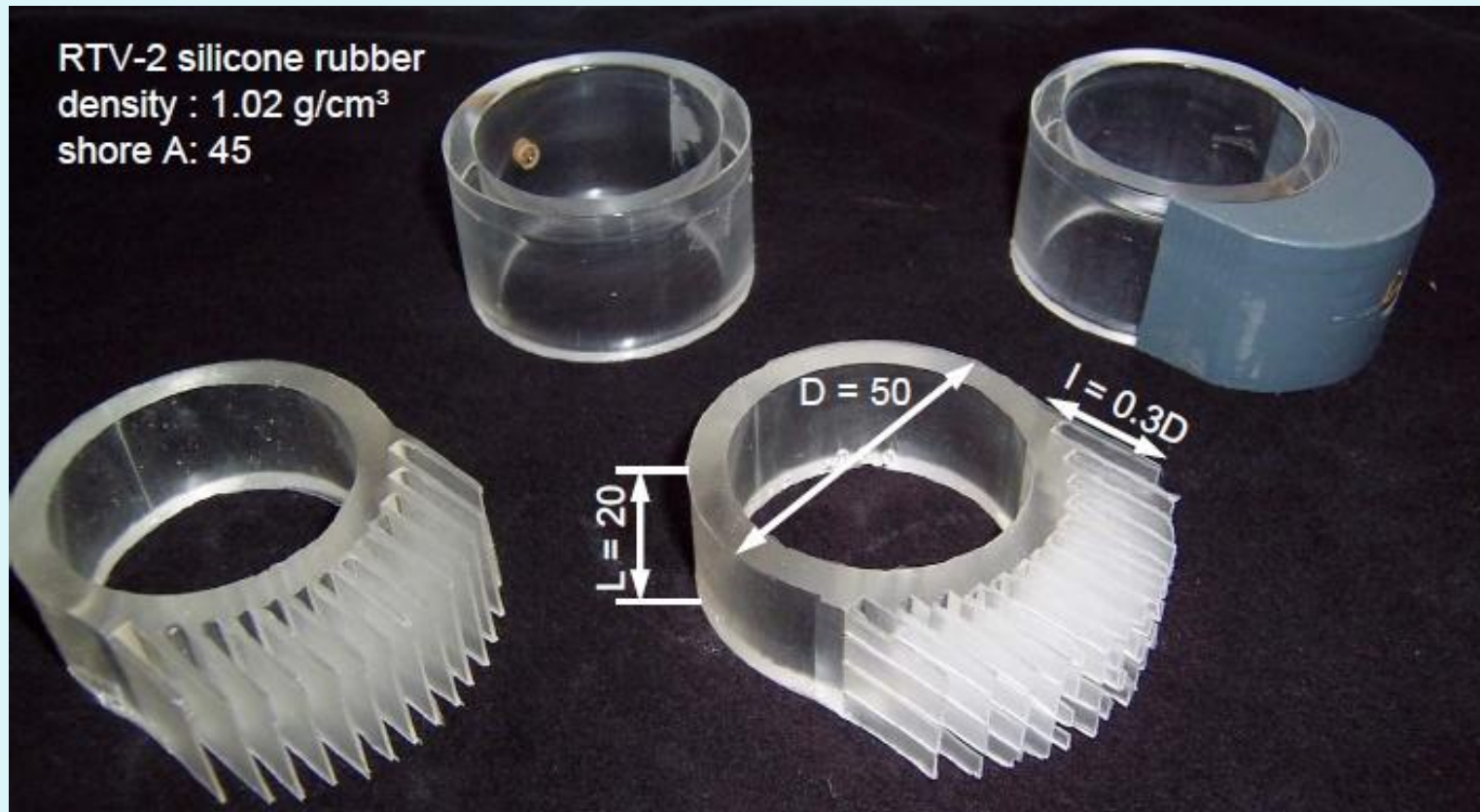
What happens?



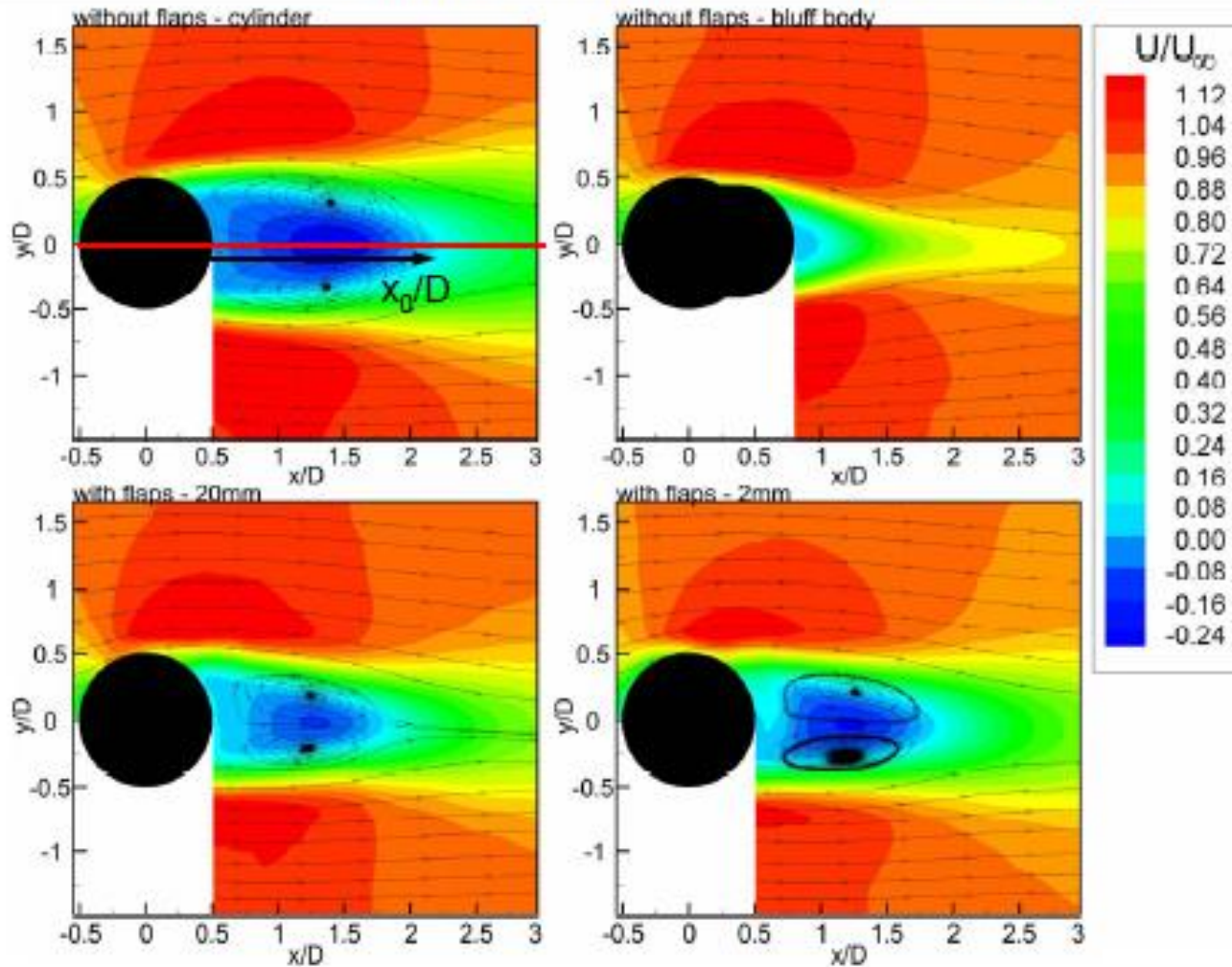
Difference of time-averaged pressure field
 $\langle P \text{ with hair} \rangle - \langle P \text{ ref} \rangle$



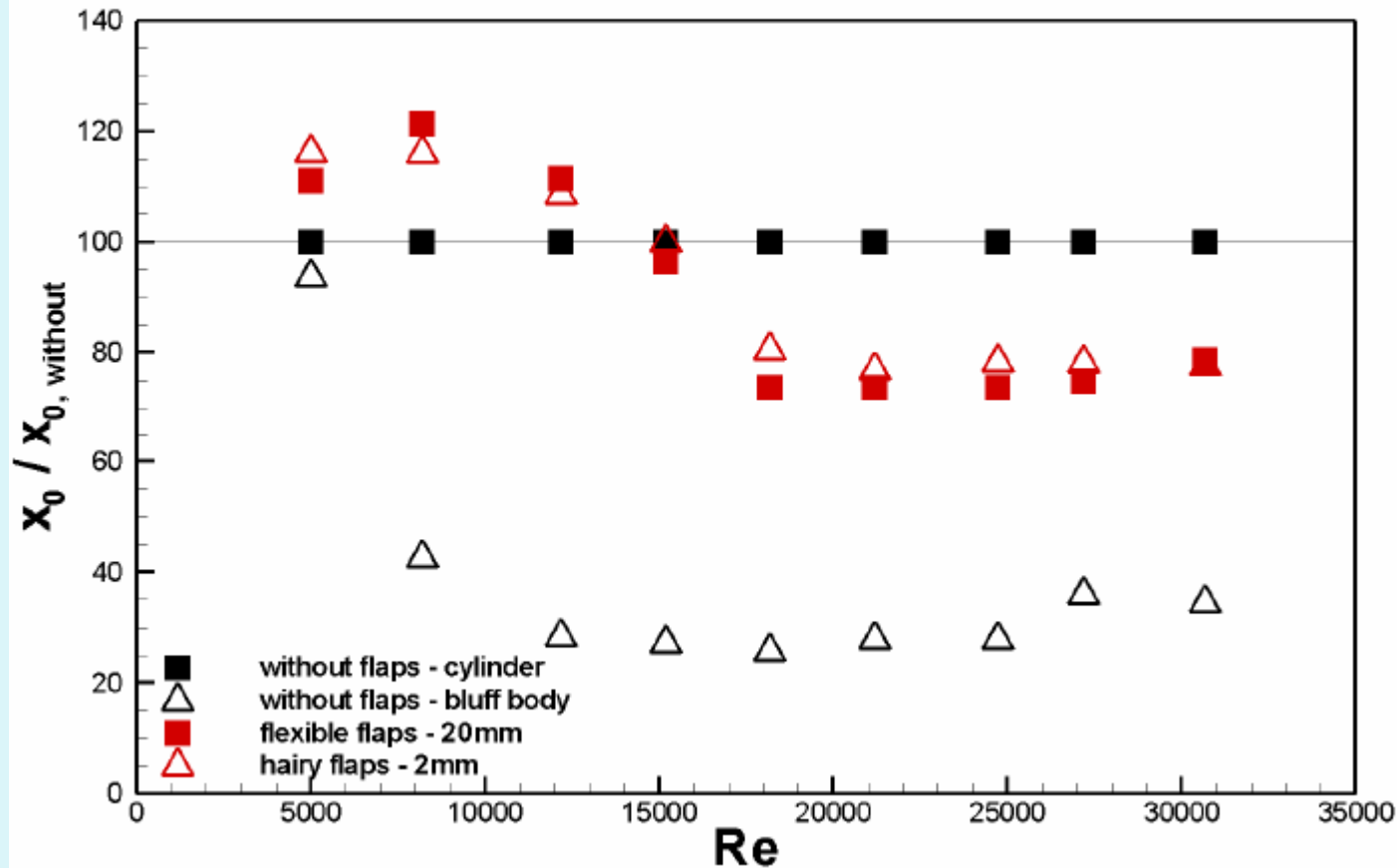
For larger Re numbers, there are the experiments in water and oil channels by Prof. Ch. Brücker



Cylinder Experiments – Results $Re = 27200$

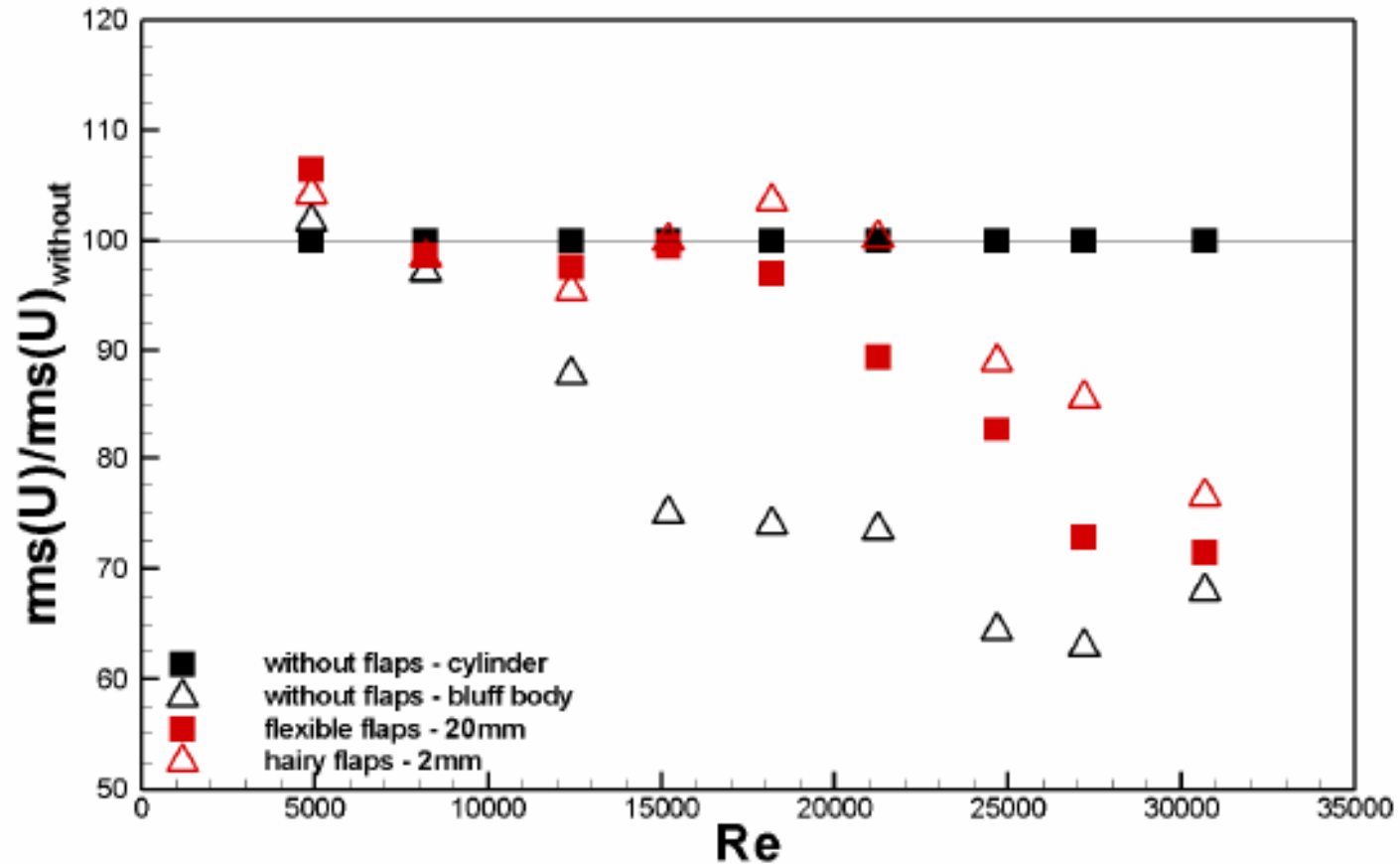


Cylinder Experiments – Results



Flexible and hairy flaps produce a comparable result: recirculation area elongated for $Re < 15000$, shortened for $Re > 15000$

Cylinder Experiments – Results

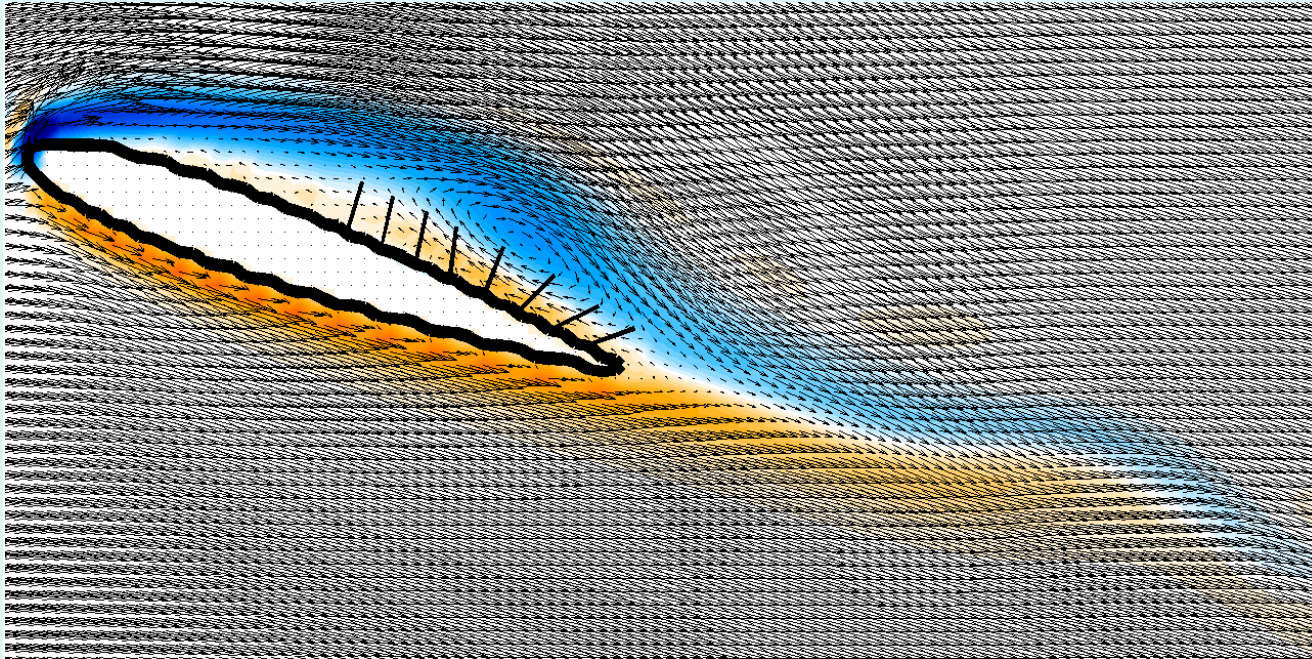


... plus reduced flow fluctuations for $Re > 15000$

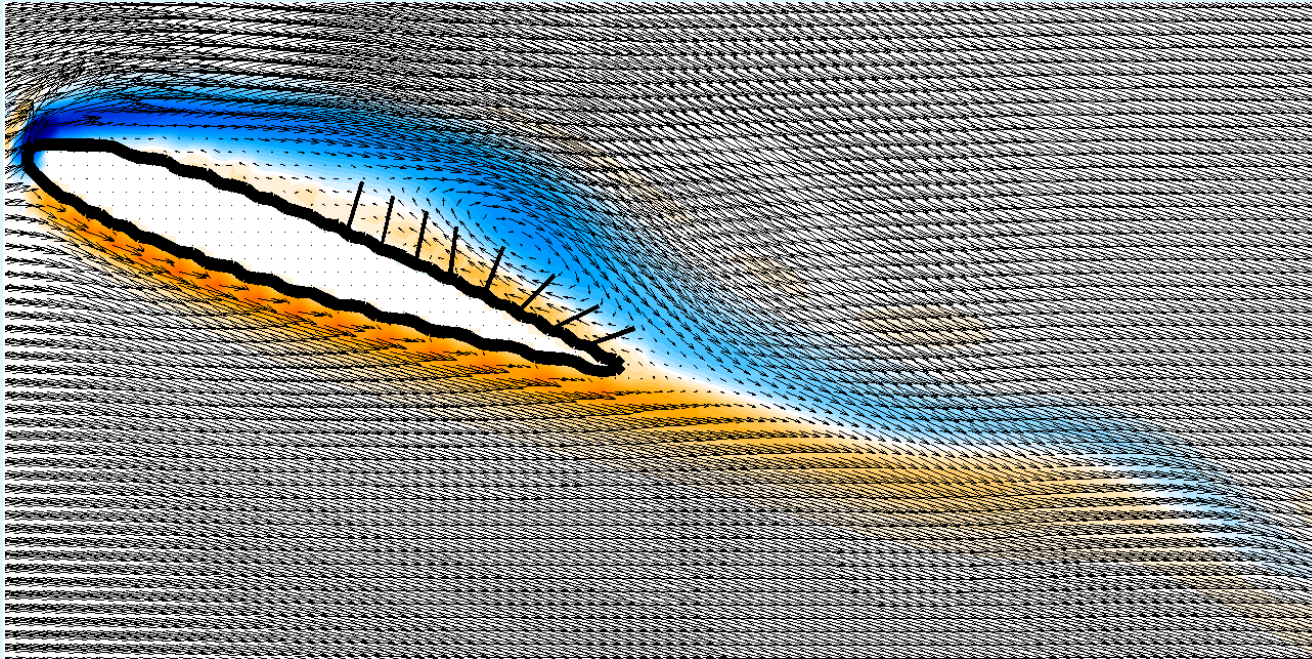


Clear effect on the cylinder wake
→ effect on drag

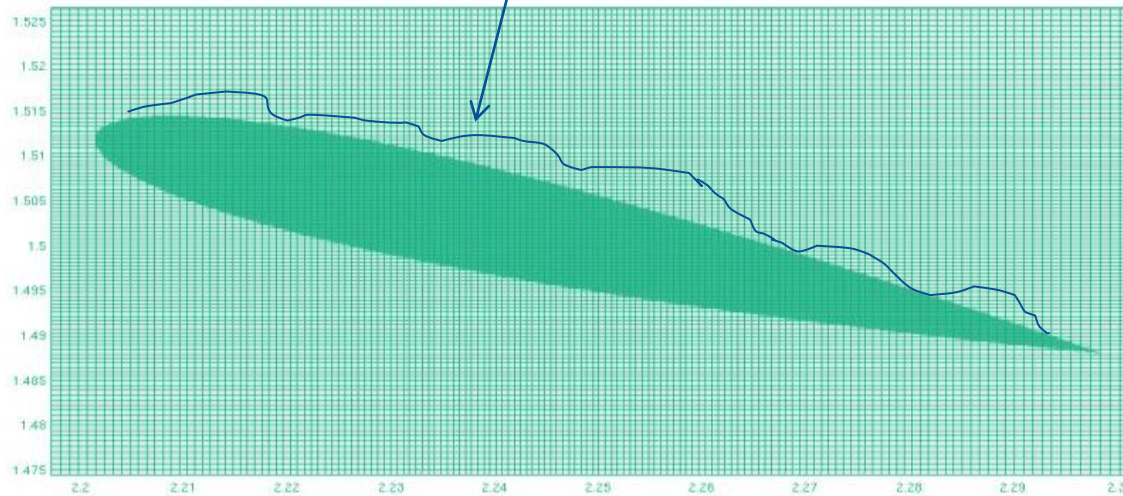
New goal: effect on lift and drag over an airfoil via a *passive, poroelastic* layer



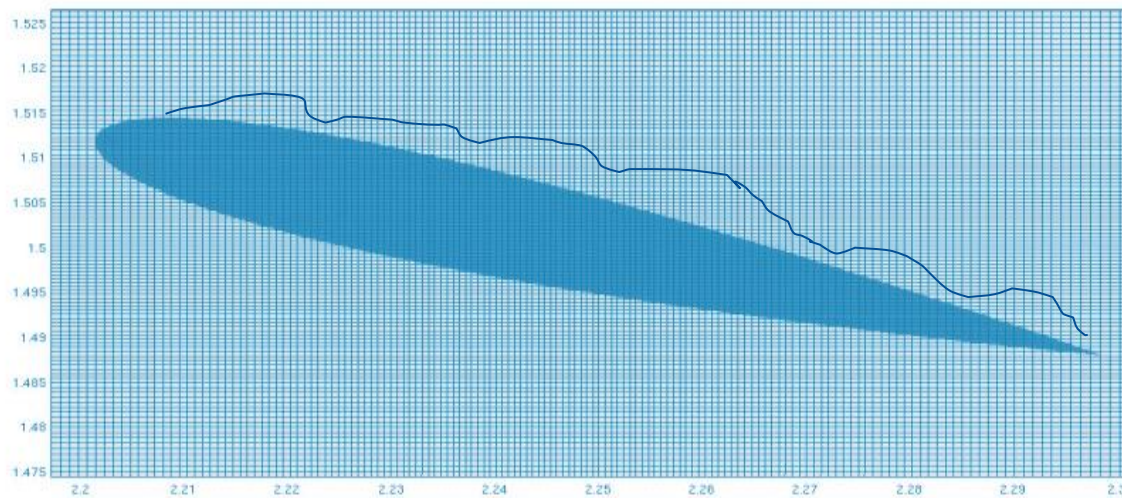
HAIRFOIL !



Goal: put a *poroelastic layer* on the suction side of an airfoil to affect the separated region and the wake



How can we model the layer (made of fibers, hence porous and compliant)?

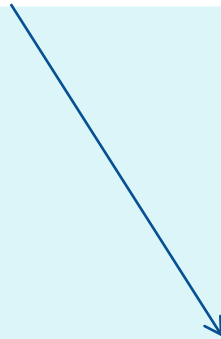


Options:

| | | | |
|--------------------|----------------------------------|-------------------------------|----------------------------|
| Experiments | Poroelasticity theory | NS IBM simulations | Low order model |
|--------------------|----------------------------------|-------------------------------|----------------------------|

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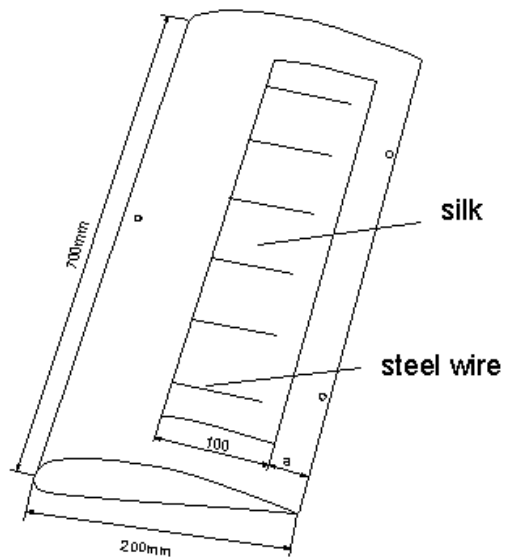
- Berlin, Prof. I. Rechenberg
- Freiberg, Prof. Ch. Brücker
- Orléans, Prof. A. Kourta
- Genova



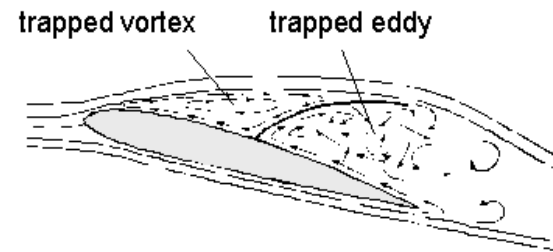
Experiments (Berlin)

Experiments (Berlin)

aerofoil with silk flaps



flow visualisation



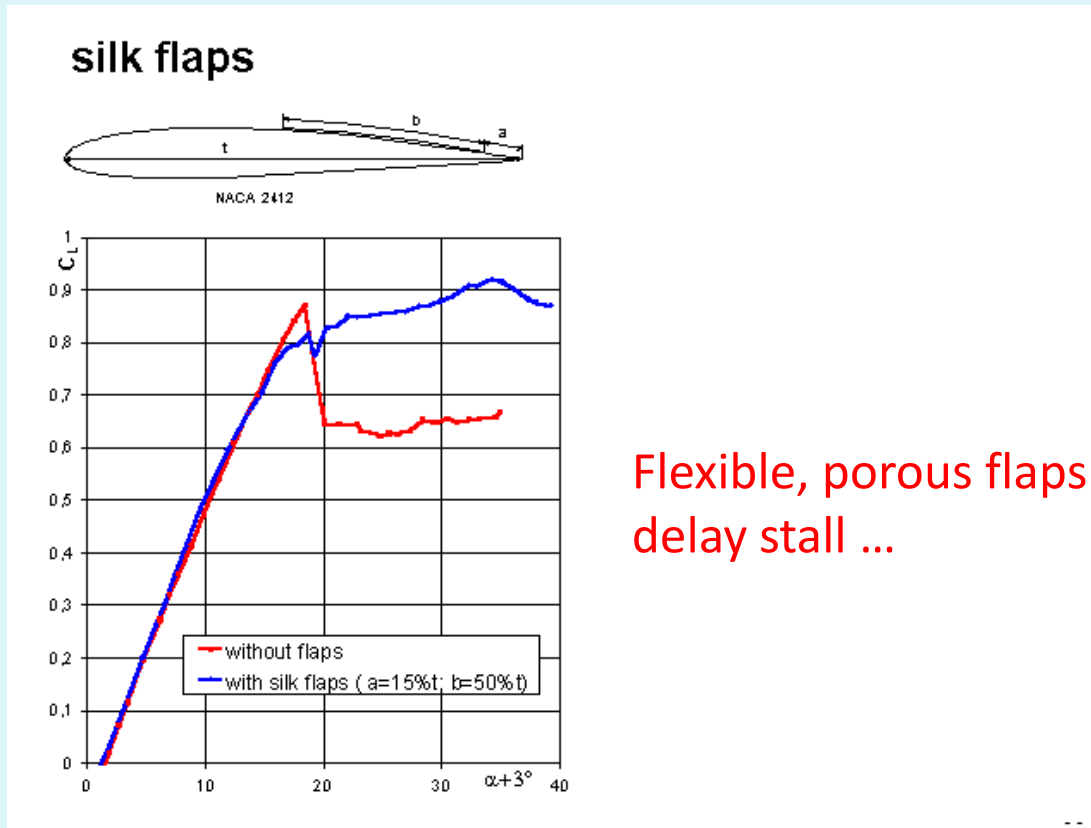
Prof. Ingo Rechenberg, TU Berlin

<http://www.bionik.tu-berlin.de/institut/xs2vogel.html>

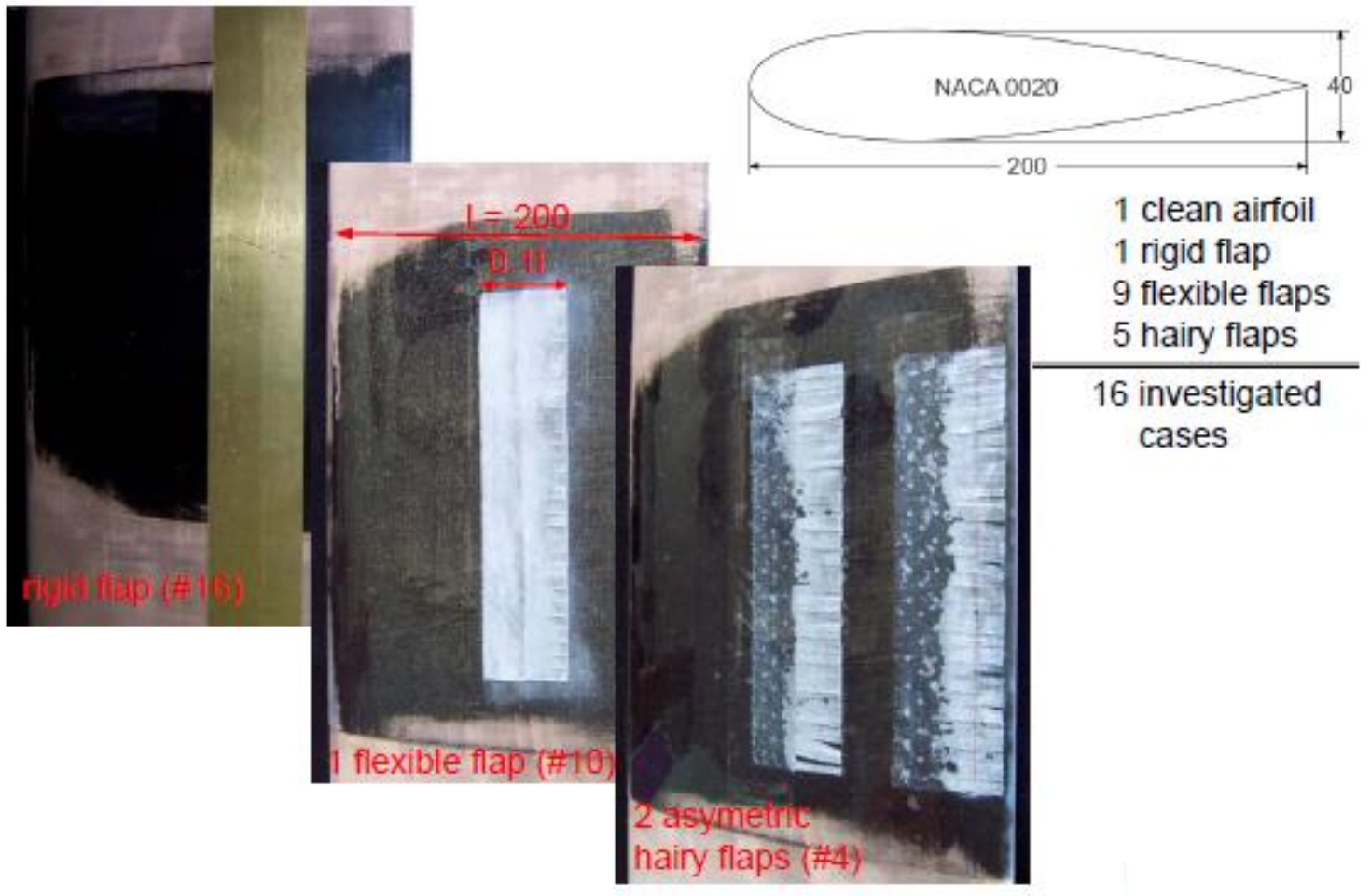


Experiments (Berlin)

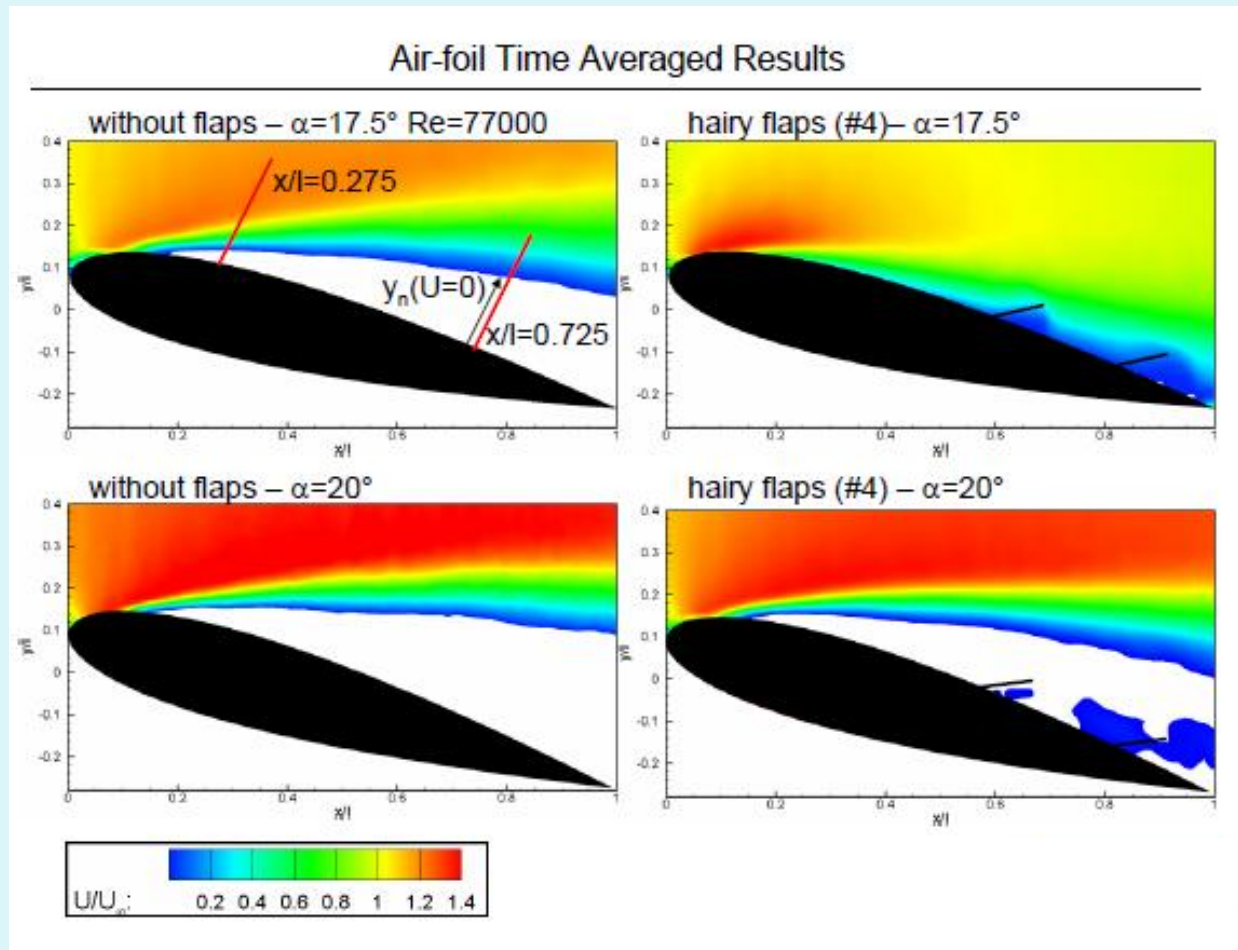
Experiments (Berlin)



Experiments (Freiberg)

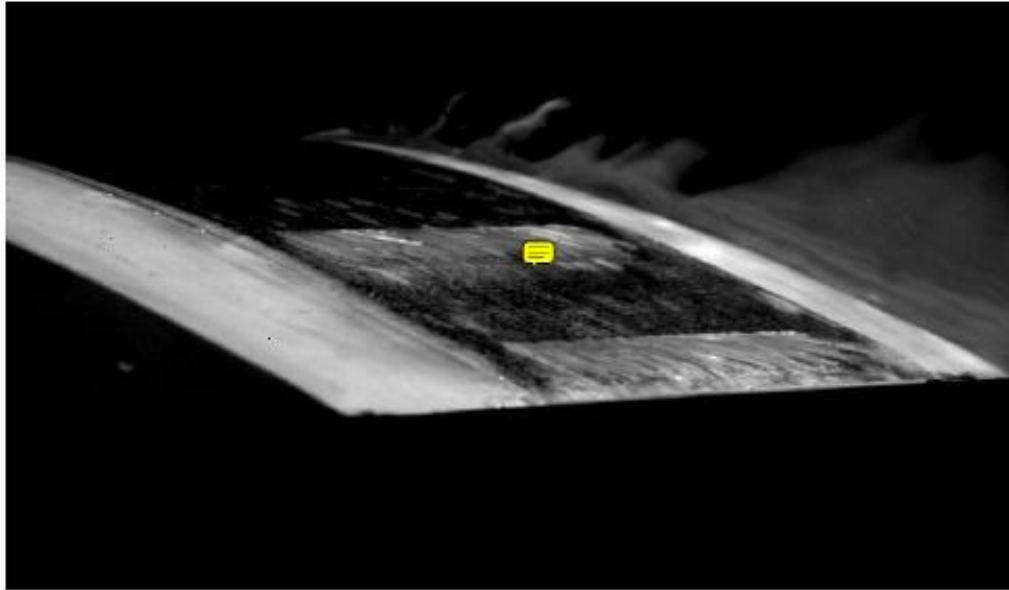


Experiments (Freiberg)



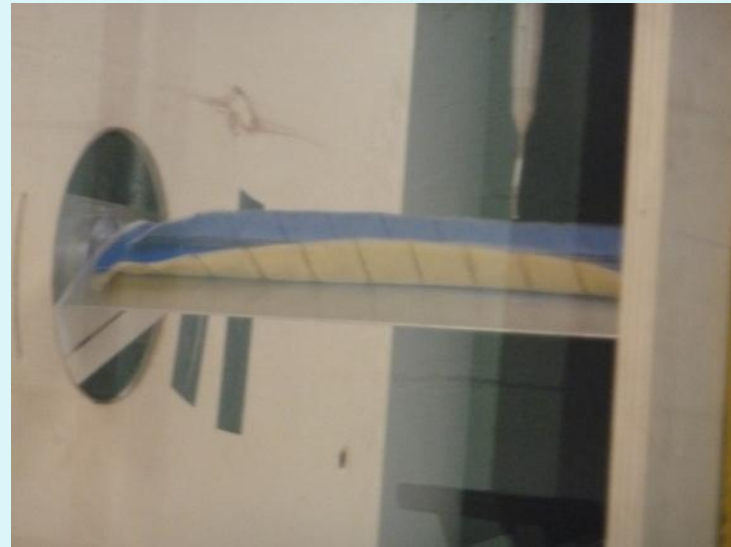
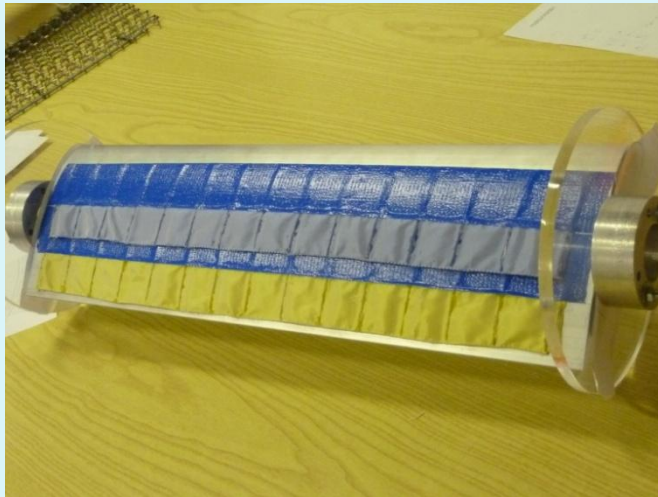
Experiments (Freiberg)

- increasing α 0° - 20° at $d\alpha/dt = 5.6^\circ/s$ - time instant for $U < 0$
at $x = 0.275l$

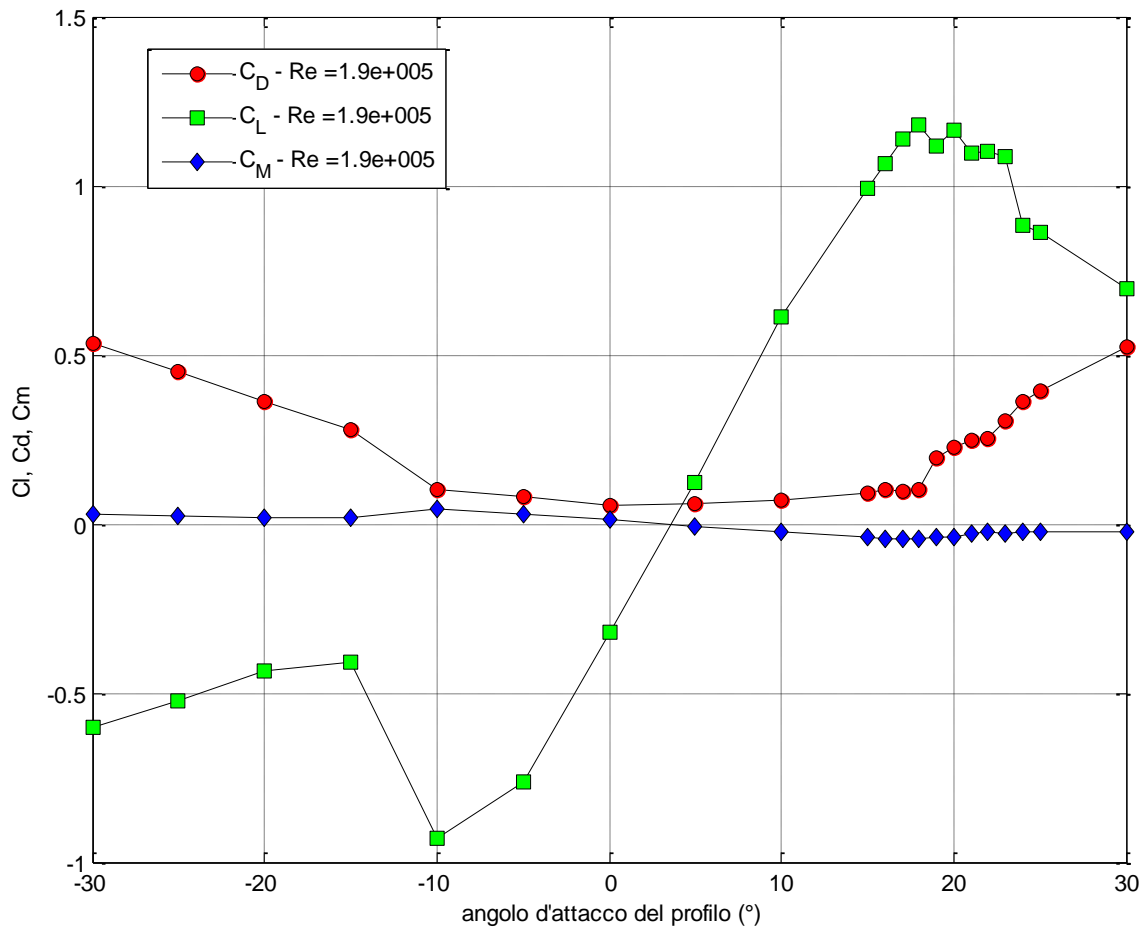


- Flexible and hairy flaps oppose separation at fixed angles of attack
- Hairy flaps delay flow separation in pitching experiments

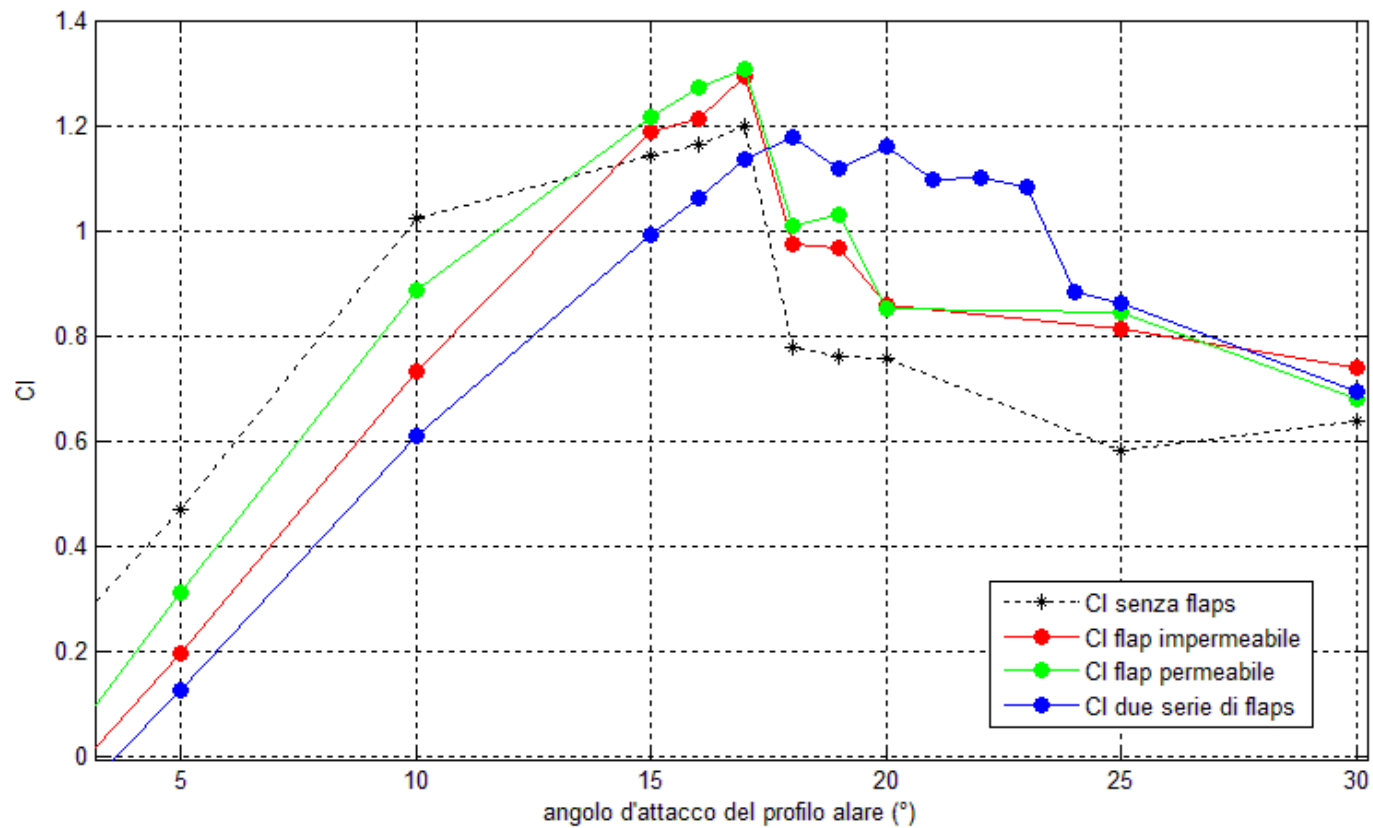
Experiments (Genova)



Experiments (Genova)



Experiments (Genova)



Options:

| | | | |
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| Experiments | Poroelasticity theory | NS IBM simulations | Low order model |
|-------------|-----------------------|--------------------|-----------------|



Gopinath & Mahadevan (*Proc. Royal Soc. A*, 2010)

Options:

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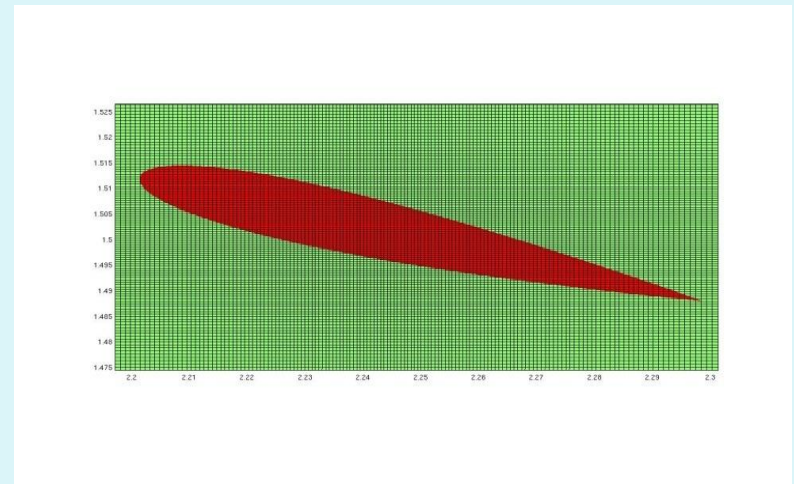
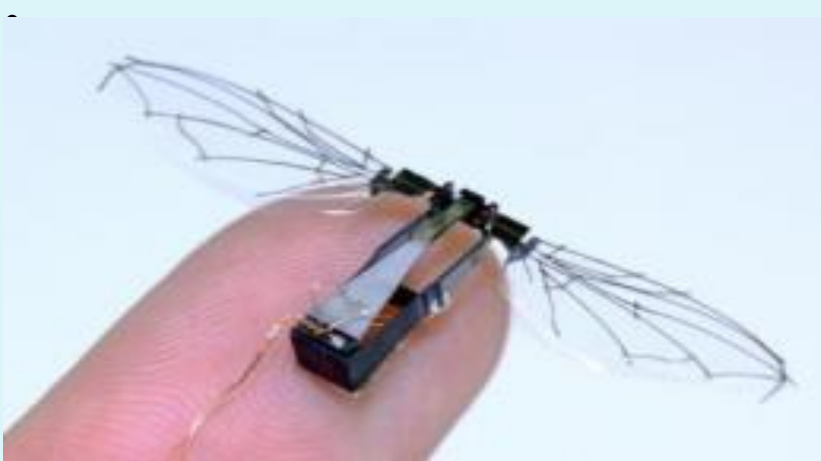


Present work (at low Re number)

- NACA0012 airfoil
- More difficult to control separation of boundary layer in laminar case – boundary layer less capable of handling adverse pressure gradient without separation.
- Low Reynolds number (1100) particularly used for testing performance of MAVs.

- Incompressible, unsteady 2D N-S eqns. with forcing
- Immersed Boundary Method: Stationary, non-conformal Cartesian grid (fine on and near airfoil)
- Feedback forcing term in N-S: Spring-mass system

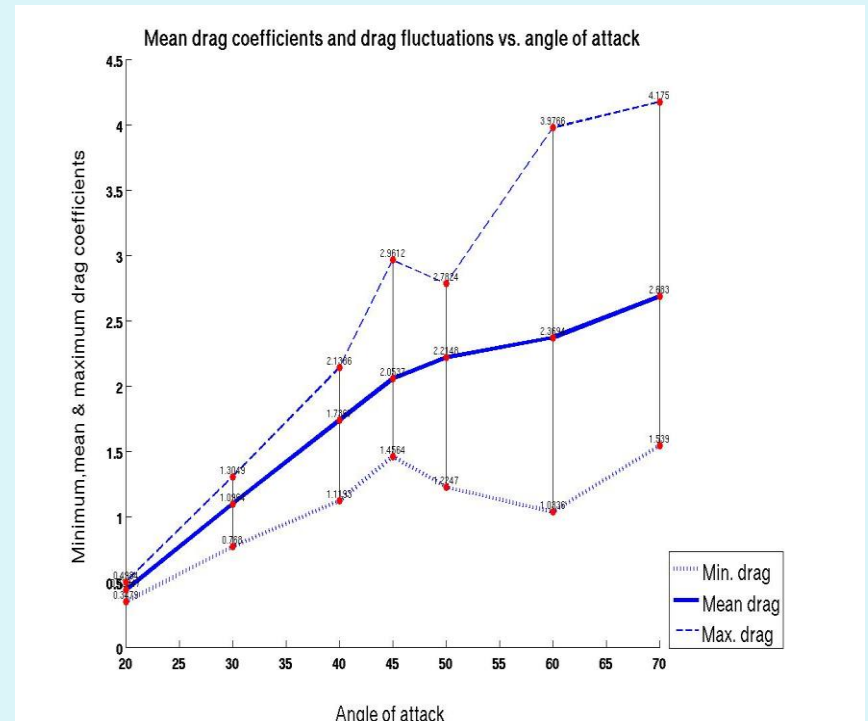
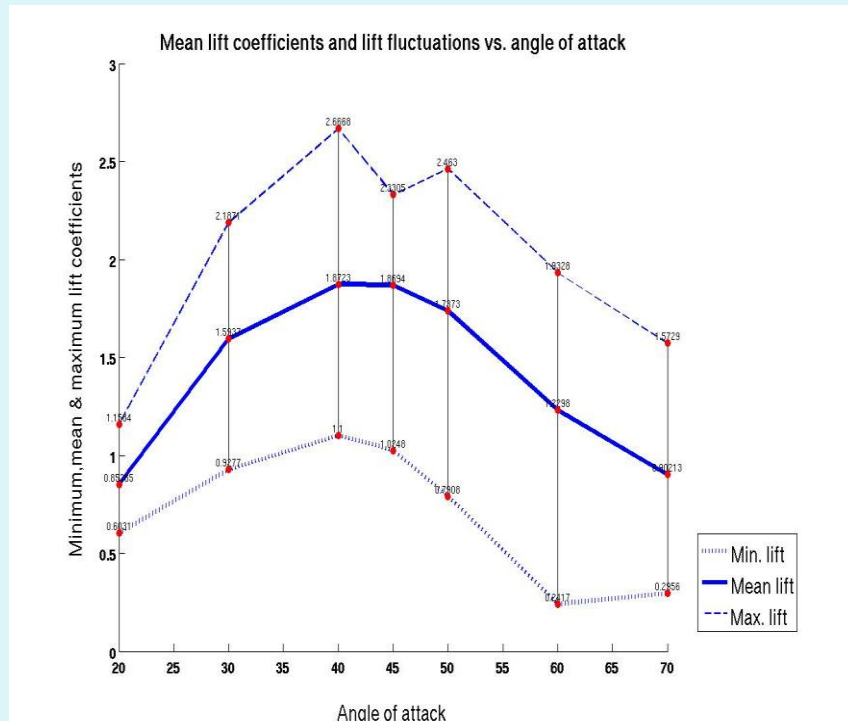
$$F = \alpha \int (\mathbf{0} - \mathbf{U}) dt + \beta (\mathbf{0} - \mathbf{U})$$



RESULTS: NO-CONTROL CASE

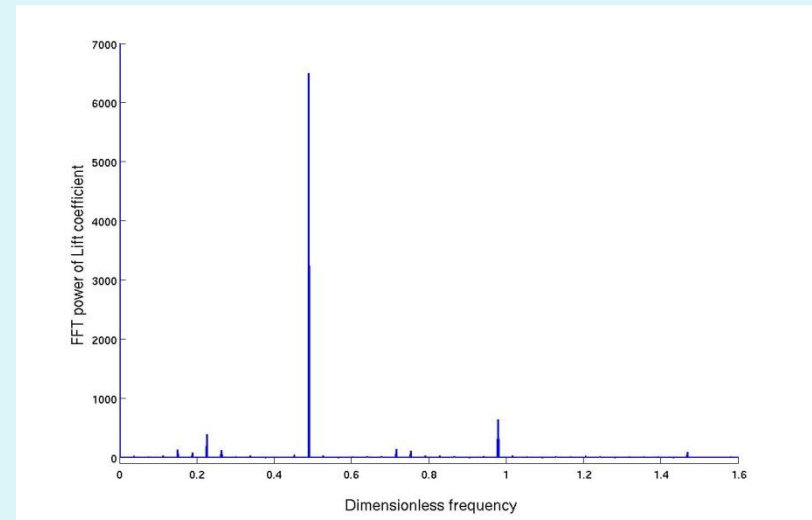
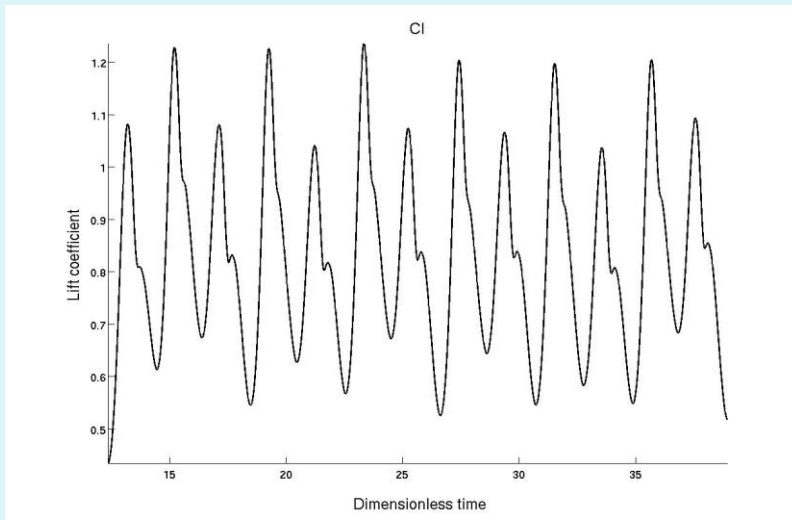
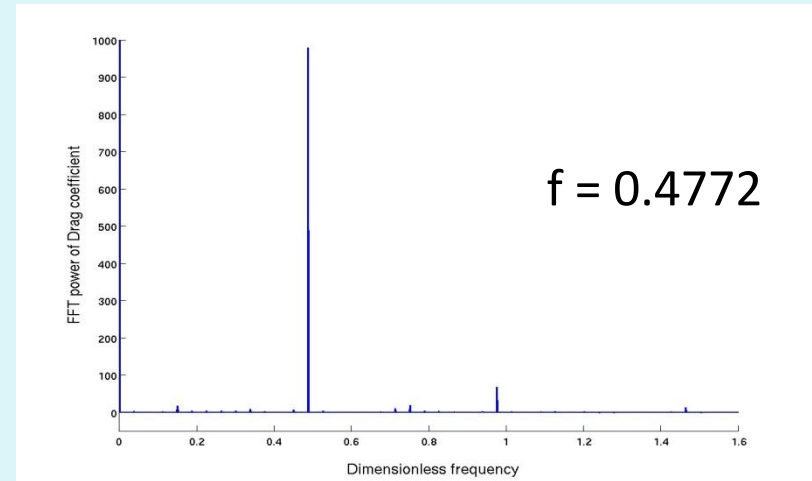
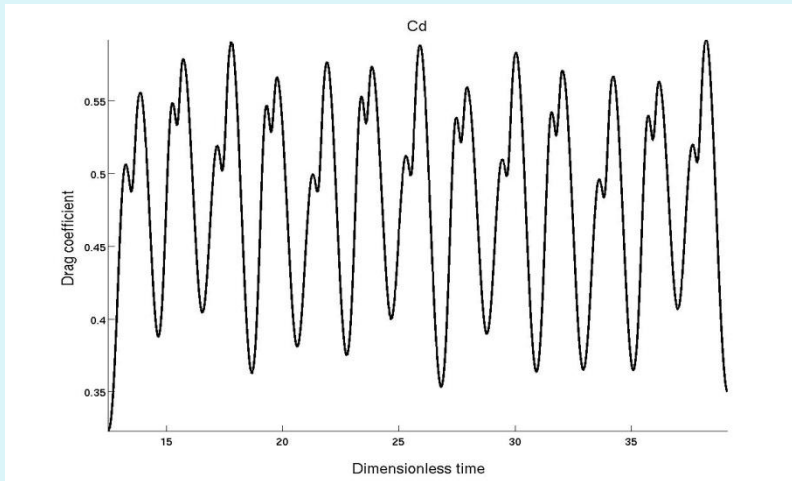
Mean lift as a function of angle of attack (between 20° and 70°)
- steady drop in lift after 45°.

Mean drag as a function of angle of attack (between 20° and 70°).

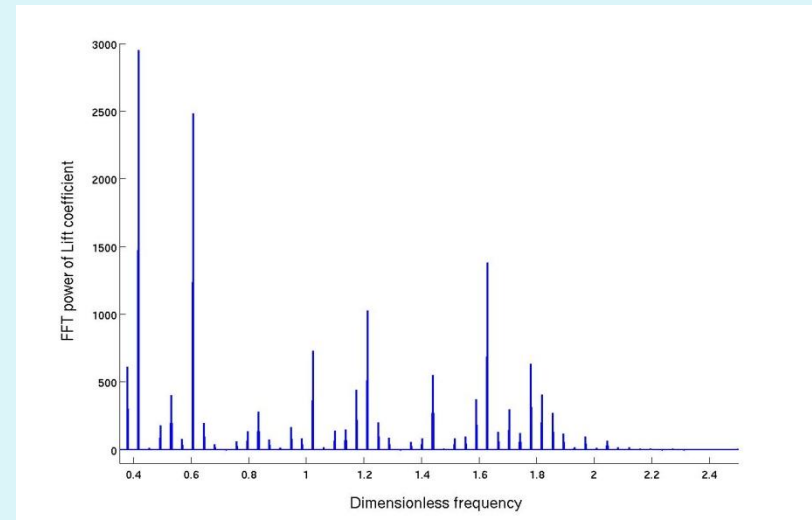
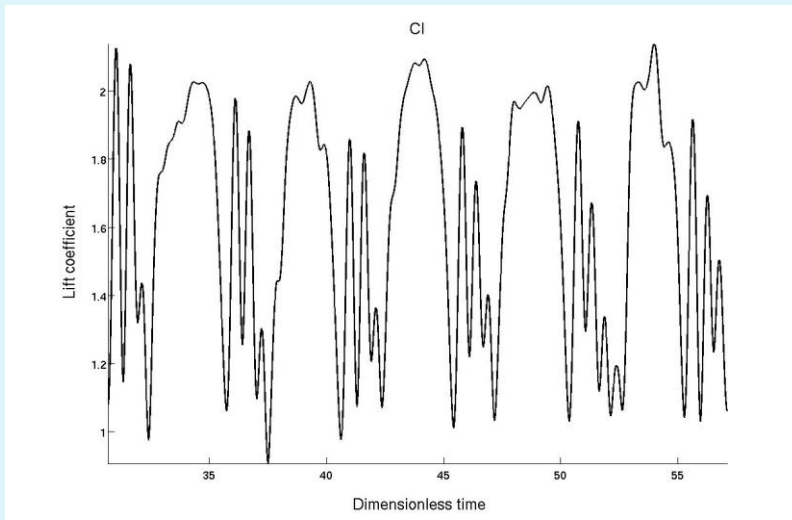
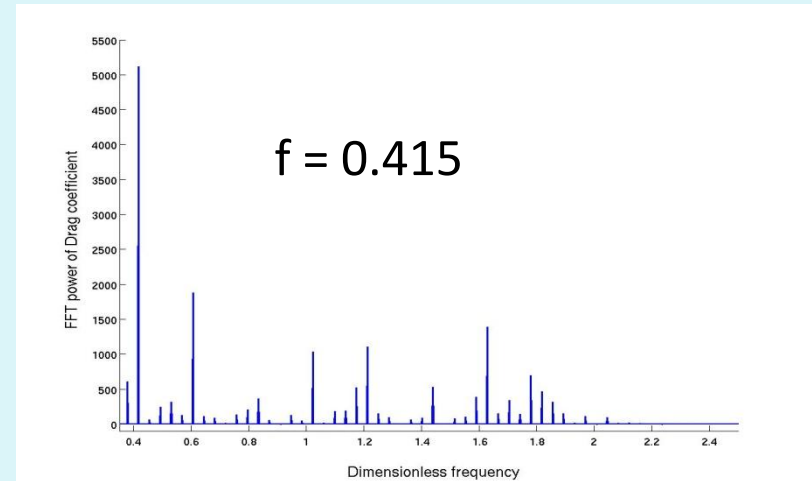
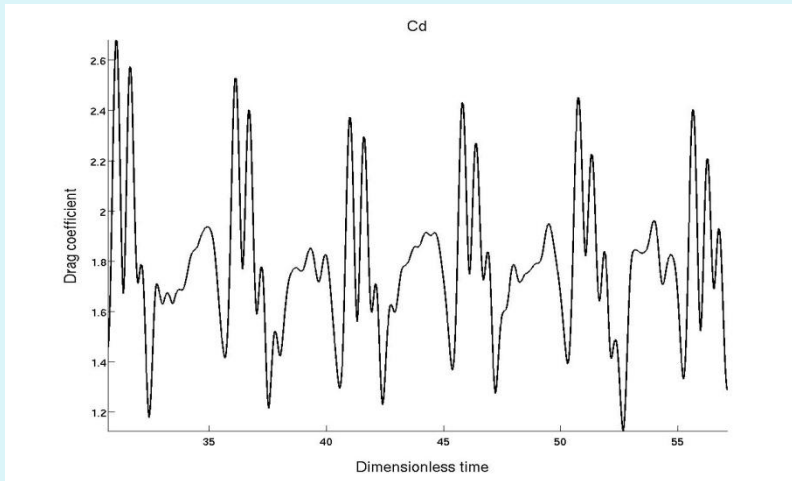


An effective control should see a correlation between the time scale of the vortex shedding frequency and the natural time scale of the structures

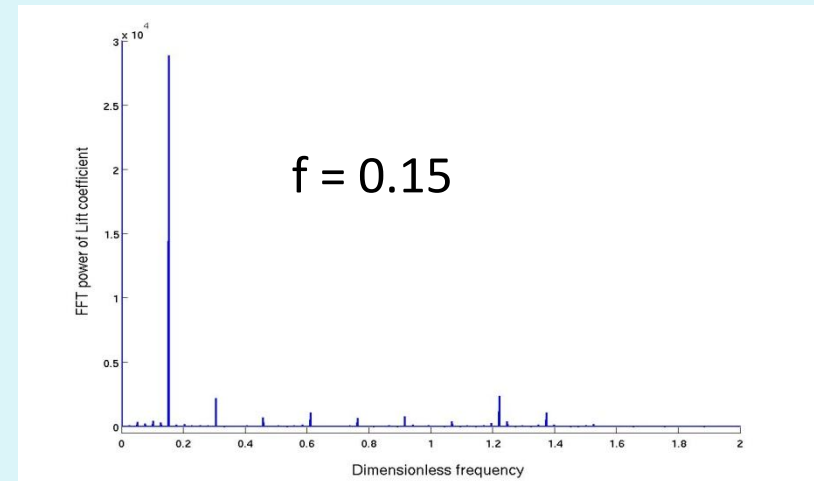
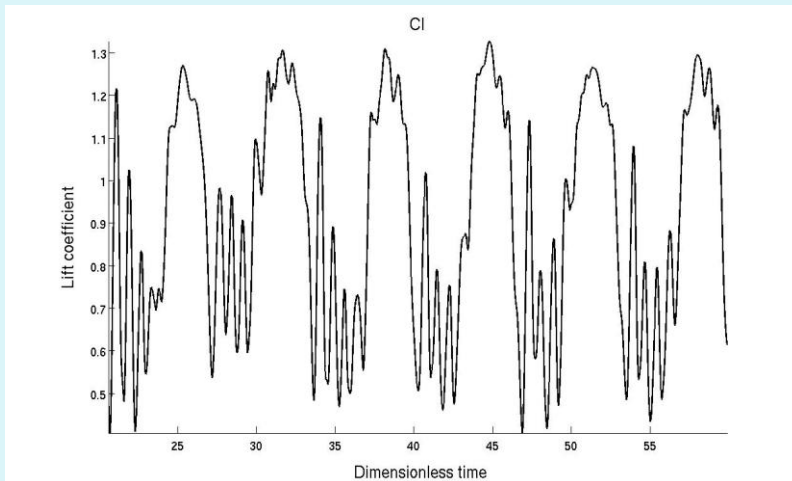
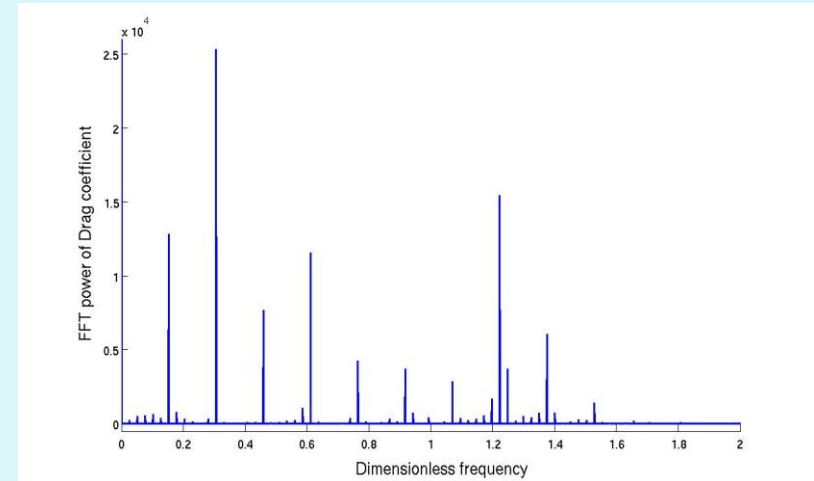
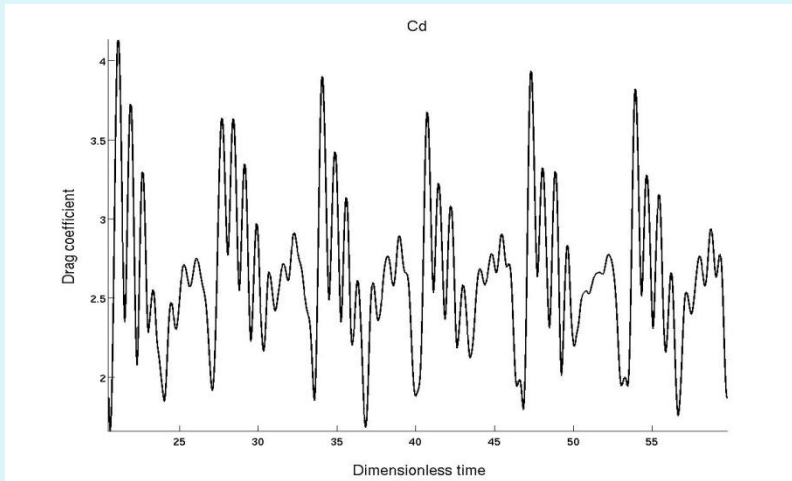
POWER SPECTRA OF DRAG & LIFT SIGNALS : 22 DEGREES



POWER SPECTRA OF DRAG & LIFT SIGNALS : 45 DEGREES



POWER SPECTRA OF DRAG & LIFT SIGNALS : 70 DEGREES



Modeling in 3 points

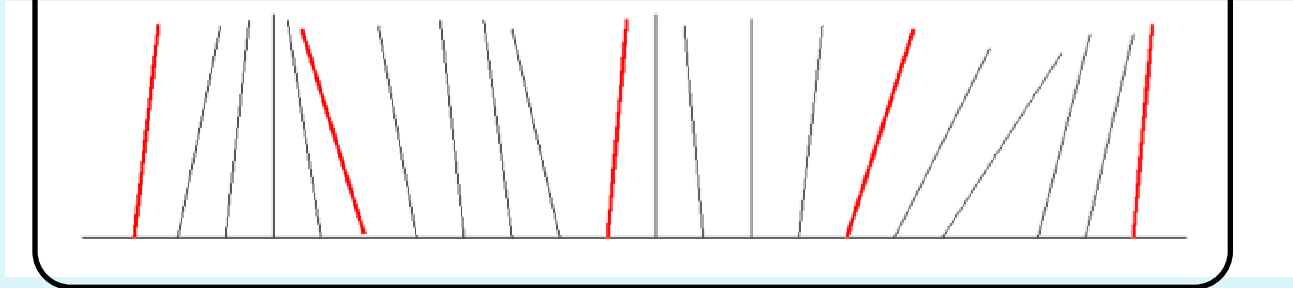
Modeling all feathers: too heavy ...



Must reduce the number of degrees of freedom

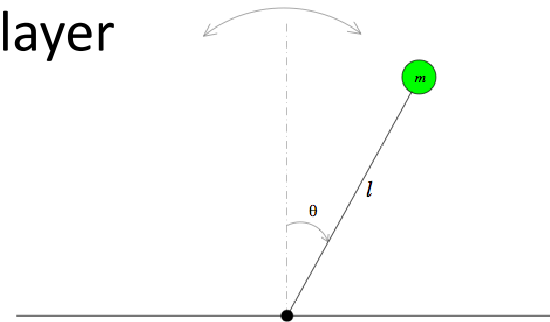
1. **Homogenized approach:** description of the layer in terms of **density** and **direction** of feathers.
2. Motion of the layer reduces to the oscillation of a small number of **reference elements**
3. The fluid “sees” the structures in terms of **volume forces** (and the same for the structures)

Homogenized approach



Dynamics of the layer

Approximation :
Rigid **reference** element



E. De Langre, ARFM, 2008



Fluid part...

$$\frac{\partial \mathbf{U}}{\partial t} + \nabla(\mathbf{U}\mathbf{U}) = -\frac{\nabla p}{\rho} + \nu \nabla^2 \mathbf{U} + \mathbf{F}; \nabla \cdot \mathbf{U} = 0$$

2D incompressible

Volume forces formulation

Staggered grid

Periodic boundary conditions,
with buffer domain to treat I/O

Convective part: Adams-Bashfort

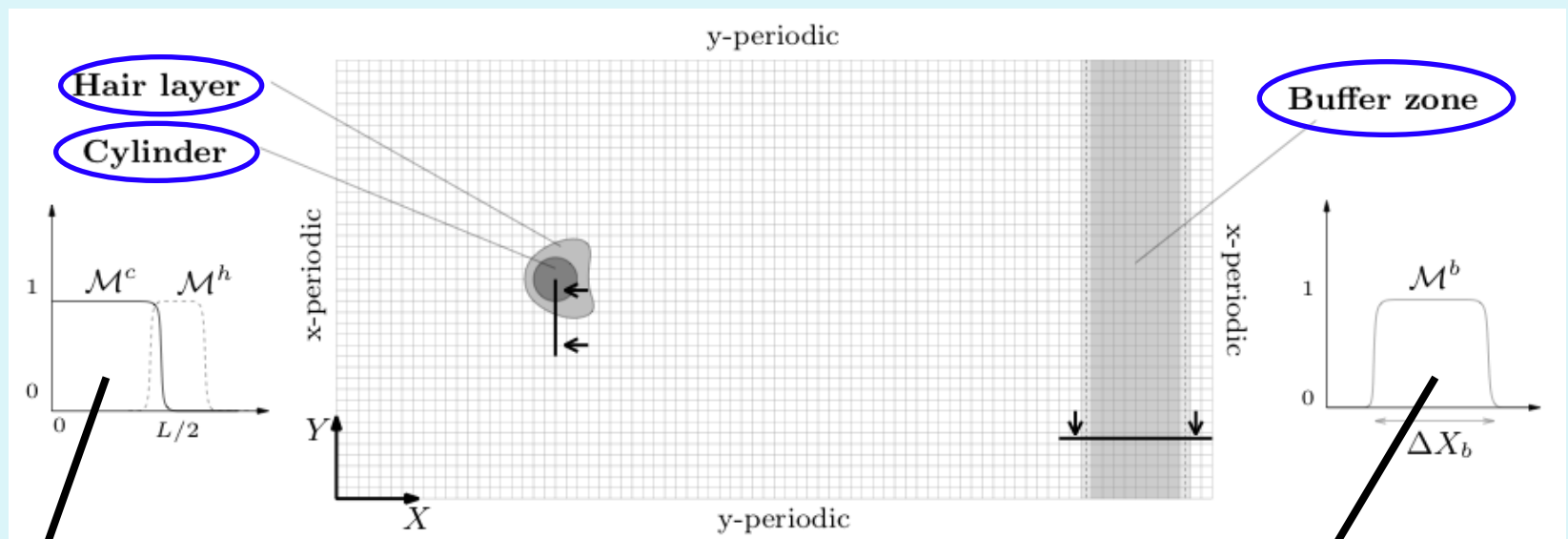
Viscous part: Crank-Nicolson

Poisson and implicit parts solved
using conjugate gradient

→ order 2 in space and time

$$\frac{\partial \mathbf{U}}{\partial t} + \nabla(\mathbf{U}\mathbf{U}) = -\frac{\nabla p}{\rho} + \nu \nabla^2 \mathbf{U} + \mathbf{F}; \quad \nabla \cdot \mathbf{U} = 0$$

Regular cartesian mesh 200 x 400 (10L x 20L)



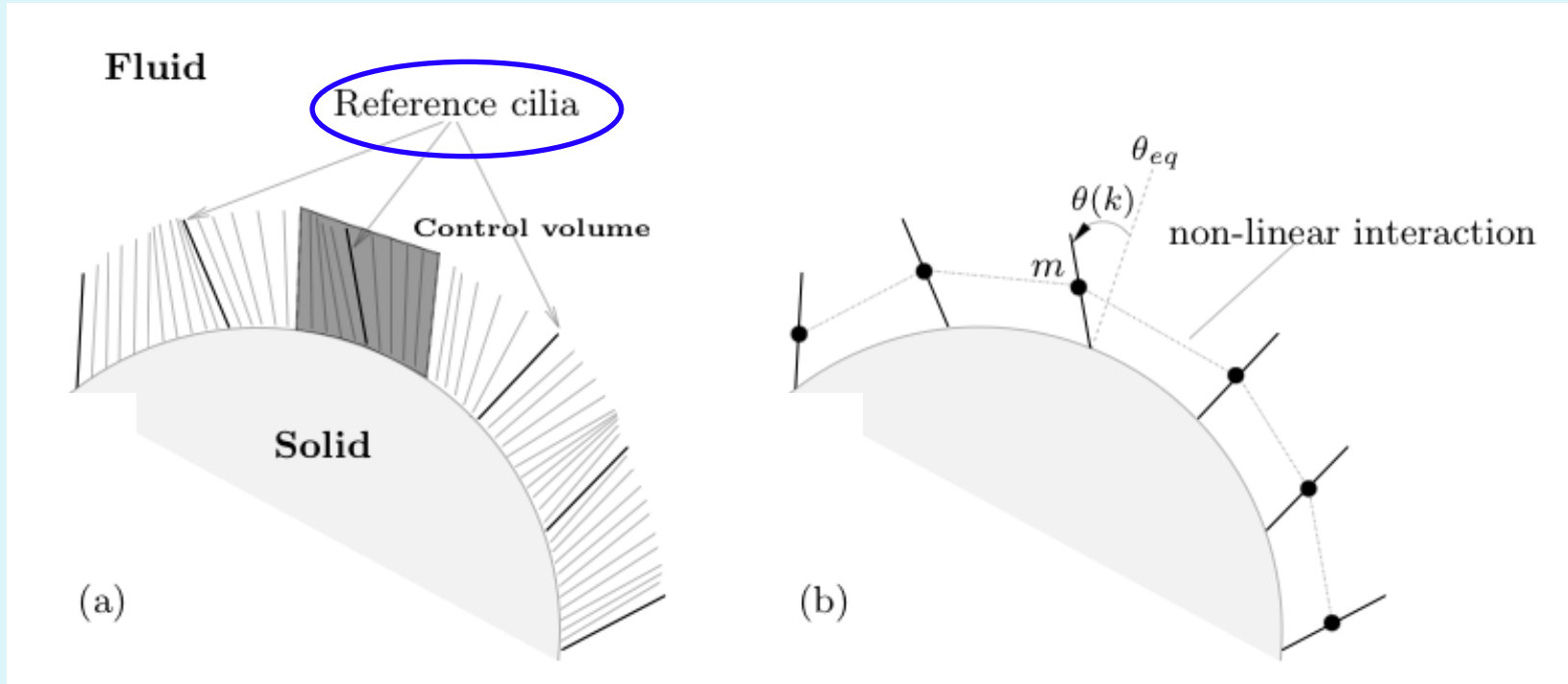
$$\mathbf{F}^c = \mathcal{M}^c \left(\alpha^c \int_{t_0}^t (\mathbf{0} - \mathbf{U}) dt + \beta^c (\mathbf{0} - \mathbf{U}) \right)$$

$$\mathbf{F}^b = \mathcal{M}^b \left(\alpha^b \int_{t_0}^t (\mathbf{U}_\infty - \mathbf{U}) dt + \beta^b (\mathbf{U}_\infty - \mathbf{U}) \right)$$

$$\mathbf{F} = \mathbf{F}^c + \mathbf{F}^b + \mathbf{F}^h$$

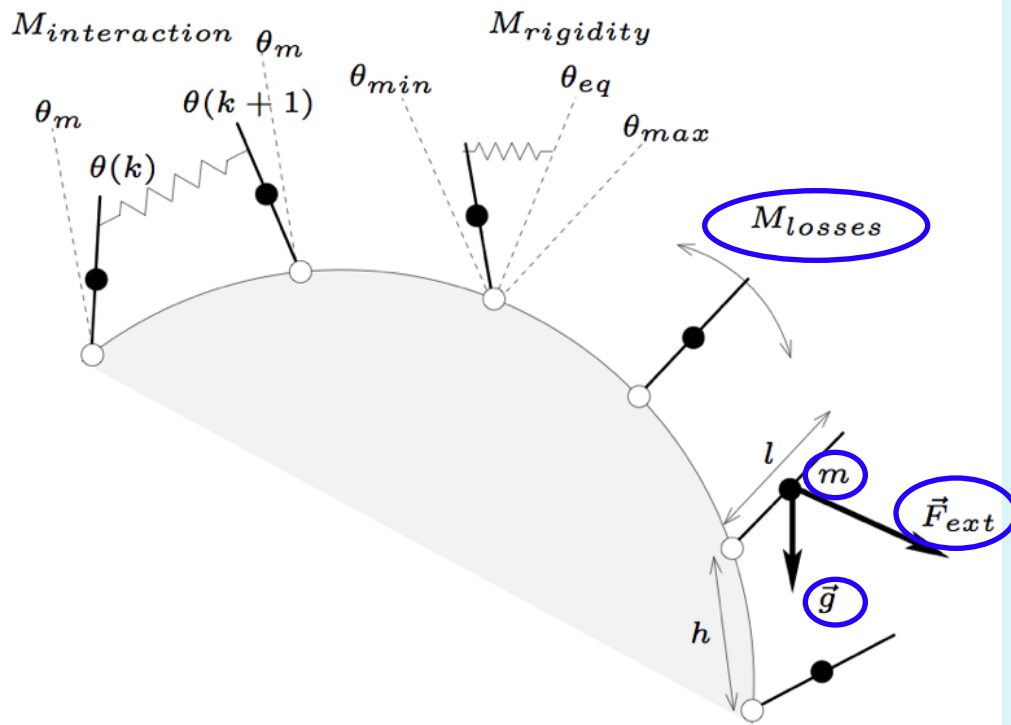
“skeleton” of
the layer

Structure part ...



→ The dynamics of the layer is governed by
the reference elements

→ The “skeleton” of the layer is governed by **six terms** in the angular momentum equation for each element

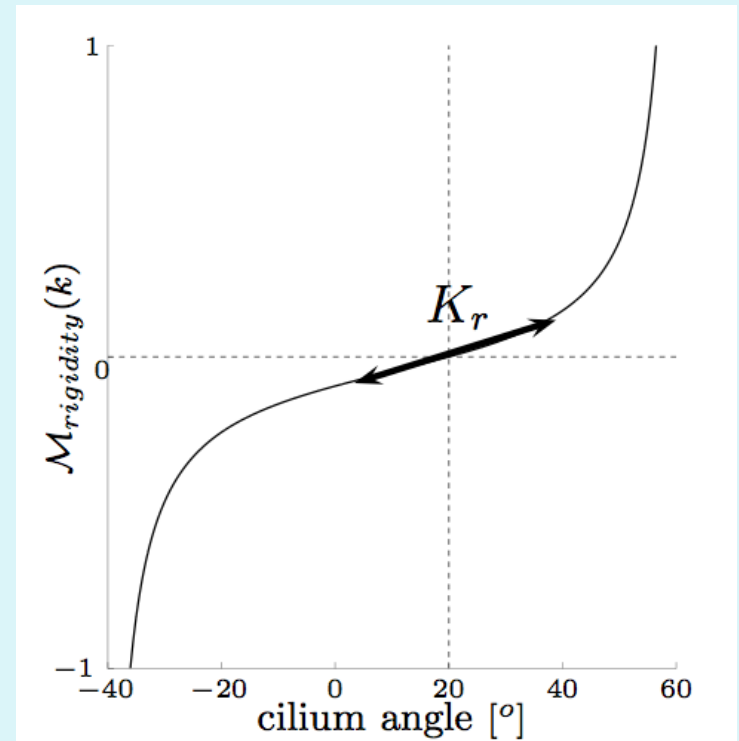
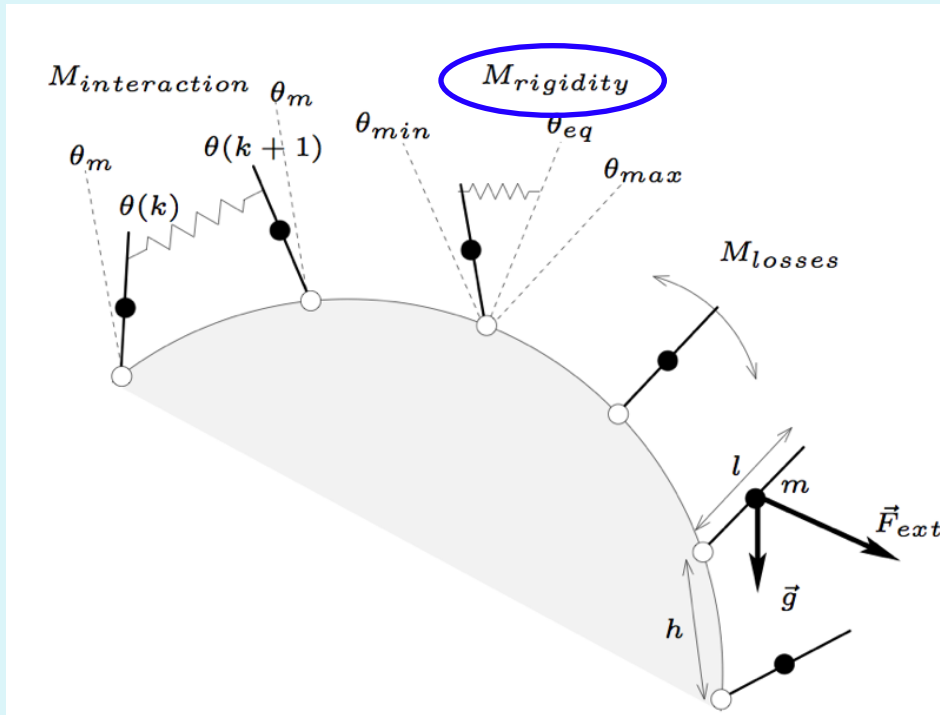


$$M_{losses}(k) = -K_l \dot{\theta}(k)$$

$$M_{inertia}(k) = -m \frac{l^2}{4} \ddot{\theta}(k)$$

$$M_{gravity} = mg \frac{l}{2} \sin(\theta)$$

$$M_{ext}(k) = \frac{l}{2} F_n(k)$$

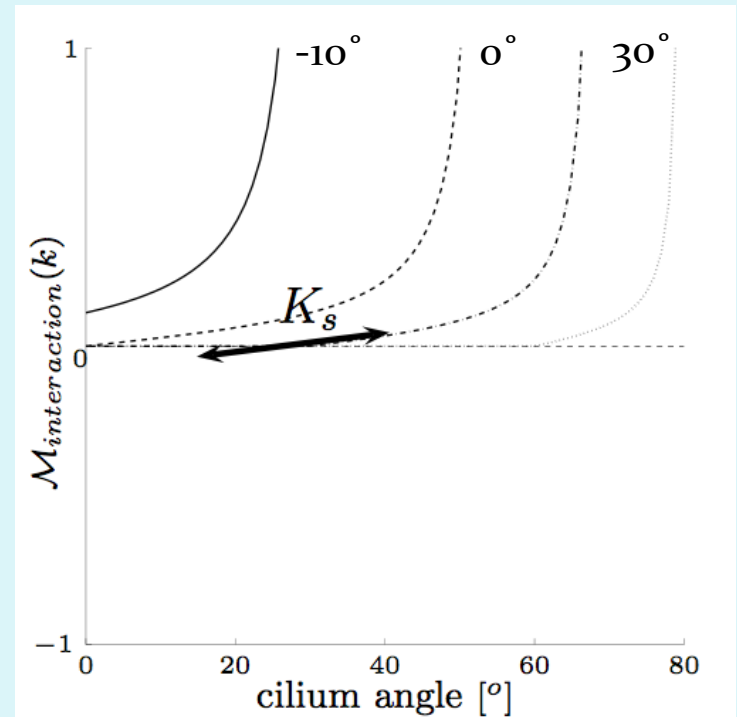
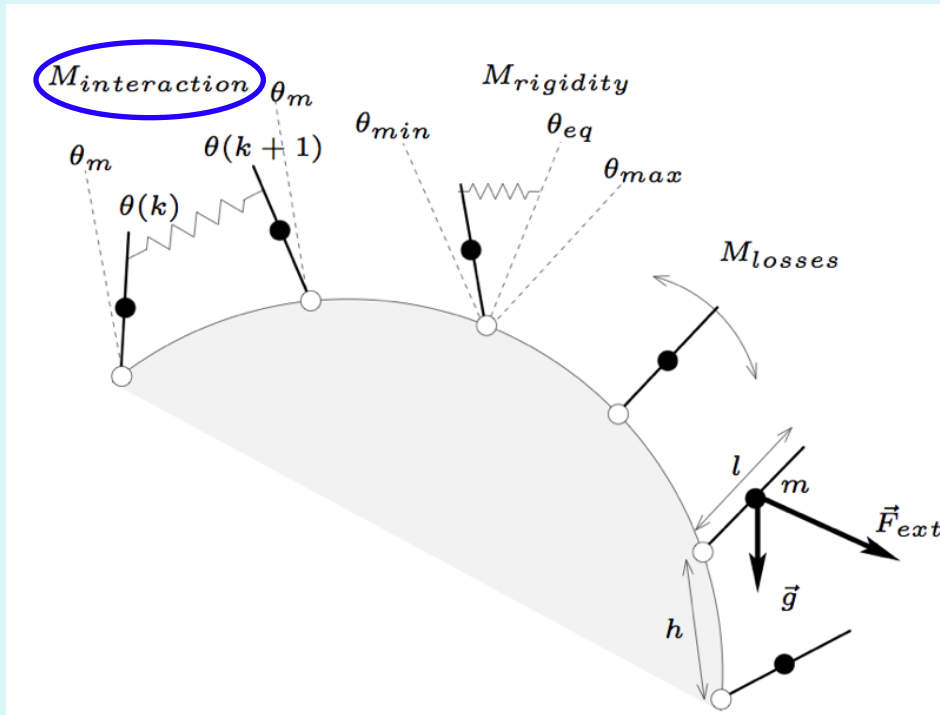


$$M_{rigidity}(k) = -K_r \frac{P(\theta) - P(\theta_{eq})}{P'(\theta_{eq})}$$

avec $P = \tan(a\theta + b)$

$$a = \pi / (\theta_{max} - \theta_{min})$$

$$b = -a(\theta_{max} + \theta_{min}) / 2$$




$$M_{interaction}(k) = K_s \tanh \frac{2l \sin(\theta(k) - \theta_m)}{h \cos(\theta_m)}$$

avec $\theta_m = \frac{\theta(k) + \theta(k+1)}{2}$

Explicit (Runge-Kutta 4) and implicit (nonlinear conjugate gradient) resolution of the angular momentum balance of each reference fiber

$$m(l/2)^2\ddot{\theta}(k) = M_{spring}(k) + M_{rigidity}(k) + M_{dissip}(k) + M_{inertia}(k) + M_{ext}(k) , k = 1, \dots, N_c$$

Equilibrium is reached after a sufficient number of sub-iterations



How to evaluate the force imposed by the fluid onto the structures ...

How to evaluate the force imposed by the fluid onto the structures ...

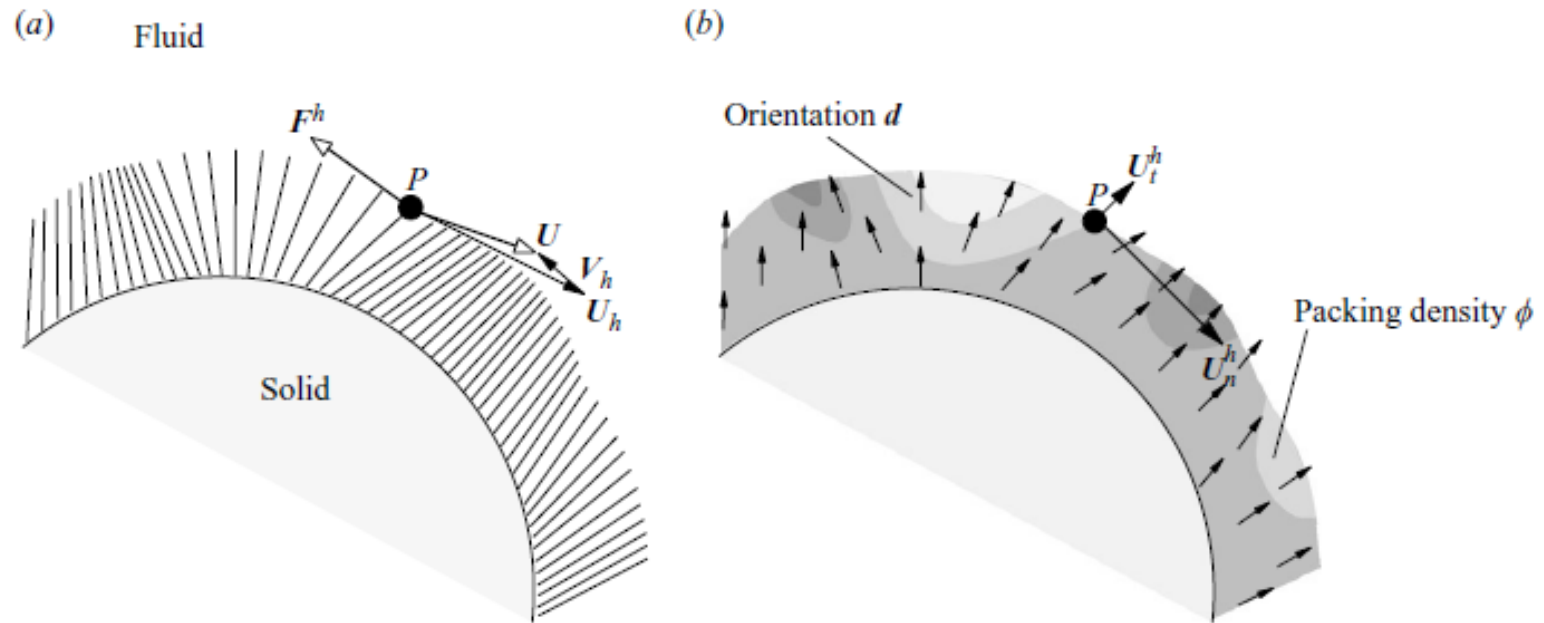
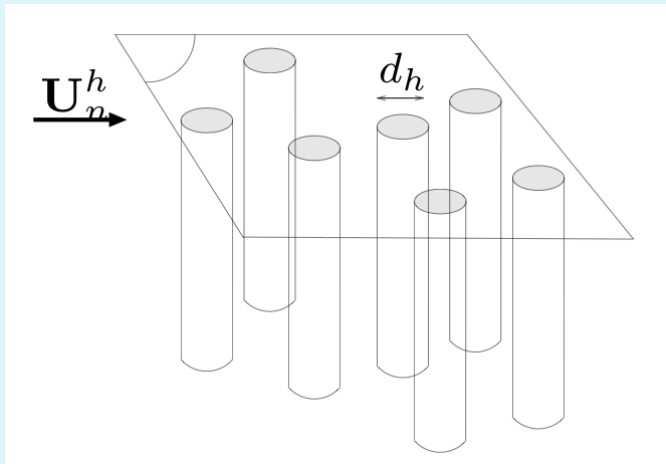


FIGURE 5. Homogenized model of the furry coating and sketch of the volume force F^h imposed on the fluid. (a) Hairy layer covering a cylinder and relative fluid velocity U^h at any point P of the layer, while the single element moves at velocity V^h . (b) Gray scale of the packing density ϕ and vector d orientation of the elements within the hairy layer. At point P , tangential and normal velocity components used to estimate the two components of F^h are displayed.

Homogenized part (fluid+structure) ...

- Each cilium is a circular cylinder
- At each point along the beam, the force is decomposed into a tangential and a normal contribution
- Force on a random cluster of cylinders

Estimate of F_n



$$\frac{||\mathbf{F}_n^h||}{\mu ||\mathbf{U}_n^h||} = c_0(\phi) + c_1(\phi) Re_n^h$$

theoretical (Re=0)

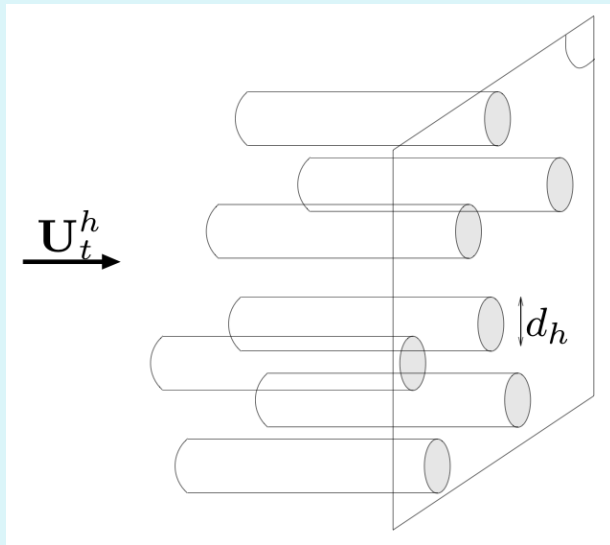
empirical Re<180

Koch & Ladd, JFM 1997

$$||\mathbf{F}_n^h|| = f_2(\phi, Re_n^h)$$

Homogenized part (fluid+structure) ...

Estimate of F_t



→ For $Re = 0$: Stokes approximation:

$$\frac{\mathcal{F}_t^h}{\mu \|\mathbf{U}_t^h\|} = \frac{8\pi(1 - \phi)^2}{\phi - 1 + \frac{2}{\phi - 1} \ln \phi - 2}$$

$\mathbf{U}_t^h(1 - \phi)$: local velocity through the pores

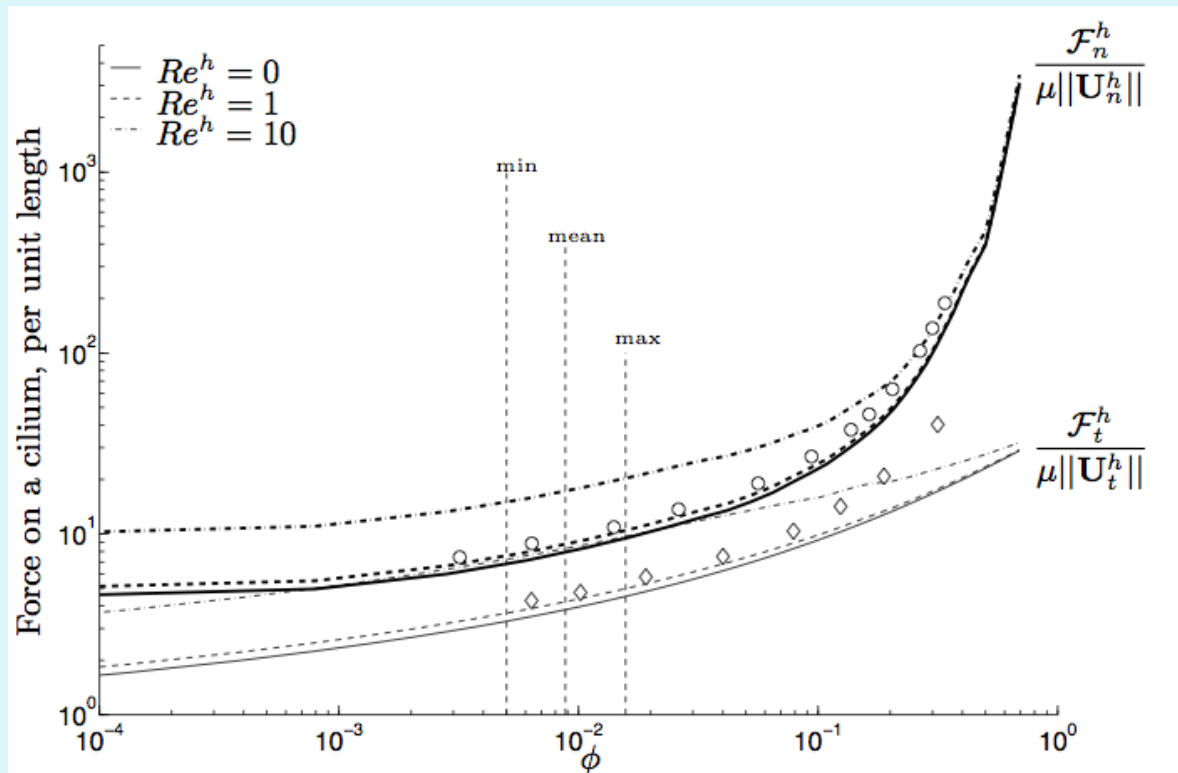
→ For $Re < 180$: same scaling in Reynolds as for F_n

$$\|\mathbf{F}_t^h\| = f_1(\phi, Re_t^h)$$

Inner constants of the layer:

Density (nb/cm²),

Diameter of cilia



Symbols :
theoretical model by
Howells, JFM 1998

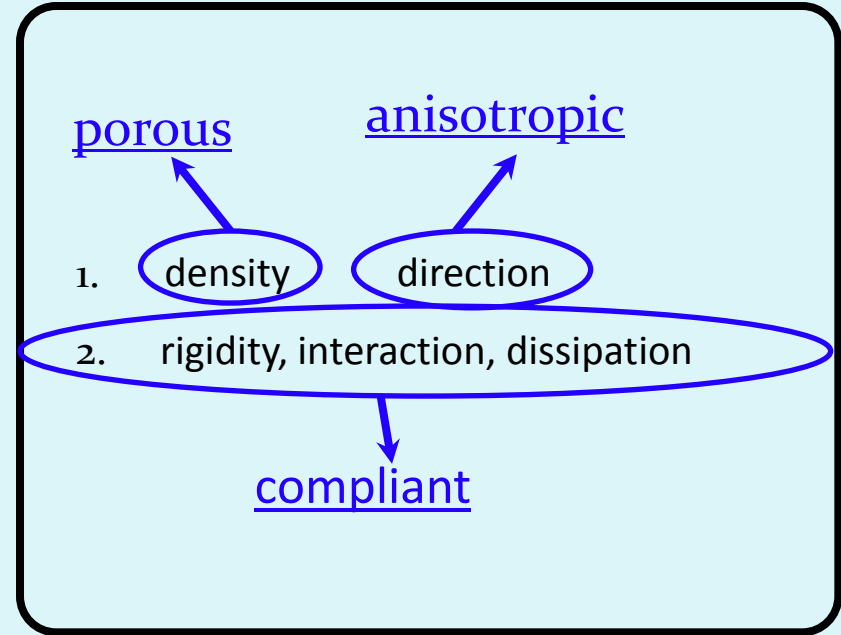
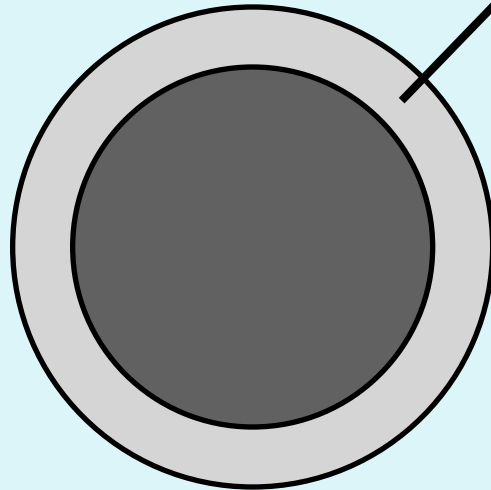
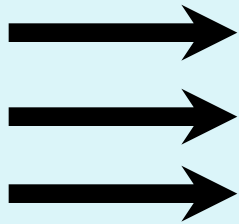
$$\longrightarrow F_{ext}(k) = \int_{V_{control}(k)} \|\mathbf{F}^h\| dV$$

Global overview

Fluid

Reynolds number

Volume forces



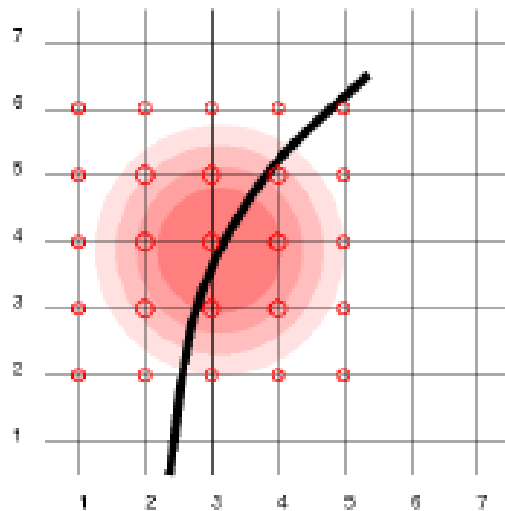
Hairy coating

Volume force of each bristle onto the fluid

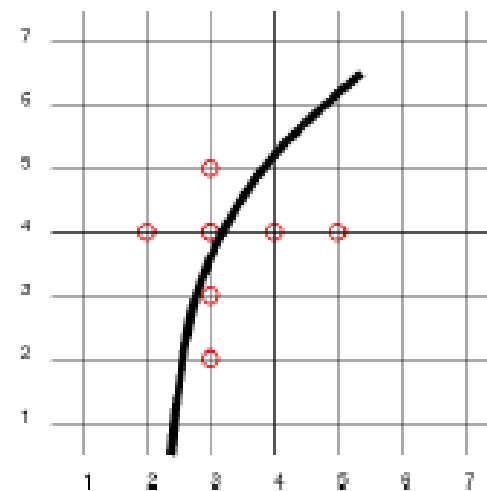
- The DNS is performed on a cartesian mesh. The cilia do not exactly coincide with the nodes \rightarrow we need to interpolate.

Two interpolation strategies are tested and compared on two test cases:

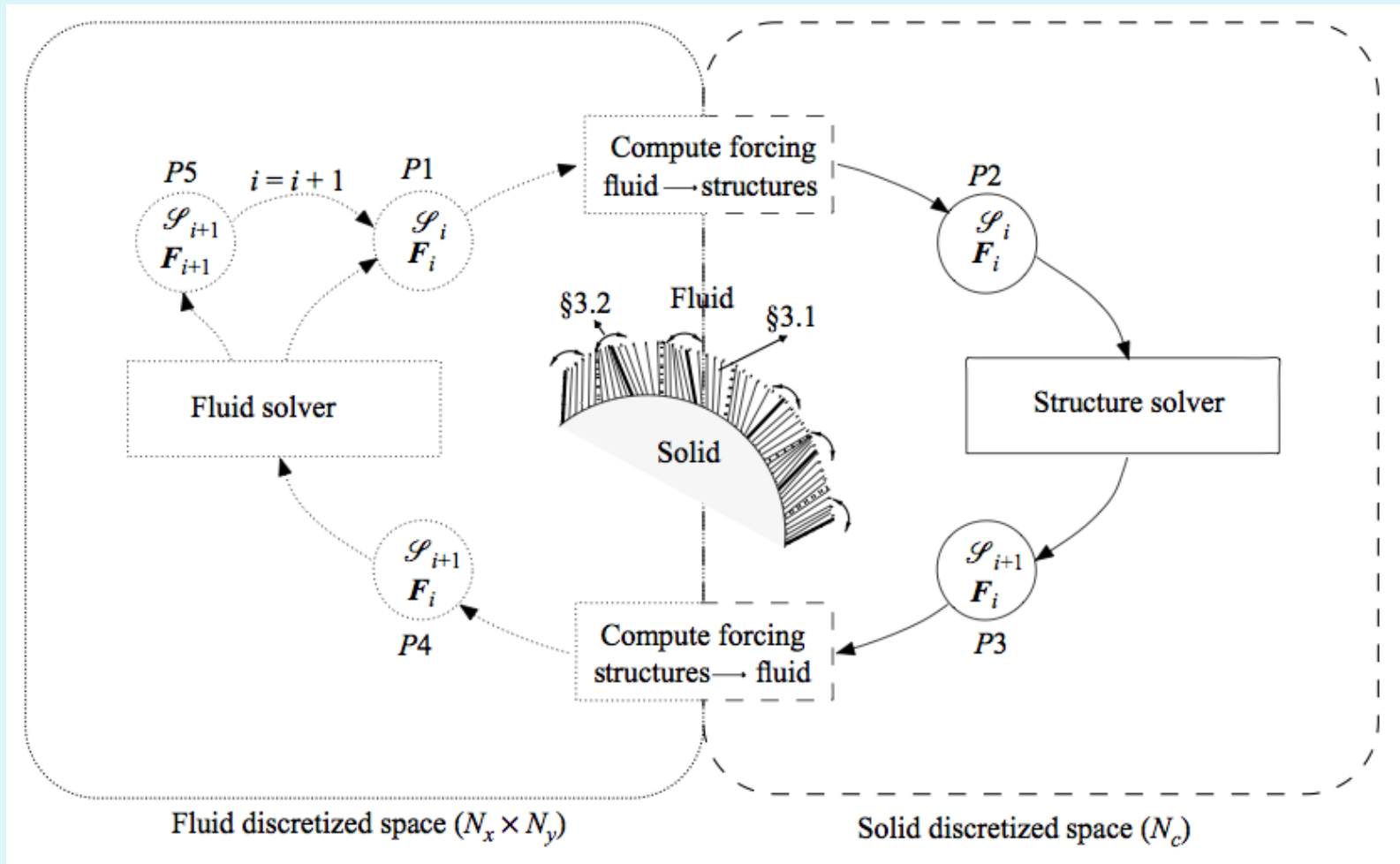
- *Distributed interpolation:*



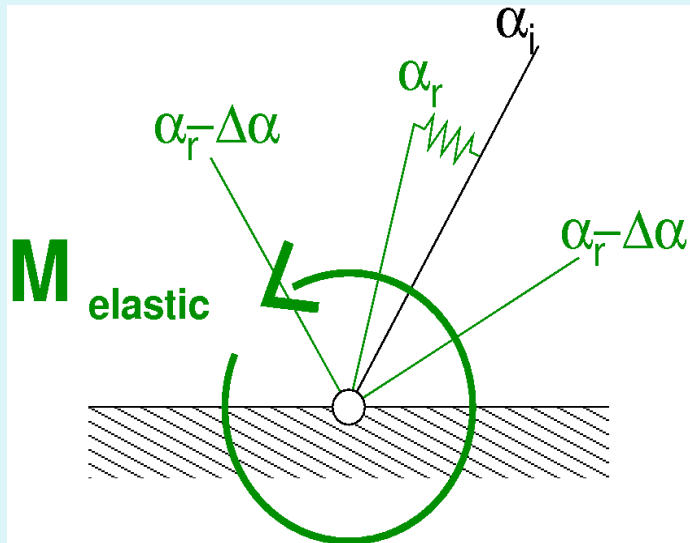
- *Linear interpolation:*



Weak coupling FSI algorithm



In the structural model, the **rigidity/elastic term**, which models the structural flexibility of the hairy layer, is the most significant. It defines a natural time scale of the layer, through which a coupling with the fluid is allowed

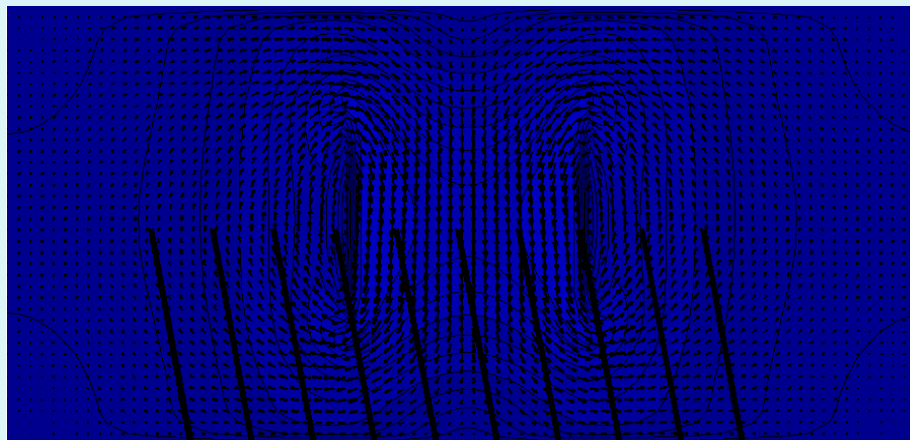
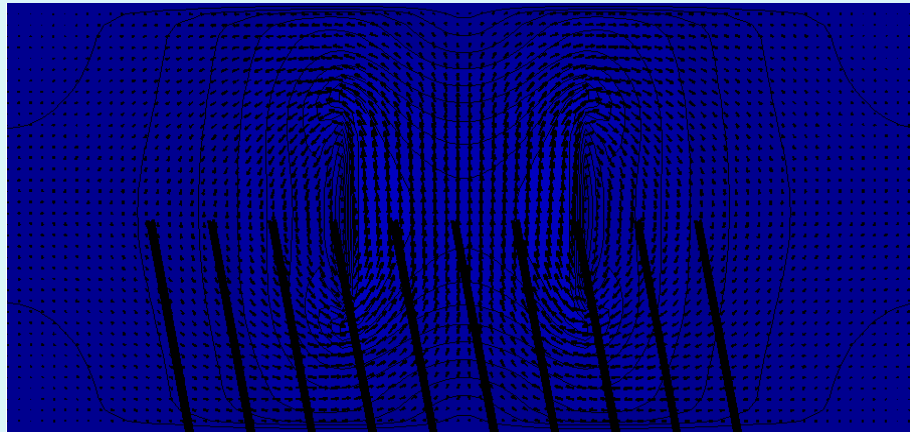


$$T_{\text{structure}} \approx \pi l \sqrt{m/K_r}$$

$$T_{\text{fluid}} \approx \text{St}^{-1} D/U_{\infty}$$

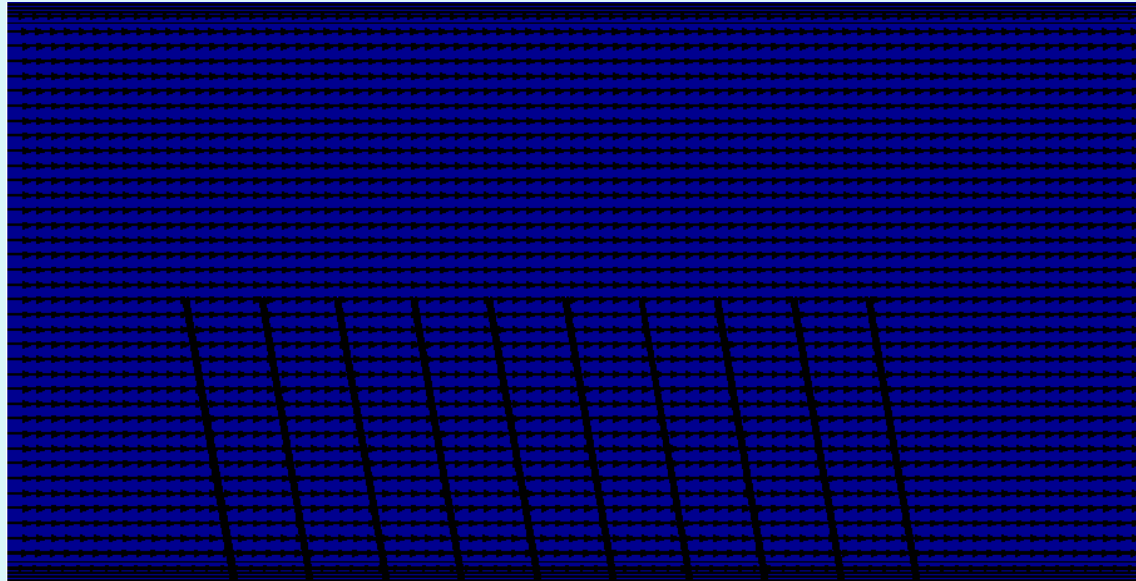
Testing the model in the fluid (a vortex pair in a periodic box)

Amplitude of the velocity



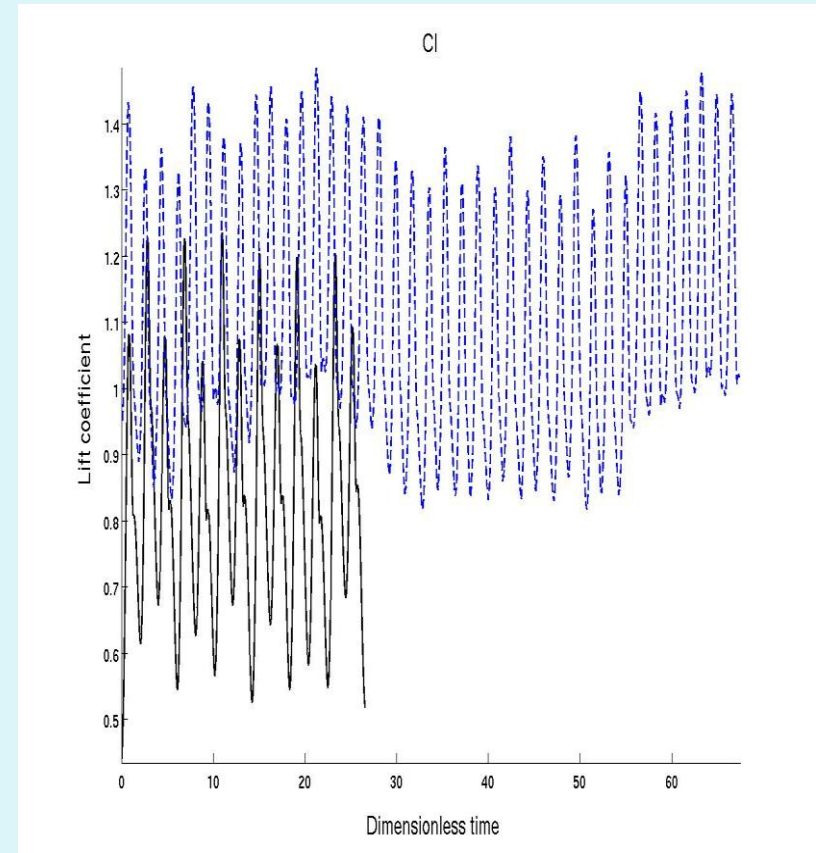
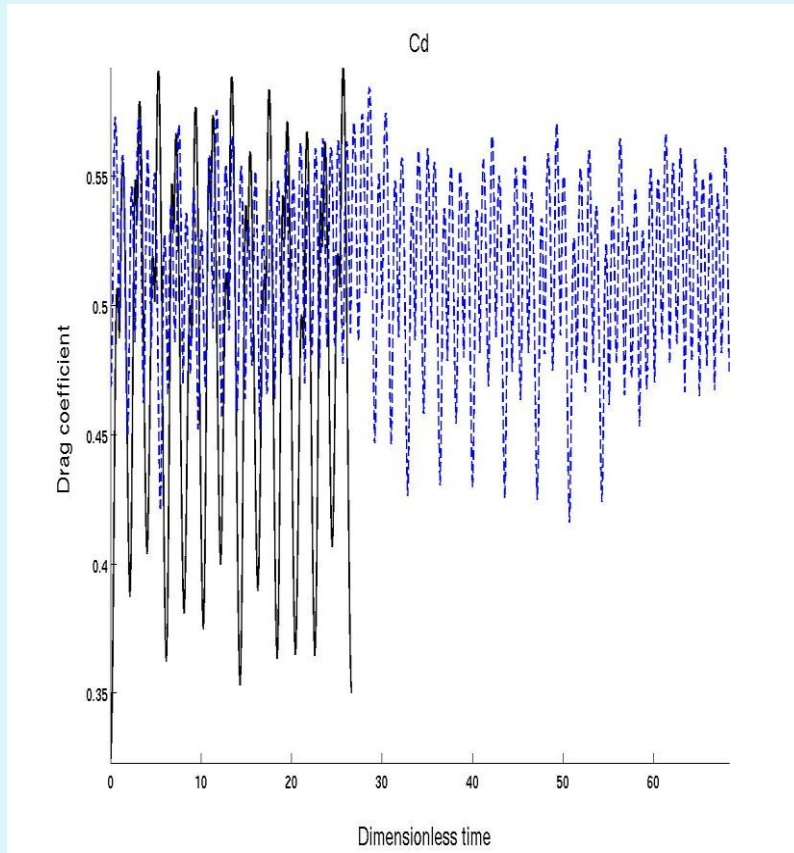
Testing the model in the fluid (flow in a channel with one hairy wall)

Amplitude of the velocity



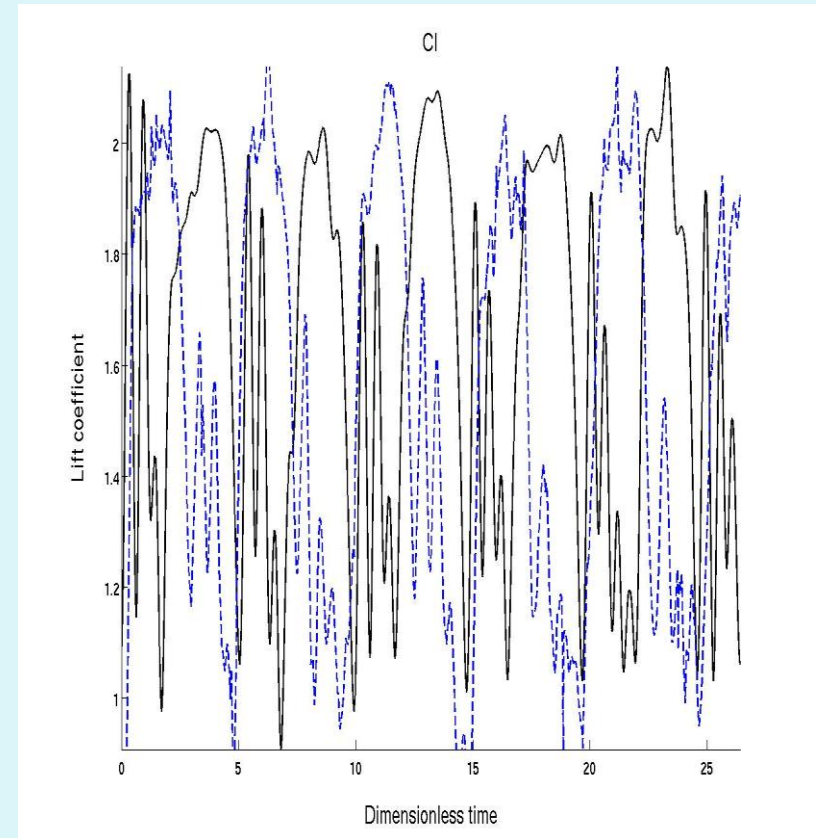
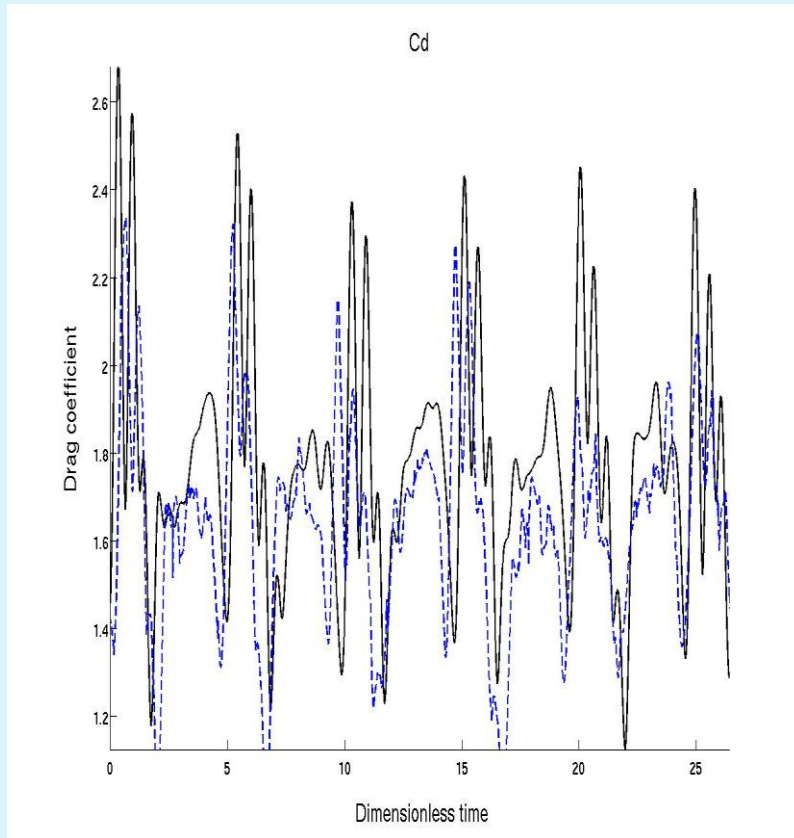
*Back to the airfoil **with** control elements*

COMPARATIVE ANALYSIS OF DRAG & LIFT – 22 DEGREES



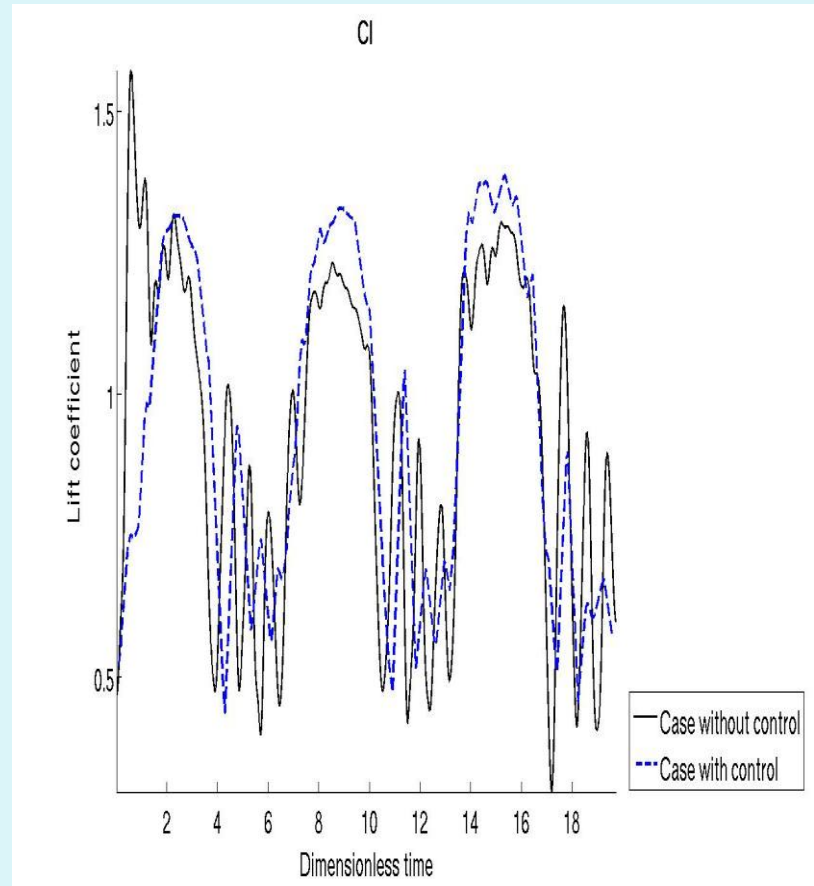
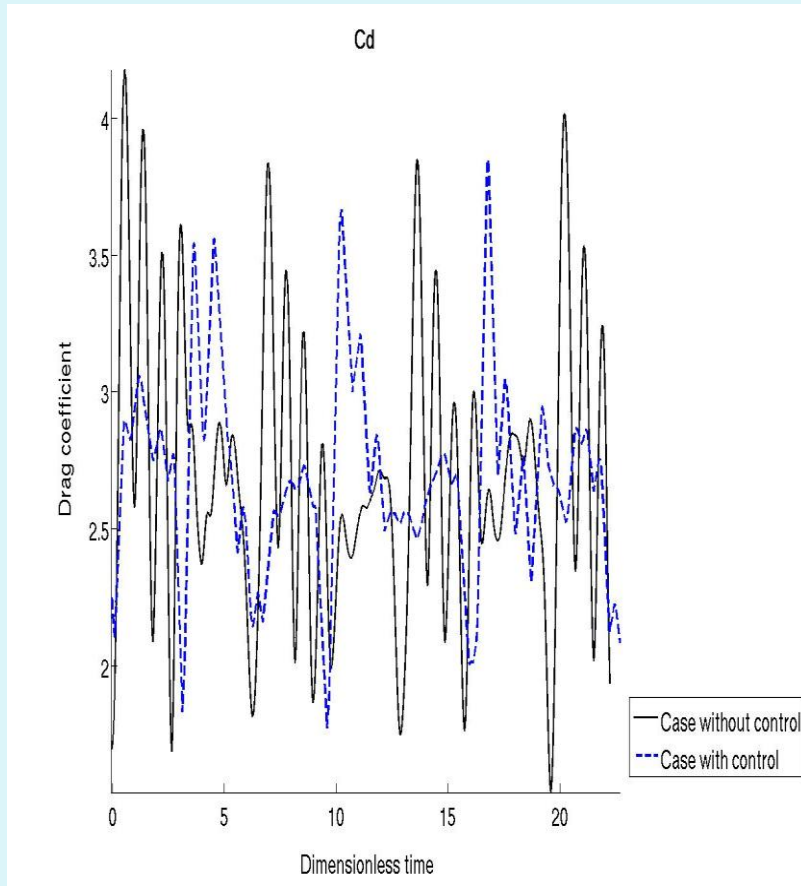
Exact synchronization of fluid shedding time scale and bristles' time scale

COMPARATIVE ANALYSIS OF DRAG & LIFT – 45 DEGREES



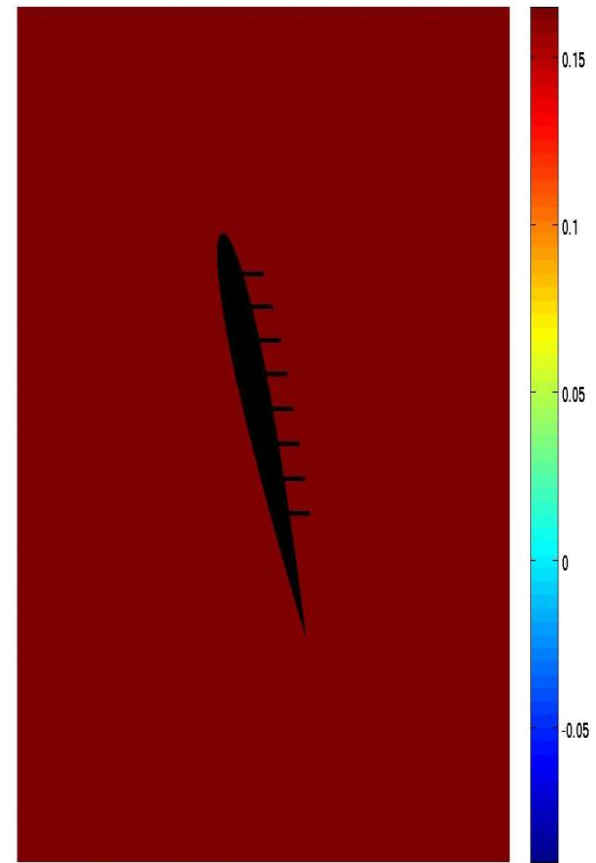
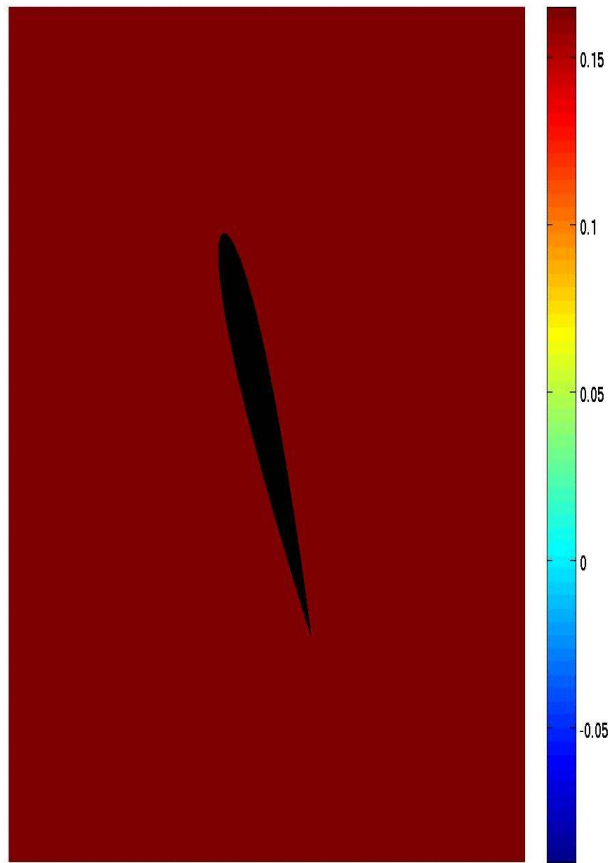
Exact synchronization

COMPARATIVE ANALYSIS OF DRAG & LIFT – 70 DEGREES



Approximate synchronization

COMPARATIVE ANALYSIS OF VELOCITY NEAR THE AIRFOIL— WITHOUT & WITH CONTROL



COMPARATIVE ANALYSIS OF VORTEX SHEDDING – WITHOUT & WITH CONTROL




Modification of the wake ... uhm

SUMMARY

Sets of control parameters are found which give enhanced aerodynamic performances with the following features:

- **For 22°:**
Increase in mean lift – 32.24%, Decrease in lift oscillations – 16.74%,
Decrease in drag oscillations – 37.44%, Mean drag increases – by 6.6%.
- **For 45°:**
Decrease in mean lift – 9.23%, Decrease in drag fluctuations – 8.79%,
Mean drag decreases – by 1.47%.
- **For 70°:**
Increase in mean lift – 16.96%, Decrease in lift fluctuations – 25.75%,
Decrease in drag fluctuations – 21.28%, Decrease in mean drag –
approx. 1.48%.

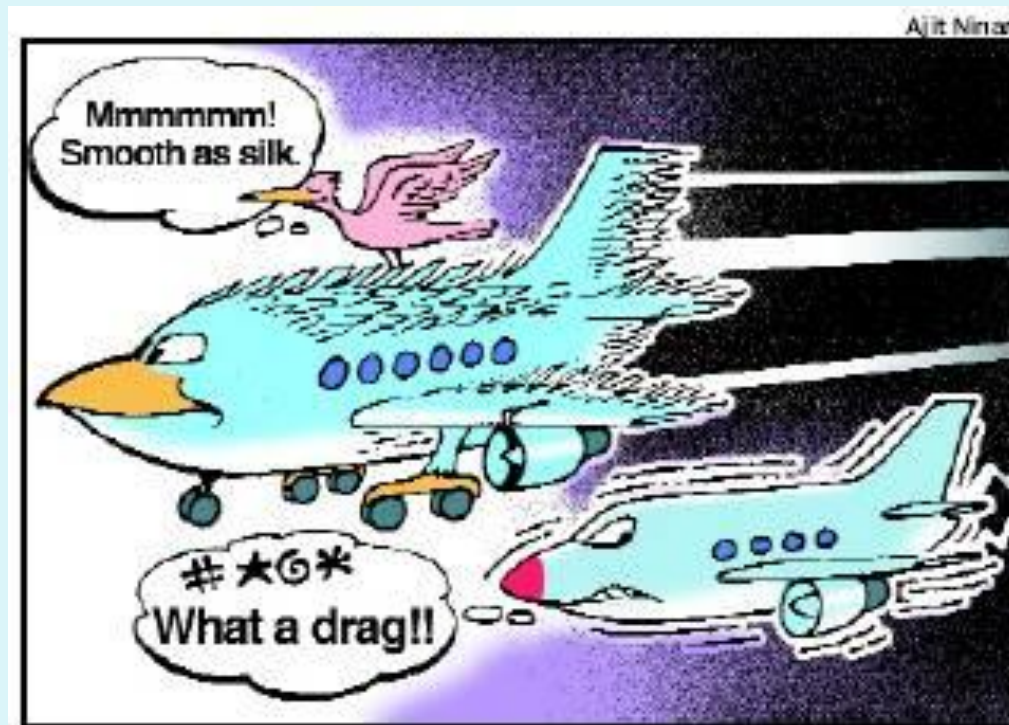


Before you ask me ...

Before you ask me ...

- How to convert model parameters into a choice for an effective material for the actuators? The “optimal” material may not be viable or may not even exist.
- How do you optimize?
- Must increase Re to render it more pertinent!
- What works at one Re /angle of attack may hamper performances for different sets of parameters ...
- Etc.

Will we ever get here?



The Times of India, April 16 2009